

POSSIBILITIES FOR THE USE OF METAL-HYBRID-STRUCTURES FOR VEHICLE CRASH LOAD CASES

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Wissen für Morgen

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- Introduction of the institute of vehicle concepts
- Motivation and basic principle
- Simulation and Optimisation
- Possible applications

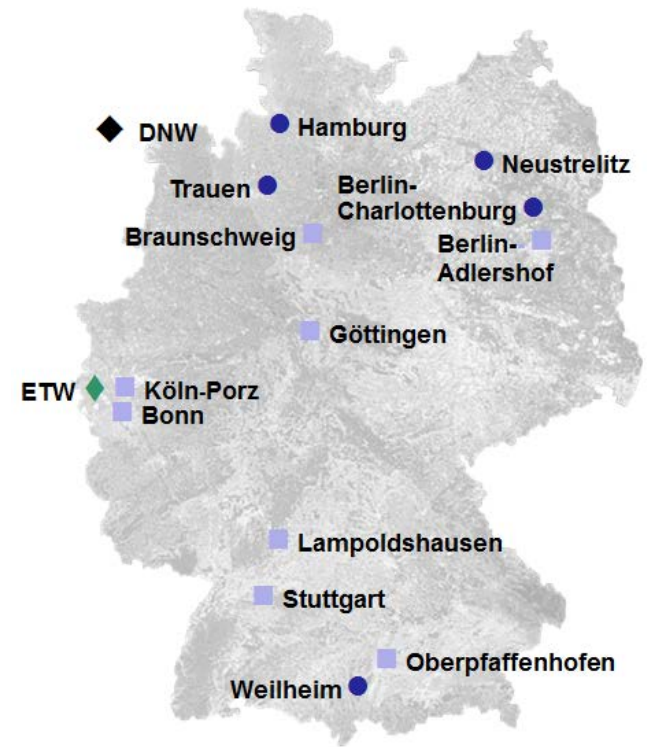


DLR – Overview

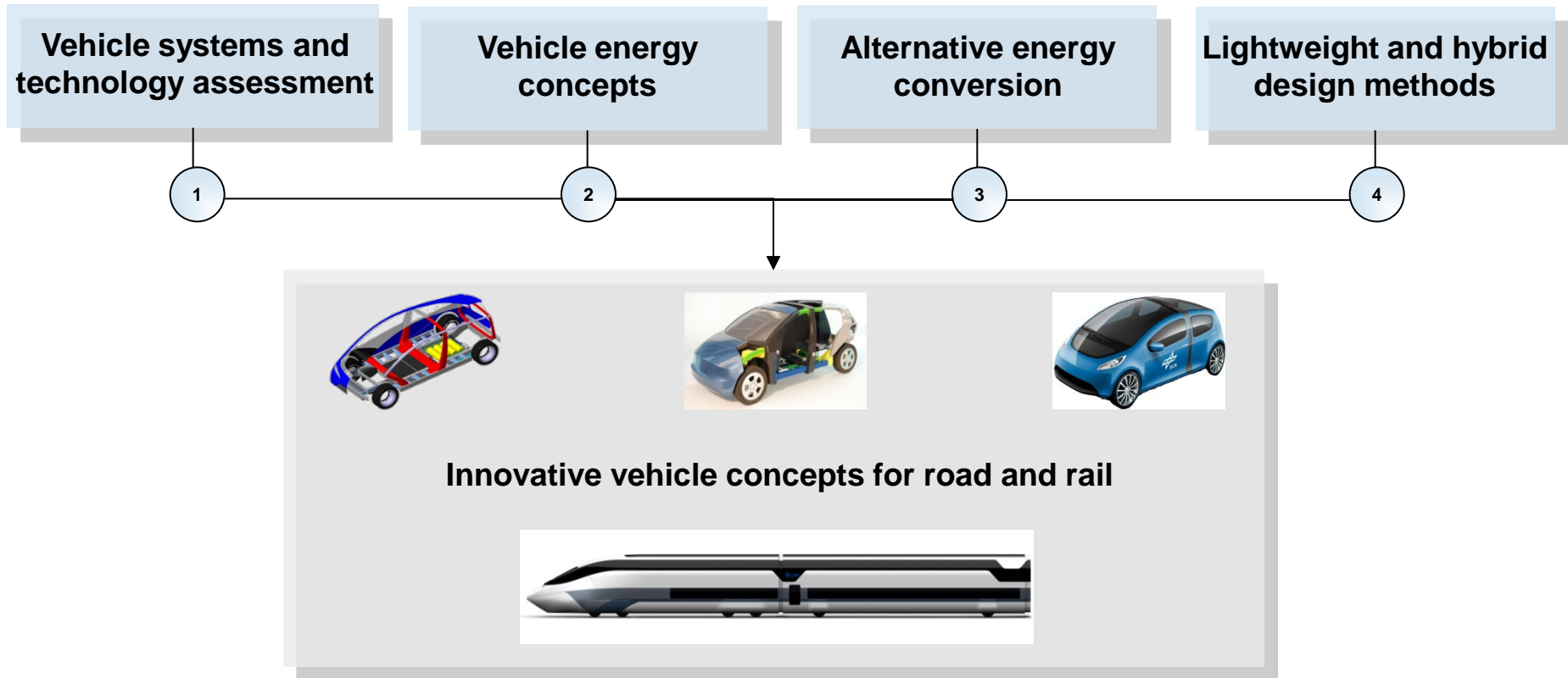
DLR's mission:

- exploration of the Earth and the solar system
- research aimed at protecting the environment
- development of environmentally-friendly technologies to promote mobility, communication and security.

7.700 employee are working at 32 research institutes and facilities in ■ 9 locations and ● 7 branch offices.



The departments of the Institute - FK



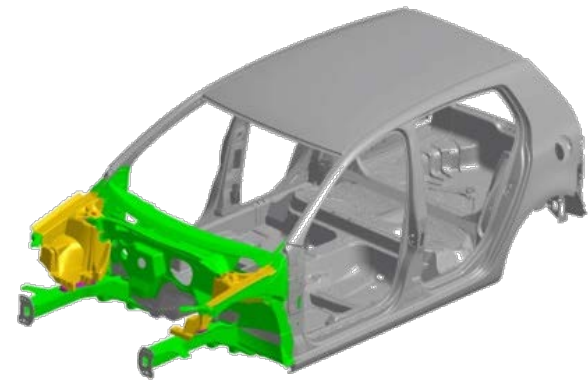
FK designs and demonstrates innovations for the vehicle concepts and technologies of future compliant transport systems



DLR Institute of Vehicle Concepts

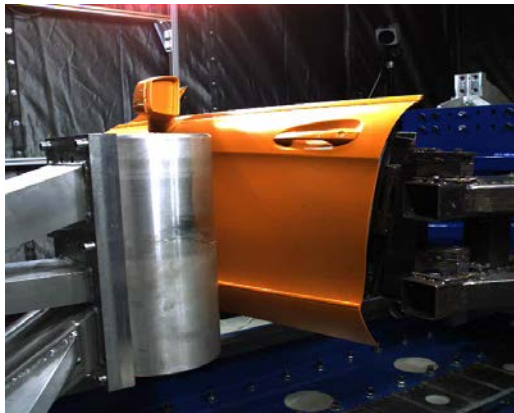
Lightweight & Hybrid Design Methods

- Development of resource-efficient, innovative vehicle concepts
- Safe, light and cost-effective
- Adaptation to alternative drive train concepts



Lightweight & Hybrid Design Methods

Passive safety / crash simulation and testing

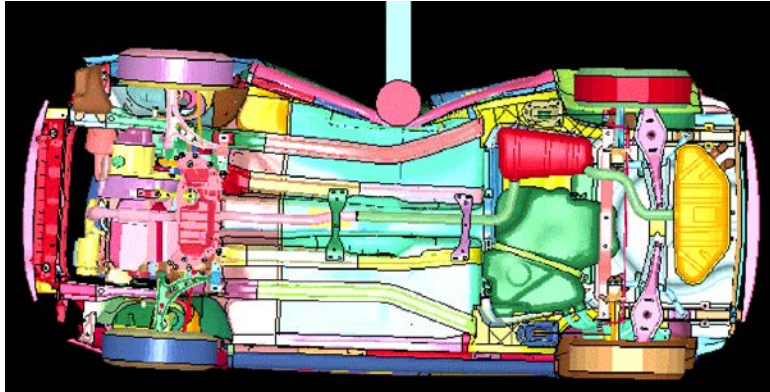


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Motivation



floor structure developed by DLR during SLC-project

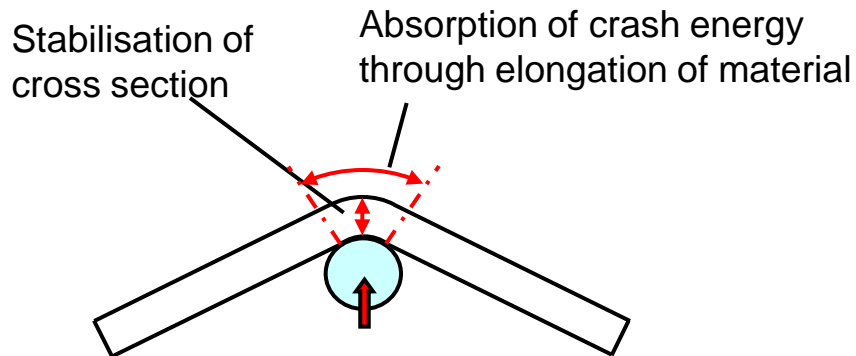
Hollow beam made of DC 04



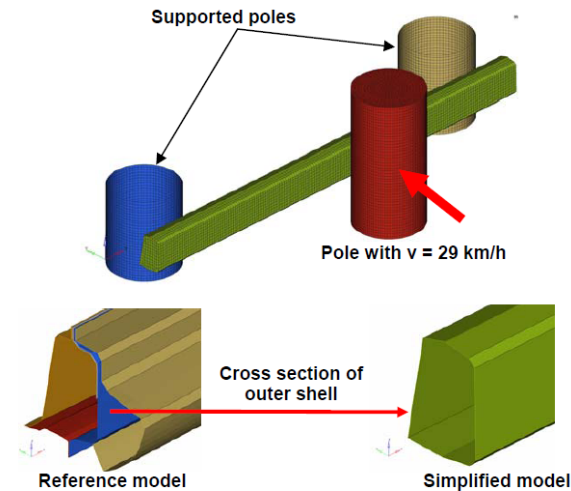
- Collapse of the rocker's and side piece's cross-section during pole-crash -> energy must be absorbed by various other components
- A stabilisation of the cross-section during bending should lead to a much higher weight specific energy-absorption of the rocker -> higher freedom of design and choice of materials for the surrounding structures, like the floor panels -> possibility of an overall weight reduction
- The storage of critical components like Li-Ion batteries in the underbody requires a low intrusion
- Demand for a simple, lightweight concept made of relatively cheap materials, adaptable to different kinds of vehicle concepts



Basic principle



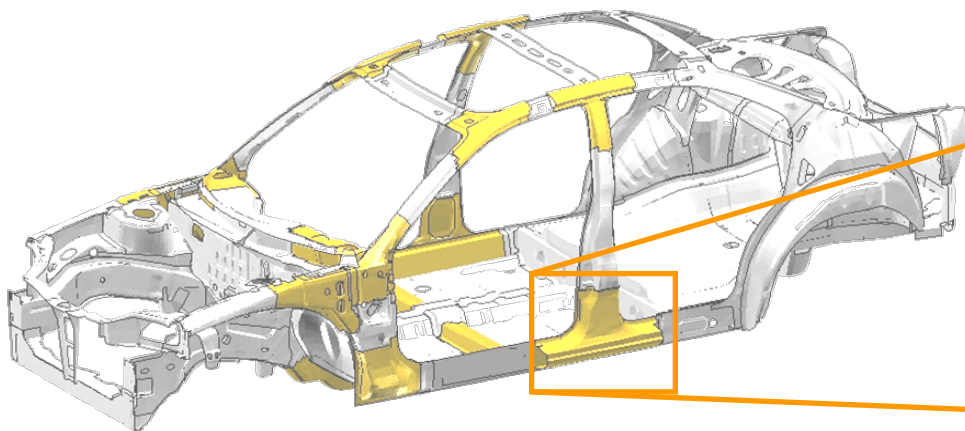
- Stabilisation of the beam by a core structure
- The core must stay intact, throughout the entire bending process, in order to increase weight specific energy absorption
- Simplified LS-Dyna-calculations showed an increase in weight specific energy absorption by a factor of about 2,5



Variant	Drawing	Total mass [kg]	Material	Energy absorption [kJ]	kJ/kg
Reference		22,39	Various types of steel	4,5	0,2
Al honeycomb		15,15	Core: 1 mm Al; shell: 1 mm TRIPLEX	5,8	0,38
Foam		28,1	Core: foam 400 kg/m ³ ; shell: 1 mm TRIPLEX	14	0,5



Cavity filling with BETAFOAM™



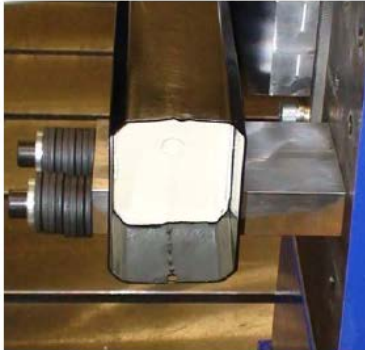
- BETAFOAM™ is a family of foam-based products
 - Two-component polyurethane foam applied as bulk
 - Fast cycle time, room temperature curing
 - Components form a rigid, closed cell foam
- Foam products range in density from 30 g/l to 400 g/l
- Higher density foams provide multi-functional benefits



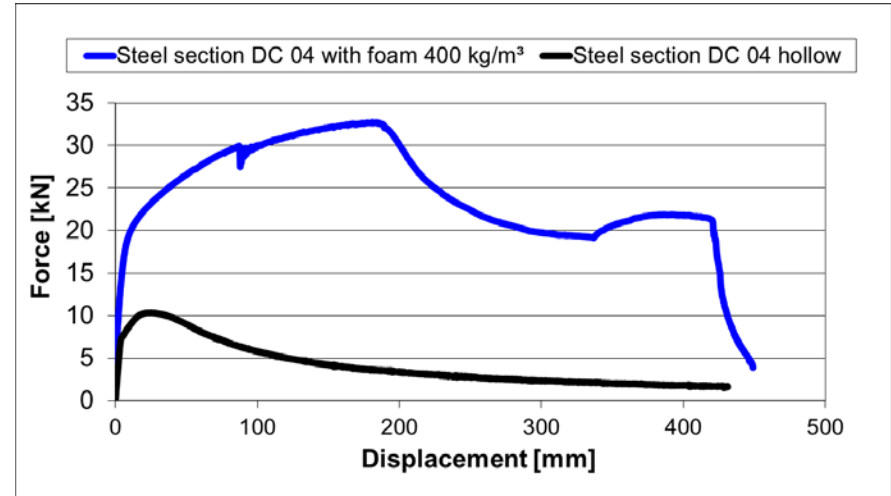
™Trademark of The Dow Chemical Company



Testing performed in cooperation with DOW



DC 04 - beam filled with foam by the DOW Chemical Company density 400 kg/m^3 -> weight increase by a factor of 1,72 compared to hollow beam



hollow beam 12,35 kg

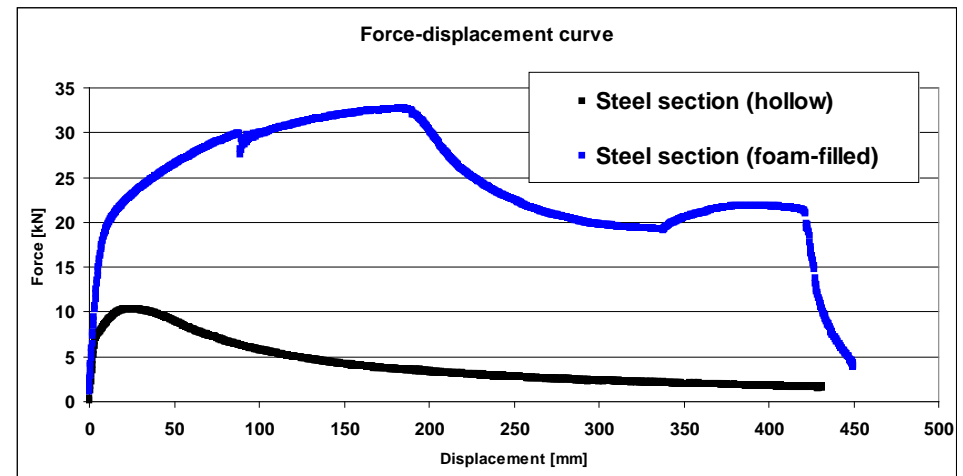
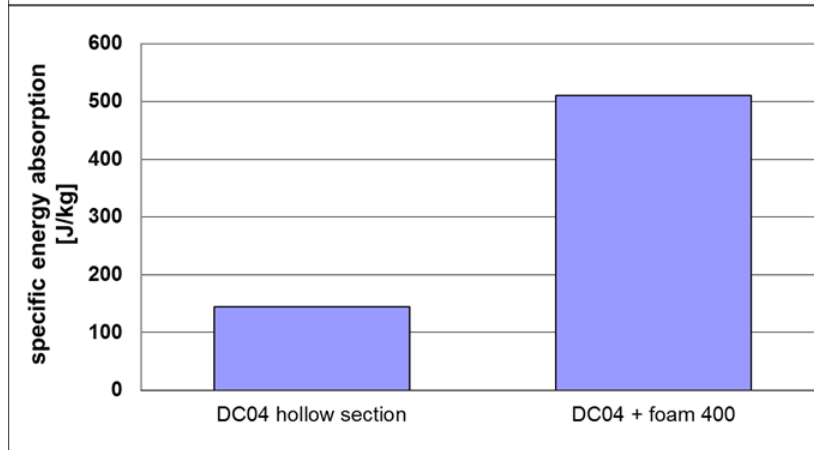
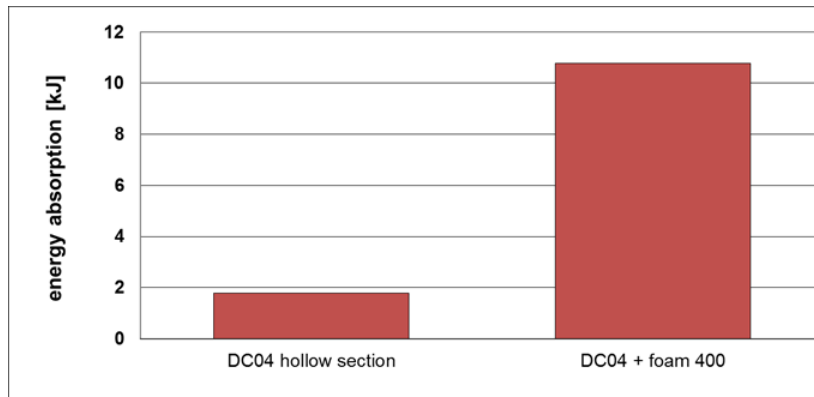


foam filled beam 21,15 kg

- 3- point bending test, distance between supports: 1300 mm
- Cross section of the beam: 128 x 90 mm
- Diameter of the pole: 300 mm
- Energy absorption = force at the pole x displacement of the pole



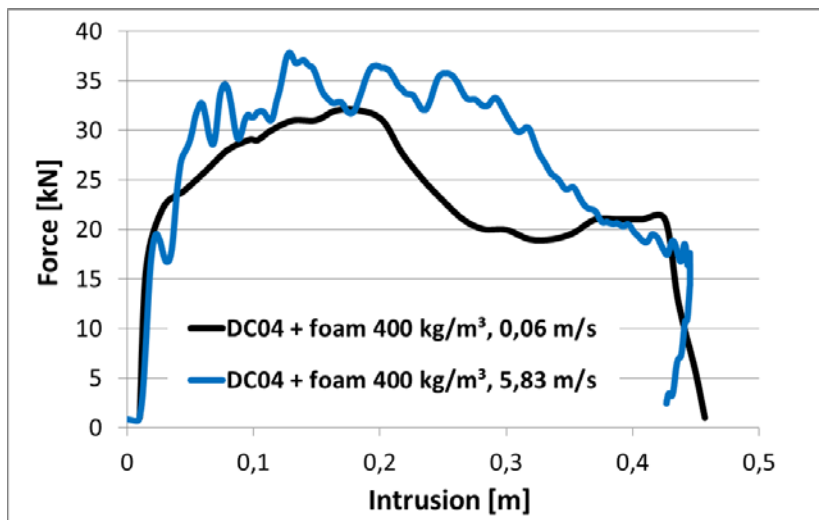
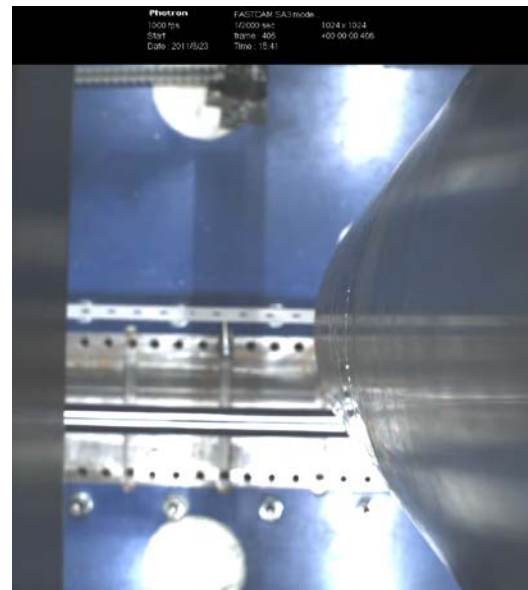
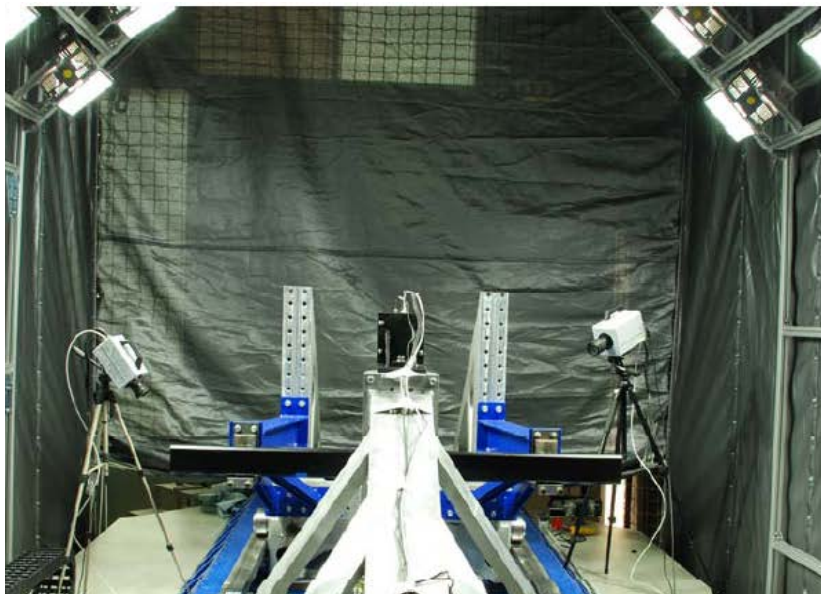
Testing performed in cooperation with DOW (2)



- Increase of the energy absorption by a factor of 6
- Increase of the weight specific energy absorption by a factor of 3



Dynamic Testing



- Dynamic testing results in a slightly higher force level



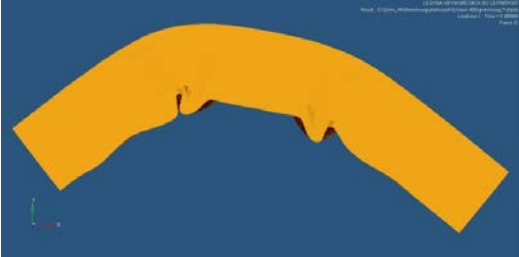



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Simulation and test

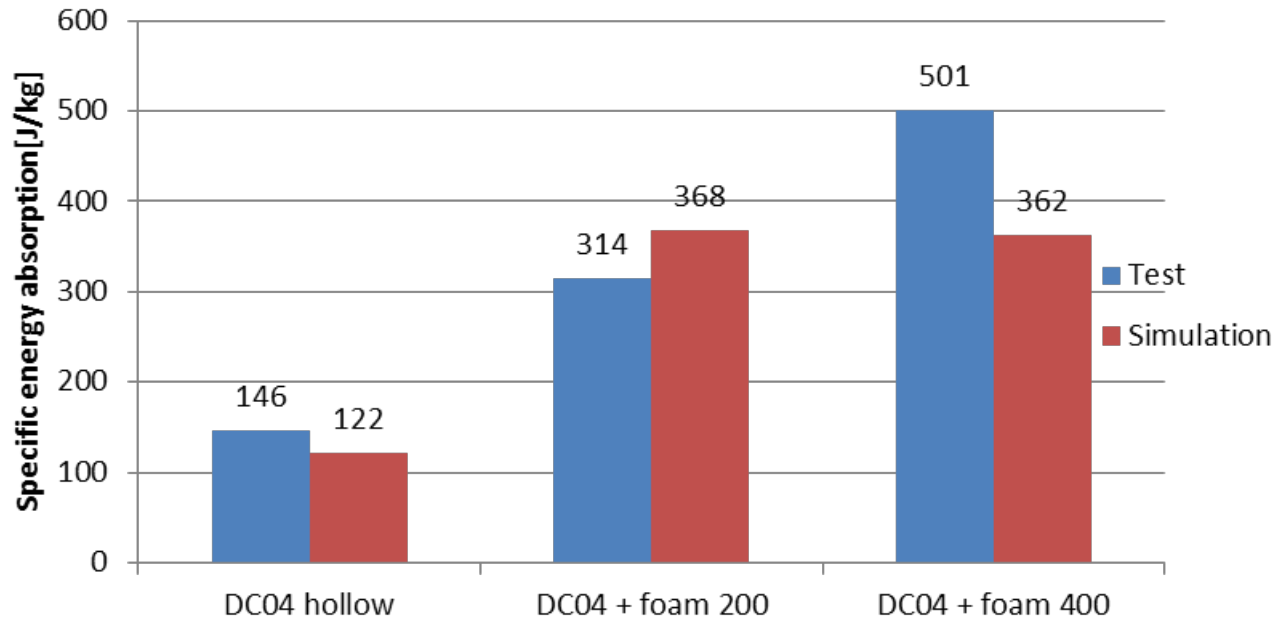
Foam density [kg/m ³]	Simulation	Test
200		
400		

- A reduction of the core density leads to a deeper fold in the outer wall
- The tendency for folding of the outer wall can be predicted
- There are visible differences in the mode of deformation



Simulation and test

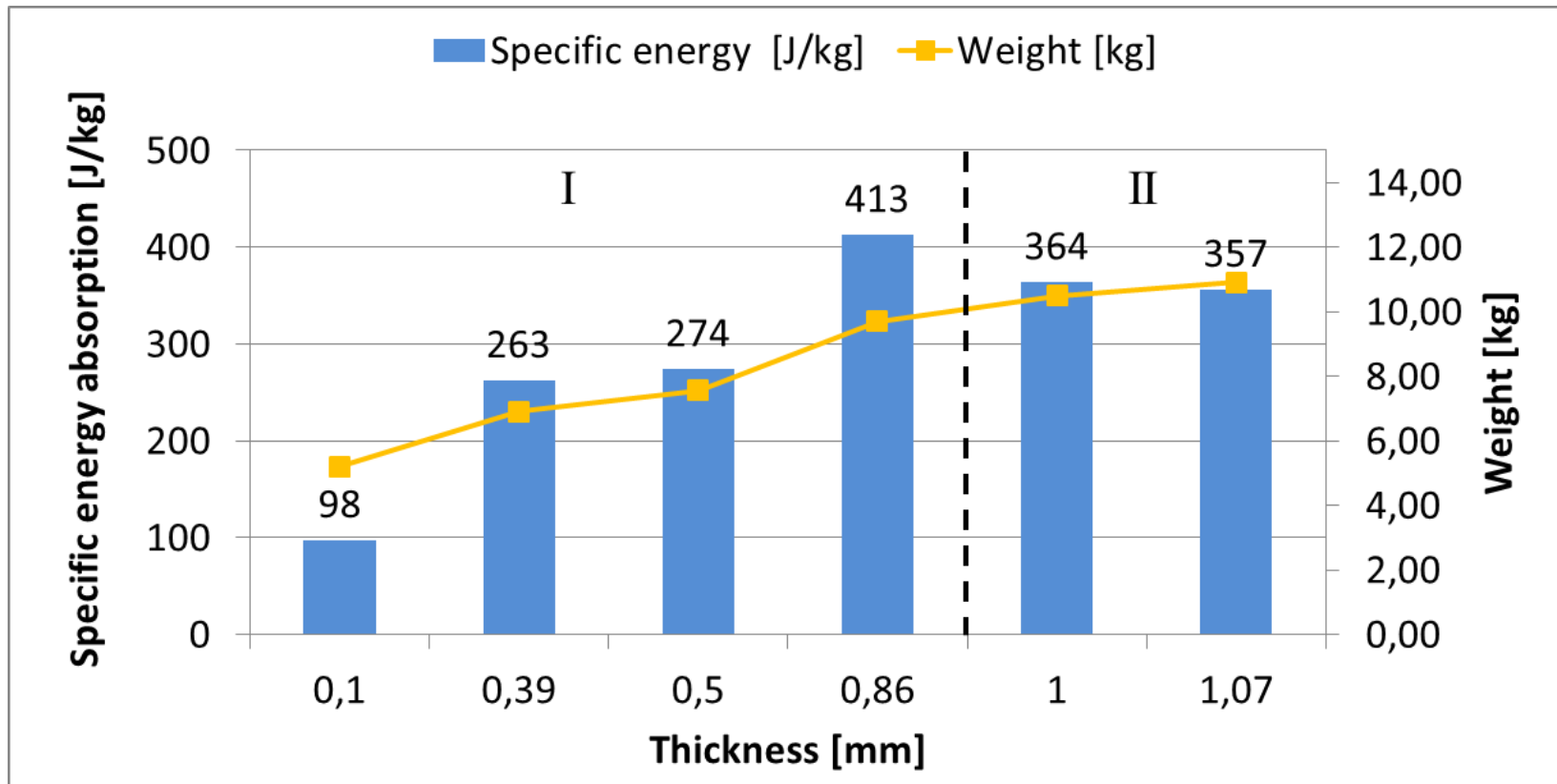
Results of simulation and test



- Influence of the foam density is lower in simulation



Influence of wall thickness



- Constant foam density of 200 kg/m^3
- Increasing wall thickness -> reversal point at 0,86 mm



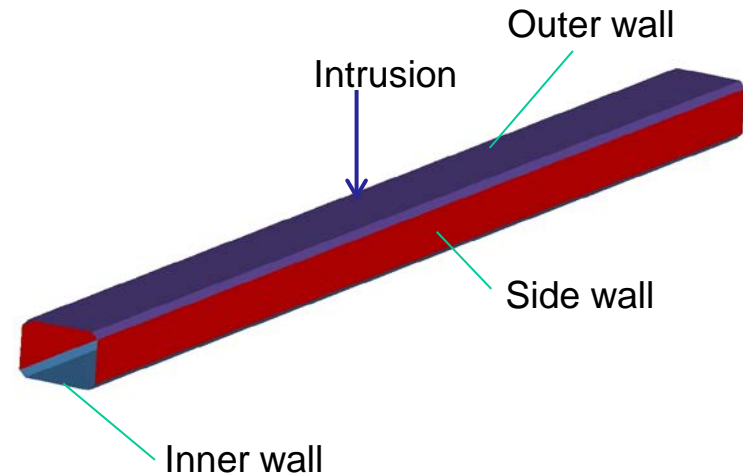
Possibilities for optimisation

Goals:

- Further increase of weight specific energy absorption
- Reduction of the weight of the core

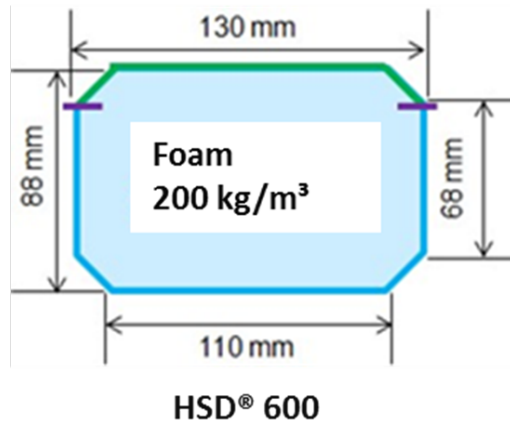
Strategies:

- Variation of the outer wall, side walls and inner wall of the beam regarding:
 - Material
 - Wall thickness

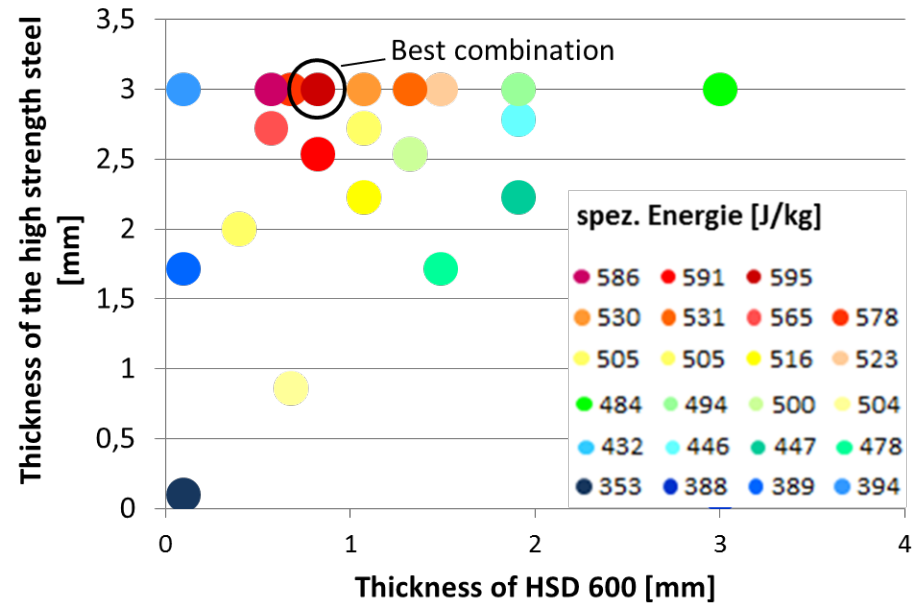


Possibilities for optimisation (2)

High strength steel, $R_{p02} = 1300 \text{ MPa}$



Optimisation of foam filled steel beams



- Constant foam density of 200 kg/m^3
- Variation of the wall thickness of the outer and inner wall from 0.1 mm to 3 mm
- Optimisation of both parameters at the same time, with LS-Opt

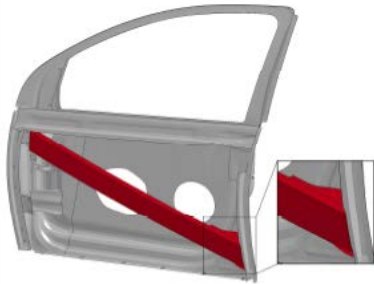


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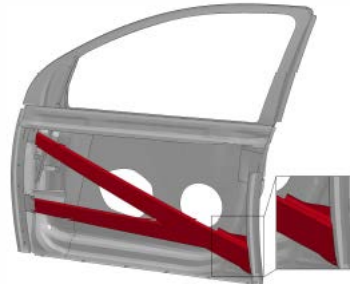


Development of a door side intrusion beam



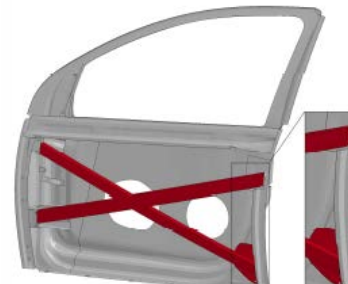
Variant 1

Beam from the hinge to the B-pillar



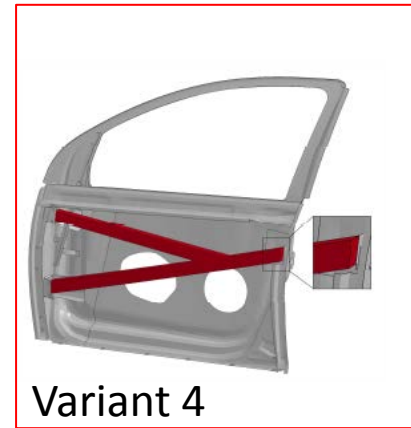
Variant 2

Y-shaped beams from the hinges to the B-pillar



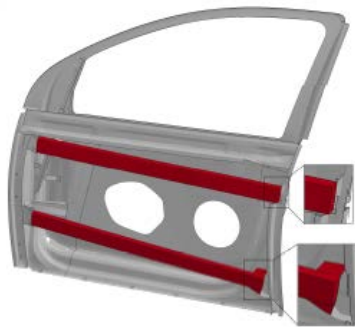
Variant 3

X-shaped beams from the hinges to the B-pillar and lock



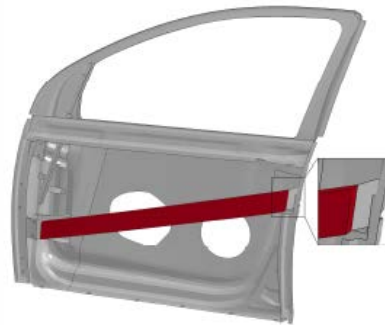
Variant 4

Y-shaped beams from the hinges to the lock



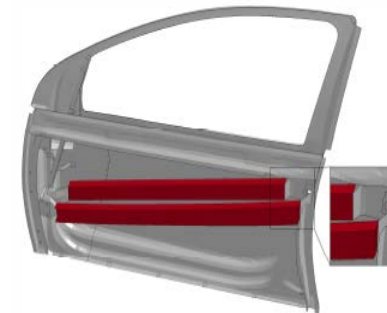
Variant 5

Two beams



Variant 6

Beam from the lower hinge to the lock

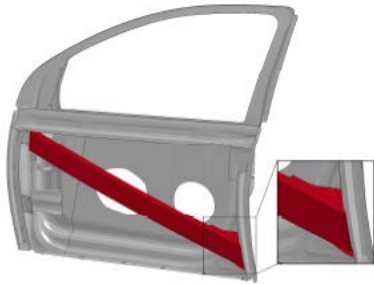


Variant 7

Double beam from the lower hinge to the lock

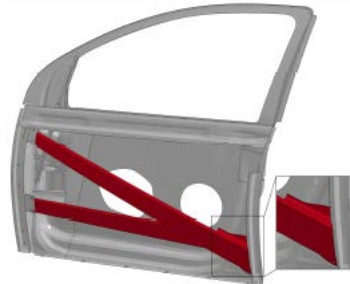


Development of a door side intrusion beam



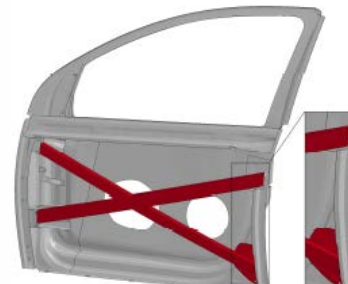
Variant 1

Beam from the hinge to the B-pillar



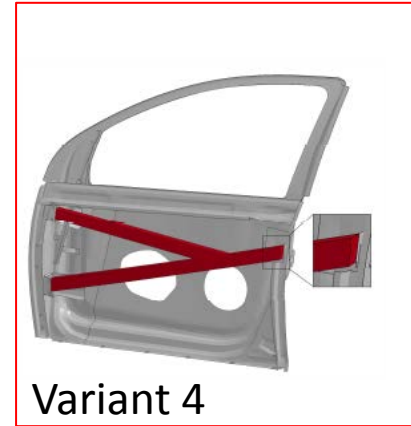
Variant 2

Y-shaped beams from the hinges to the B-pillar



Variant 3

X-shaped beams from the hinges to the B-pillar and lock



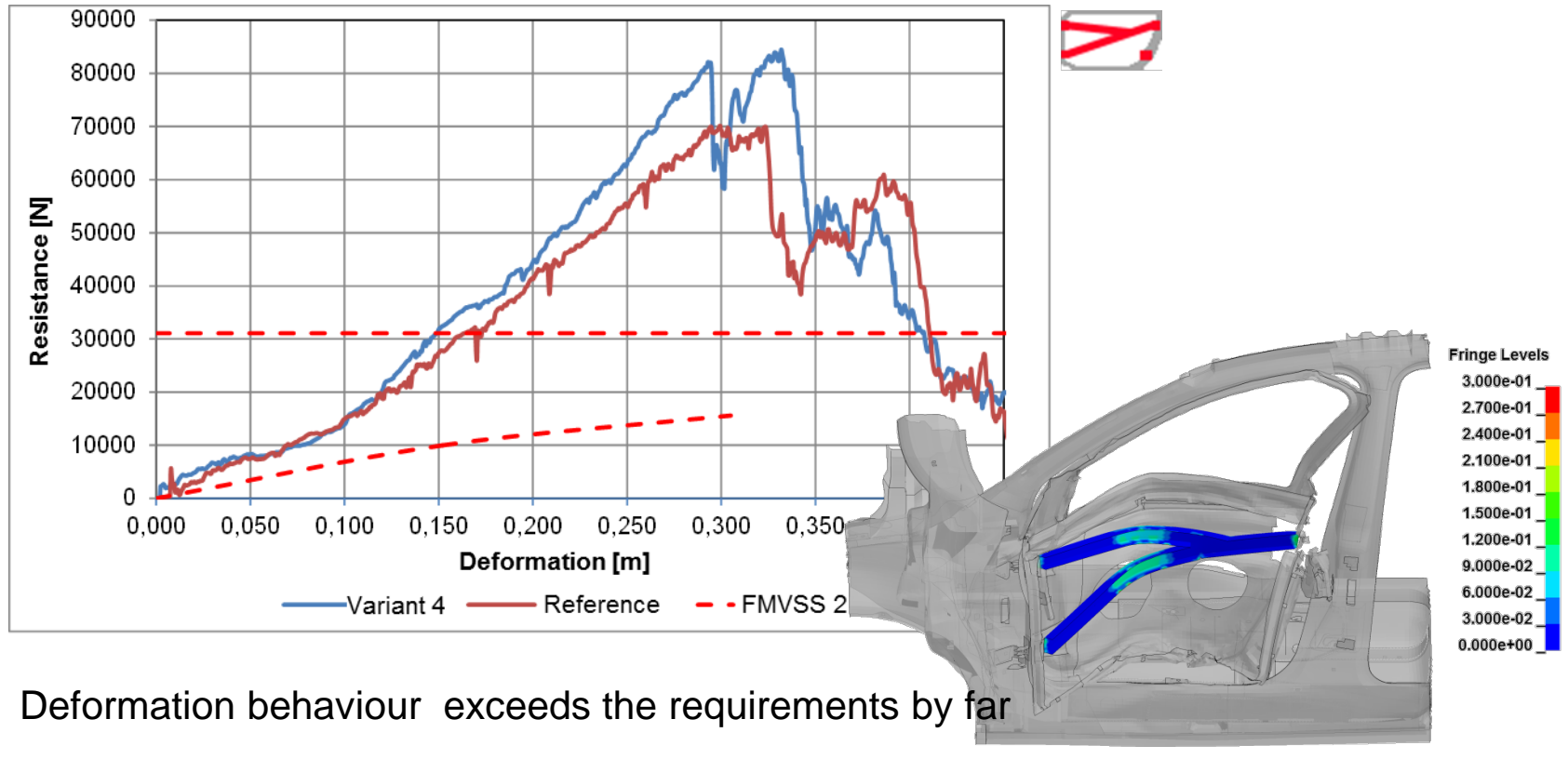
Variant 4

Y-shaped beams from the hinges to the lock

Best ratio of performance to weight



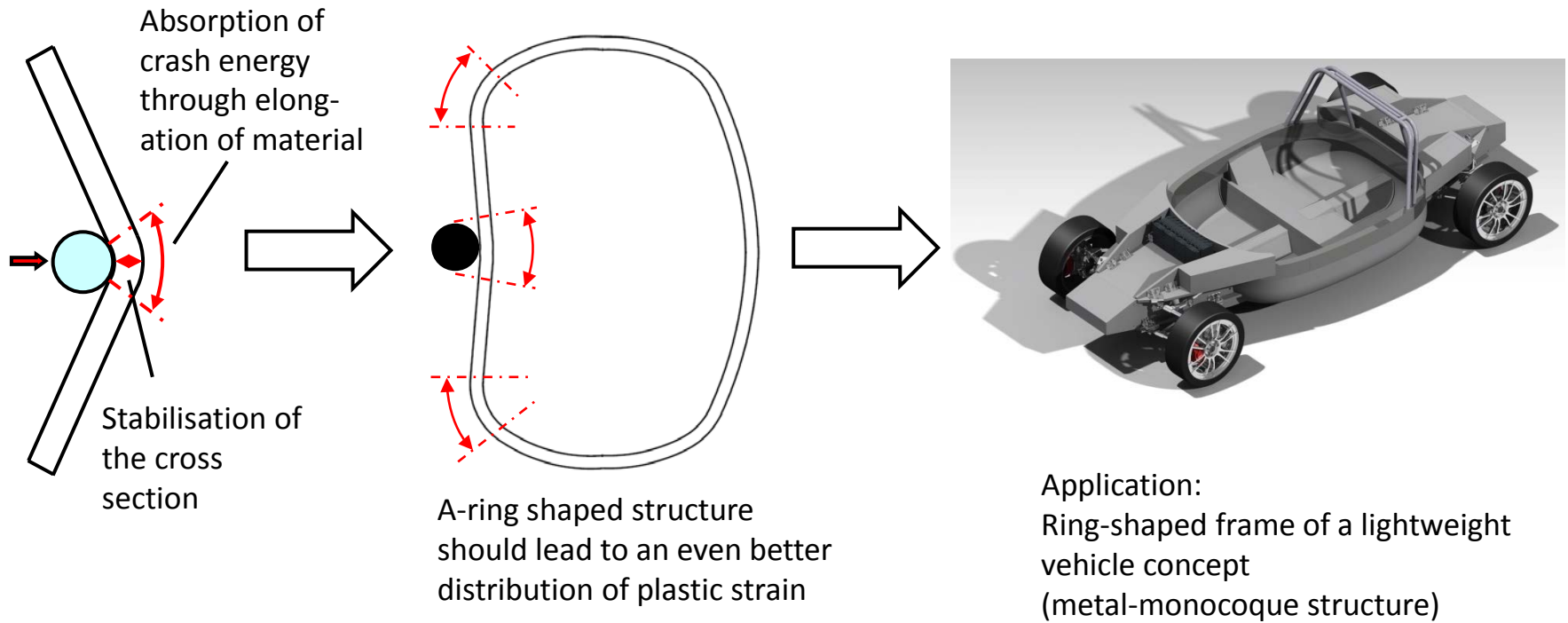
Development of a door side intrusion beam (3)



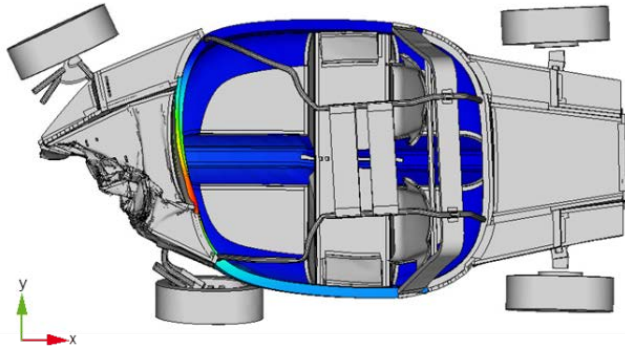
- Deformation behaviour exceeds the requirements by far
- Energy absorption 10 % higher than the reference
- Weight of the side intrusion beam reduced from 2 to 1 kg



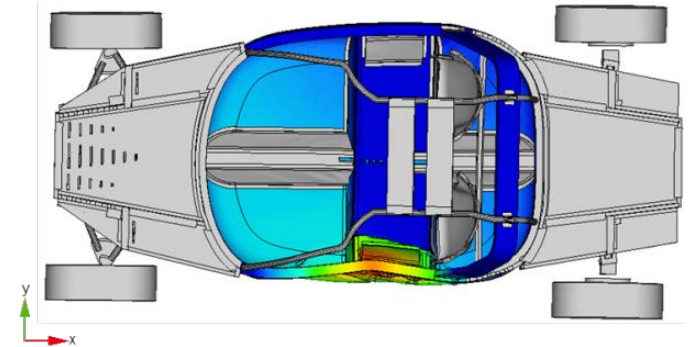
Development of a ring-shaped frame for a lightweight car body



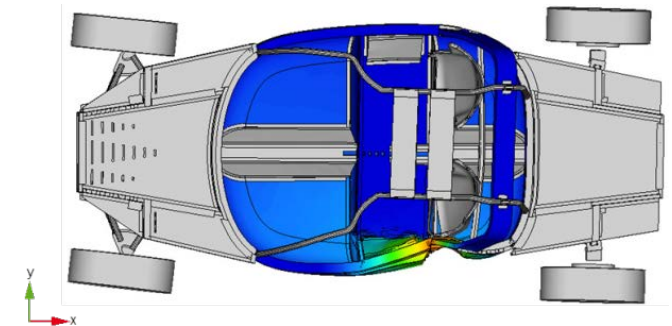
Simulation of crash tests



Euro-NCAP 40% overlap, offset deformable barrier



Euro-NCAP side impact

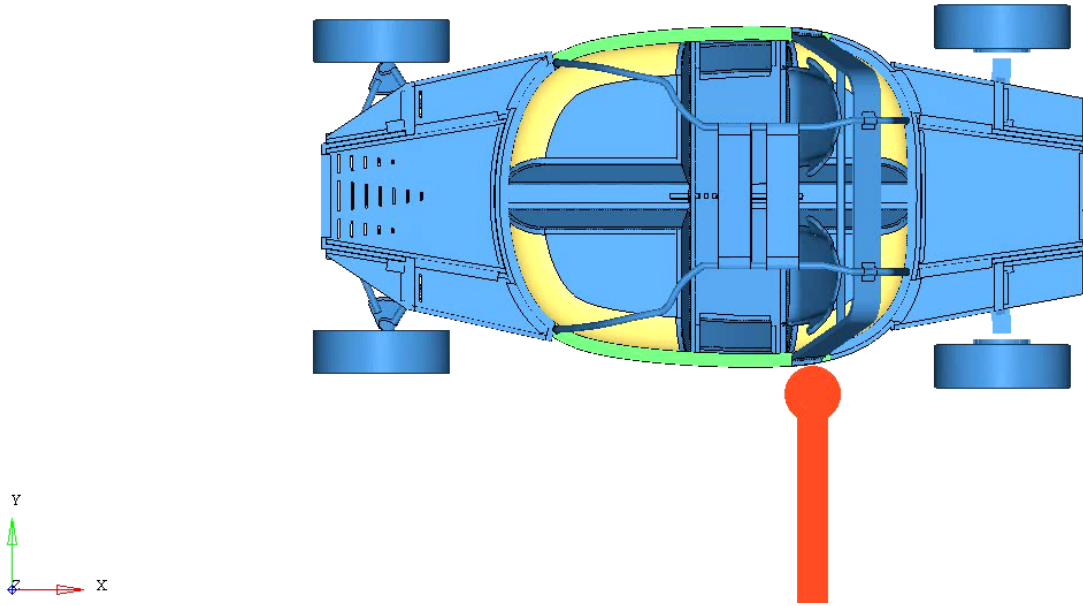


Euro-NCAP pole impact

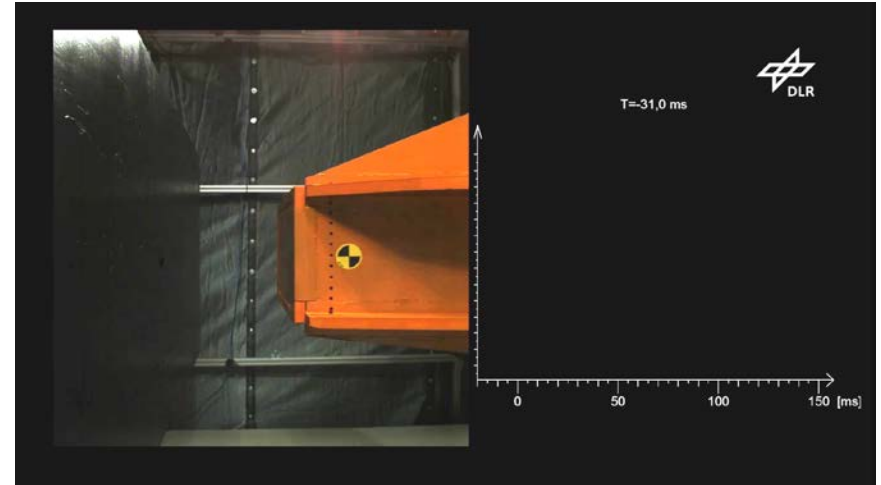
Intrusion		
Crashtest	Reference	Metal monocoque [mm]
Euro-NCAP front ODB	84	102
Euro-NCAP side impact	211	155
Euro-NCAP pole impact	381	207



Euro-NCAP polecrash



Testing of components: Sandwich front structure



- Weight of the front structure: 12 kg
- Integration of various functions in one part:
 - Loads from the chassis
 - Support for various drive-train components
 - Energy absorption in frontal crash load cases
- Relatively uniform force-deformation-curve
- Regular folding of the aluminium layers



Thank you for your attention!



Wissen für Morgen



Automotive Systems