

Damage Tolerance of Sandwich Structures - Modelling, Analysis and Testing -

Jan Teßmer, Luise Kärger, Uwe Pfeiffer, Anja Wetzel

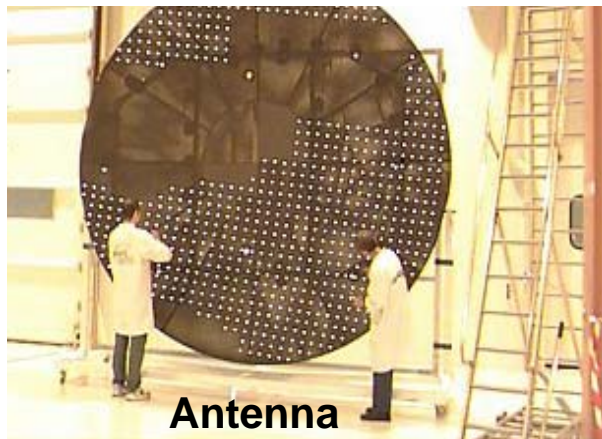
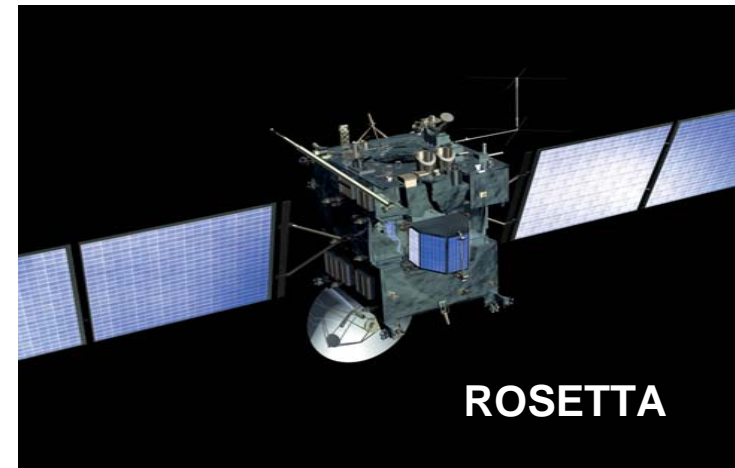
Conference on Damage in Composite Materials, Stuttgart, 18.-19.09.2006



Deutsches Zentrum
für Luft- und Raumfahrt e.V.
in der Helmholtz-Gemeinschaft



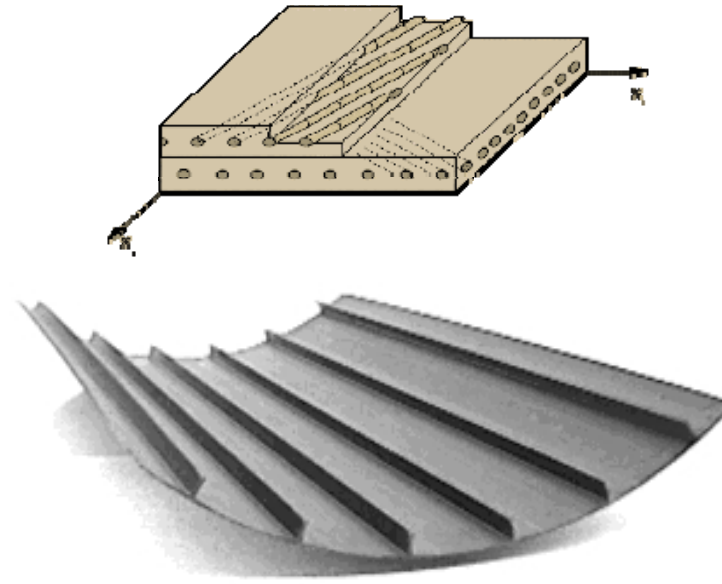
High Performance Composites in Aerospace Structures



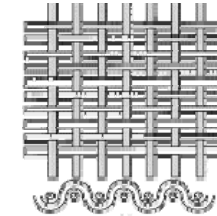


High Performance Composites in Aerospace Structures

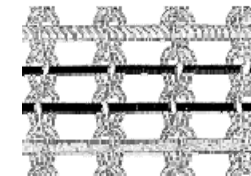
Monolithic Composites



Sandwich Structures with Composite Face sheets



Textile Composites

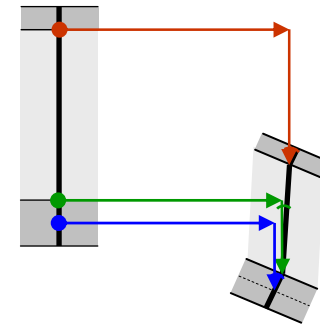
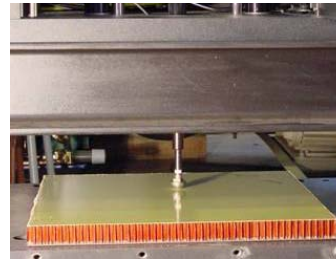




1 Double shell structures (Sandwiches)

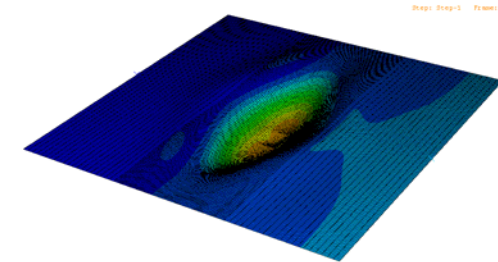
2 Impact analysis

Experiments
Simulation tool CODAC
Damage Modelling
Simulation results



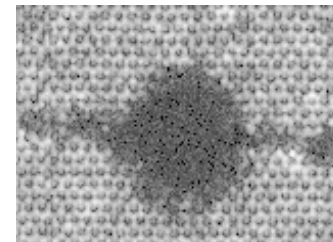
3 Residual strength analysis

CAI-Tests
Geometrical non-linear simulation using ABAQUS
Simulation results



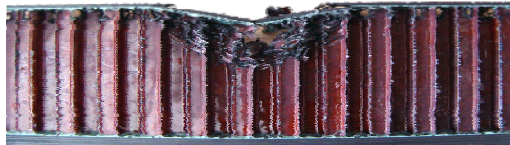
4 Non-destructive Testing

Air-coupled Ultrasonic Testing for Sandwiches
Signal Processing for monolithic CFRP (skins)





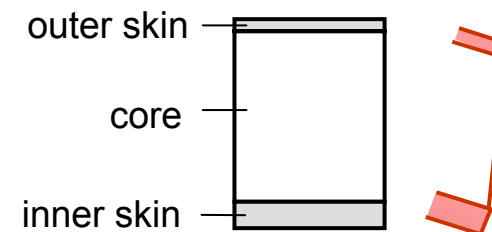
1 Double shell structures



Modelling requirements

by accounting for the specific deformation behaviour:

- fast for being used in the design process
- sufficiently accurate



Development of two new shell elements

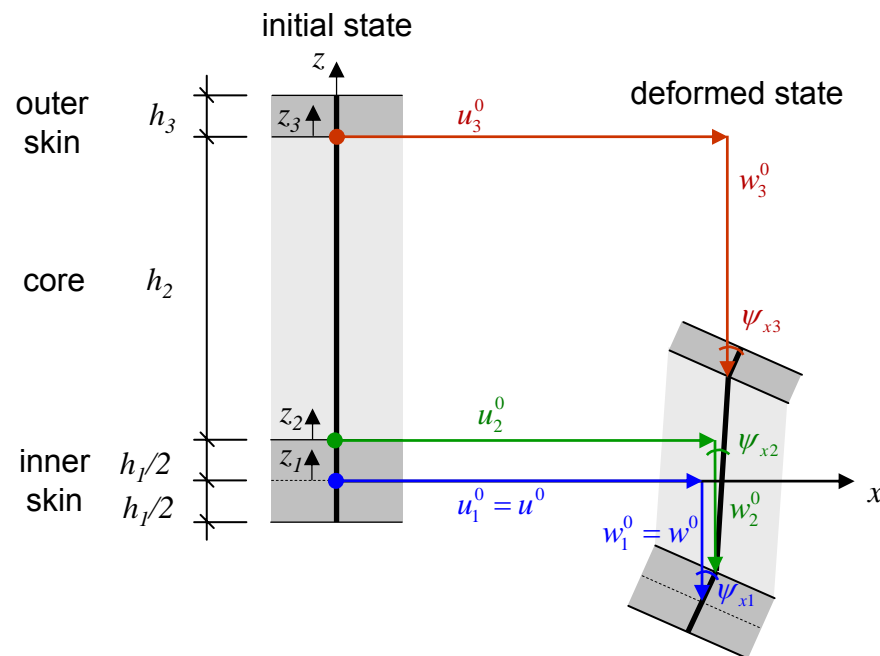
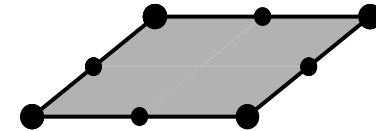
Ref.:

Element S89: *Kärger, Wetzel, Rolfes, Rohwer. Computers & Structures 84. 2006.*

Element S815: *Wetzel, Kärger, Rolfes, Rohwer. Computers & Structures 83. 2005.*

1 Double shell structures

Element S815: 3-layered shell element with 3-D stress analysis
(**S**andwich element with **8** nodes and **15** dof per node)



Kinematics of layer L :

$$\begin{bmatrix} u_L \\ v_L \\ w_L \end{bmatrix} = \begin{bmatrix} u_L^0 \\ v_L^0 \\ w_L^0 \end{bmatrix} + z_L \begin{bmatrix} \psi_{xL} \\ \psi_{yL} \\ \psi_{zL} \end{bmatrix} + z_L^2 \begin{bmatrix} 0 \\ 0 \\ \varphi_{zL} \end{bmatrix}$$

→ 15 dof per node

Layer-wise full 3D-material law:

$$\sigma_L = \mathbf{C}_L \epsilon_L$$

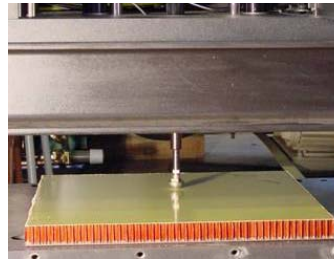
- Stress computation:**
- in-plane stresses: material law
 - transv. shear stresses: equilibrium approach by Rolfes & Rohwer
 - transv. normal stress: material law and equilibrium approach



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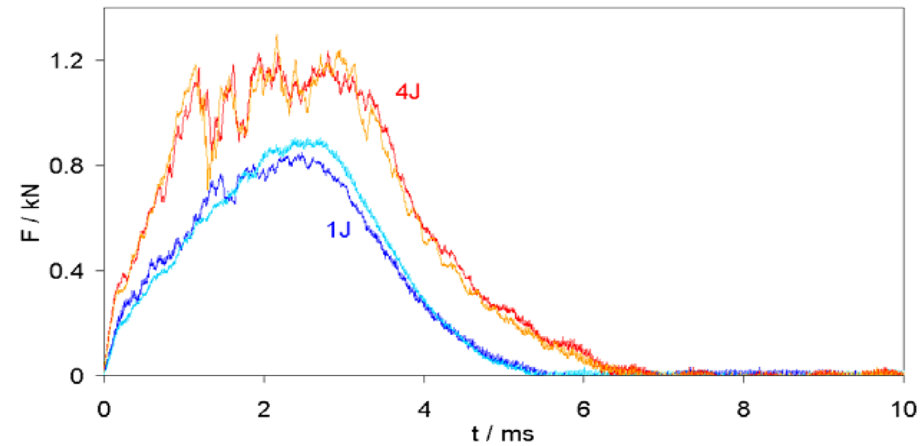




2 Impact analysis: Experiments

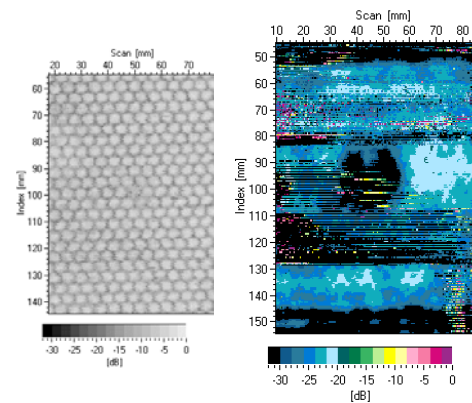
Tests conducted at ILR, TU Dresden

Force-time histories:



completely supported panel

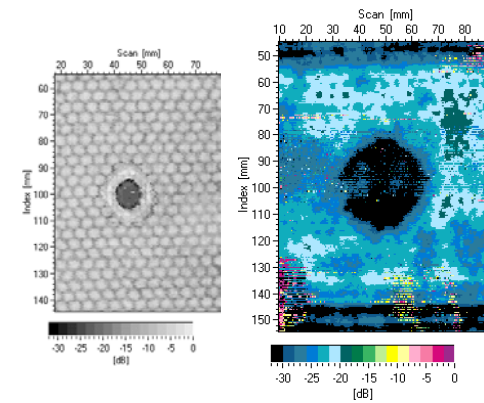
1 Joule damage:



top skin

core

4 Joule damage:



top skin

core



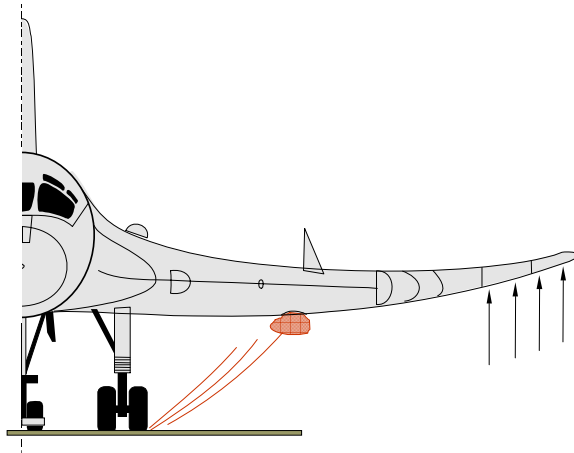
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Institute of Composite Structures and Adaptive Systems

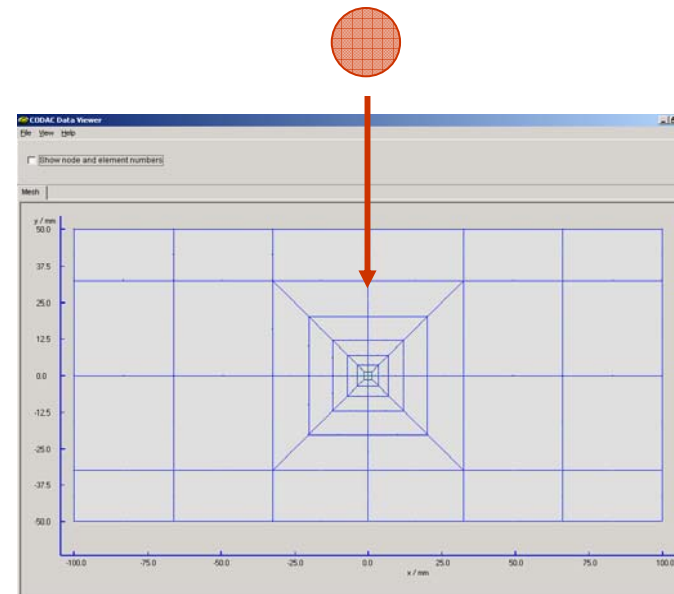
2 Impact analysis: Simulation tool CODAC

- **CODAC = Composite Damage Tolerance Analysis Code**
- fast evaluation of impact damage and residual strength of composite structures
- Finite Element Method



Impactor ($\varnothing 25.4$ mm):

- point mass
- parabolically distributed surface load



Transient impact analysis:

- dynamic FEA with Newmark time integration
- application of Hertzian contact law



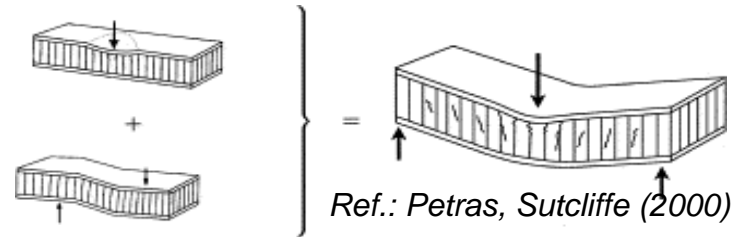
2 Impact analysis: Modelling of core damage

Ref.: Kärger, Baaran, Teßmer. *Composite Structures*. 2006

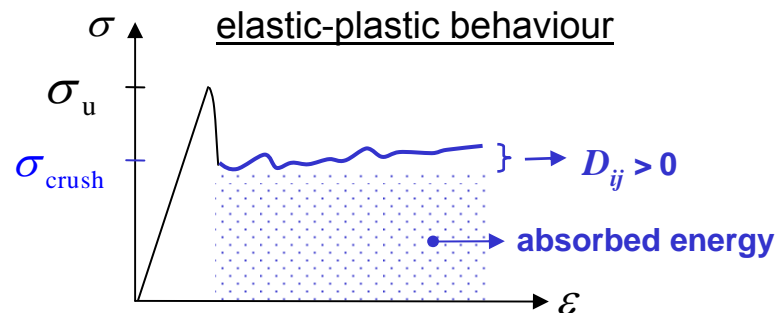
Failure criterion:

- transverse shear and compression failure of honeycomb core
- criterion by Besant et al.:

$$\left(\frac{\sigma_{zz}}{\sigma_{cc}} \right)^n + \left(\frac{\tau_{xz}}{\tau_{cls}} \right)^n + \left(\frac{\tau_{yz}}{\tau_{cts}} \right)^n \geq 1$$



Degradation:



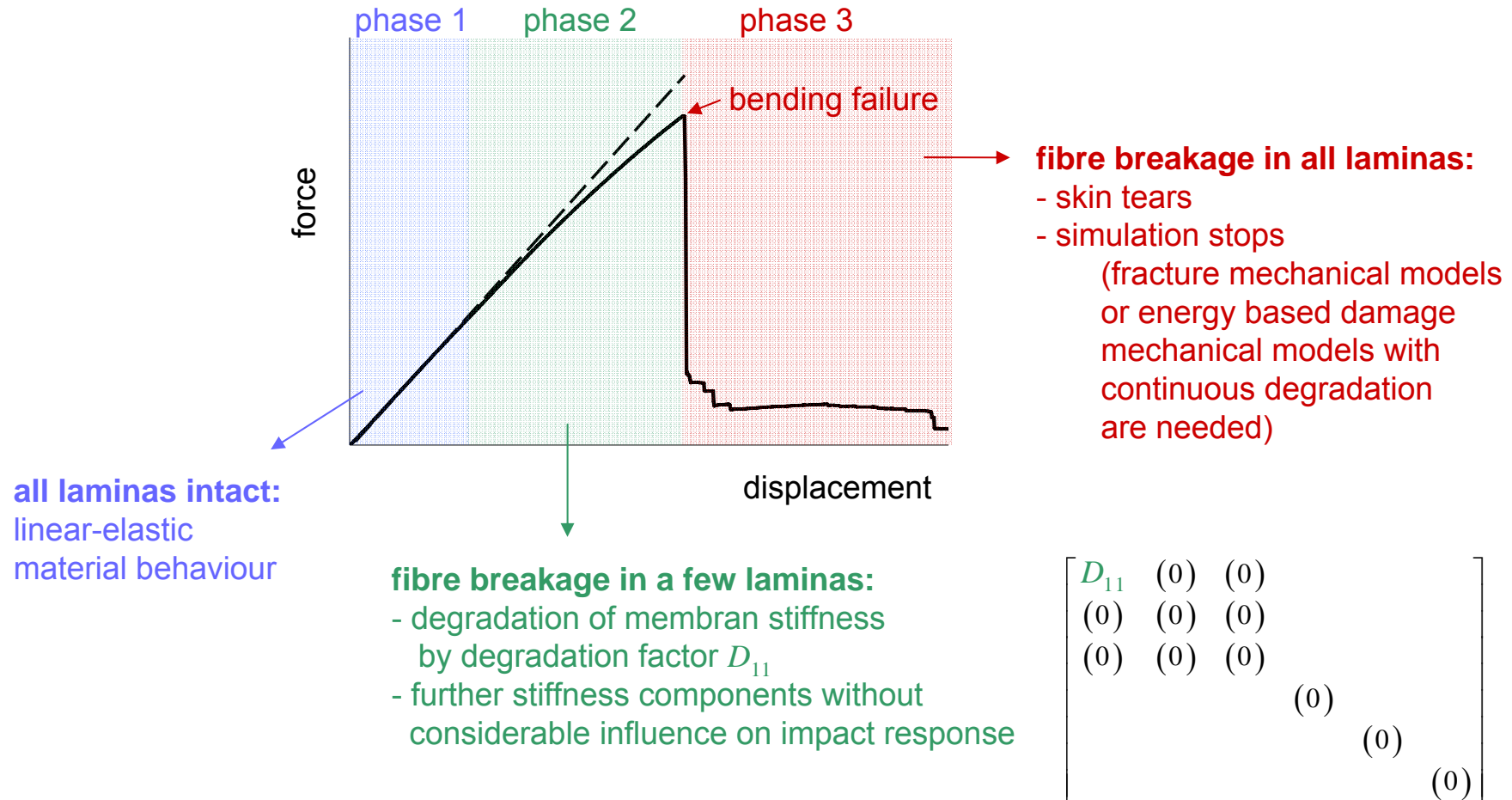
$$\begin{bmatrix} \sigma_{xx}^* \\ \sigma_{yy}^* \\ \sigma_{zz}^* \\ \tau_{yz}^* \\ \tau_{xz}^* \\ \tau_{xy}^* \end{bmatrix} = \begin{bmatrix} D_{11}C_{11} & D_{12}C_{12} & D_{13}C_{13} & & & \\ & D_{22}C_{22} & D_{23}C_{23} & & & \\ & & D_{33}C_{33} & & & \\ & & & D_{44}C_{44} & & \\ & & & & D_{55}C_{55} & \\ & & & & & D_{66}C_{66} \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ \gamma_{yz} \\ \gamma_{xz} \\ \gamma_{xy} \end{bmatrix}$$

sym



2 Impact analysis: Modelling of skin damage

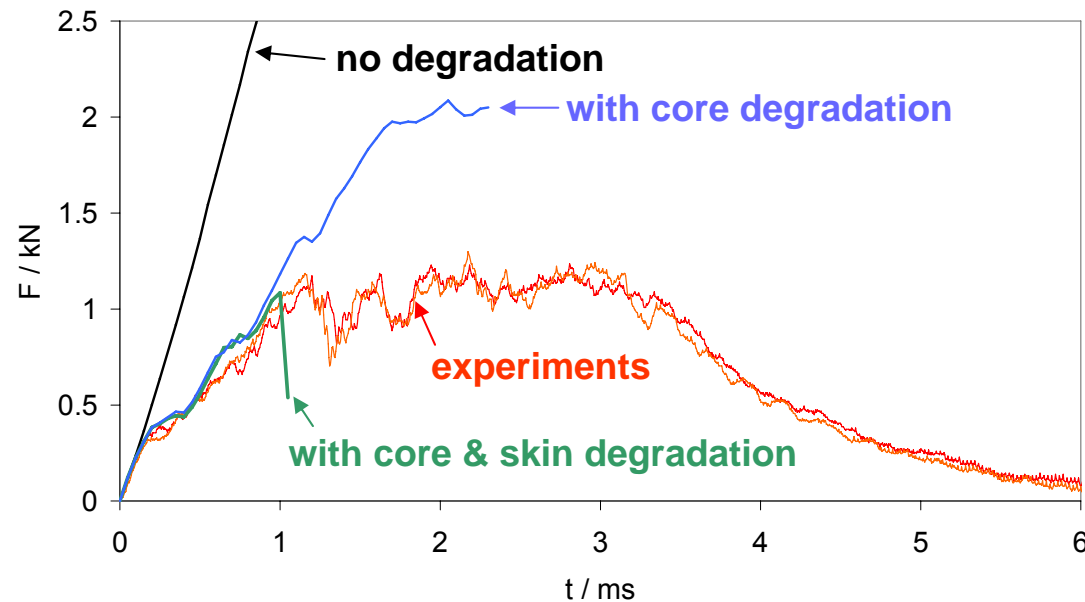
3-point bending test:



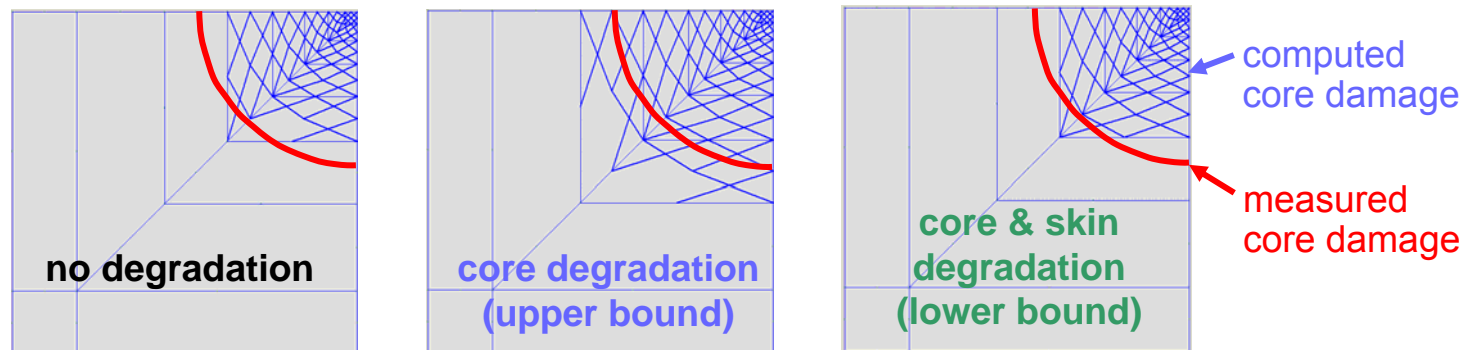
2 Impact analysis: Simulation results

4J Impact

Force time history:



Core damage:





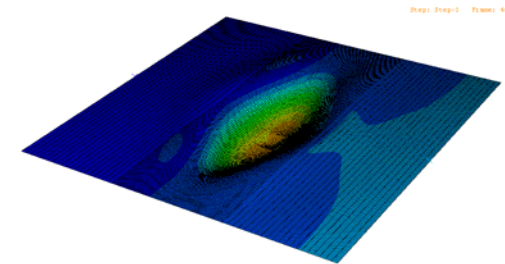
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2 Impact analysis

- Experiments
- Simulation tool CODAC
- Damage Modelling
- Simulation results

3 Residual strength analysis

- CAI-Tests
- Geometrical non-linear simulation using ABAQUS
- Simulation results

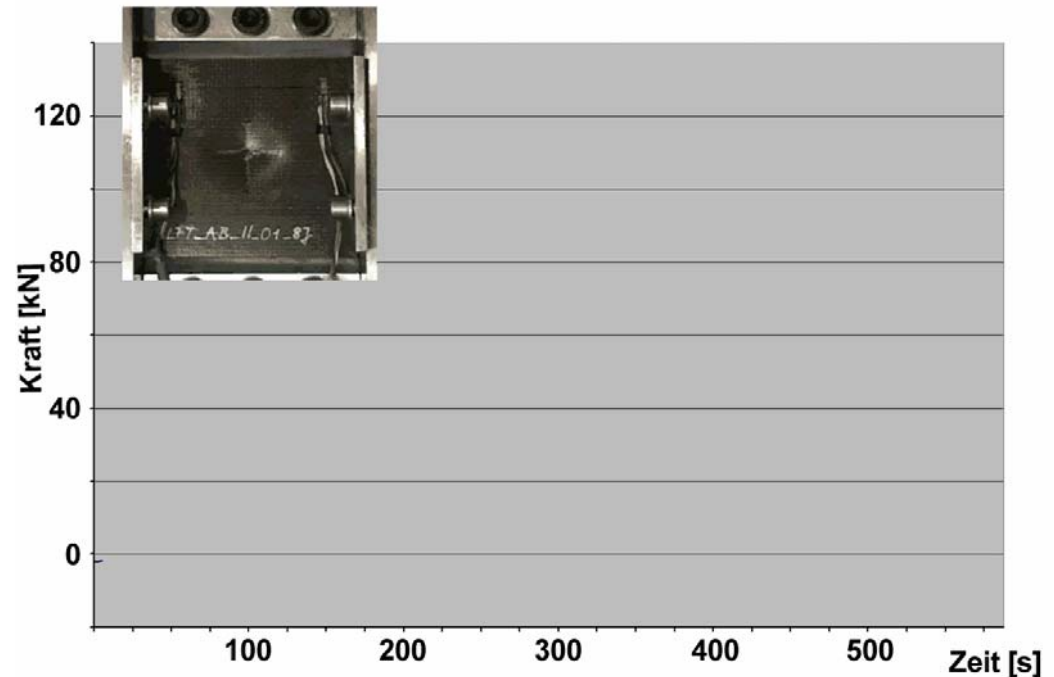
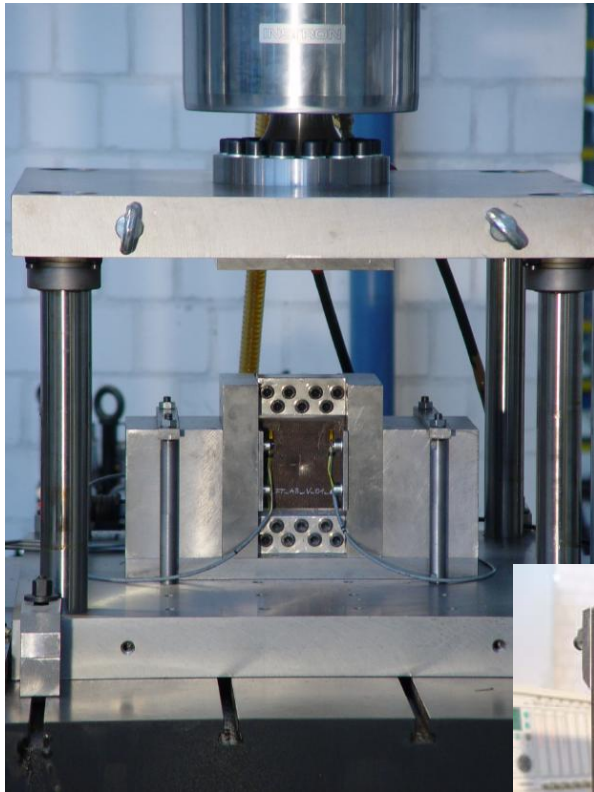


4 Non-destructive Testing

- Air-coupled Ultrasonic Testing for Sandwiches
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3 Residual strength analysis: CAI-Tests

CAI-Test-Equipment:

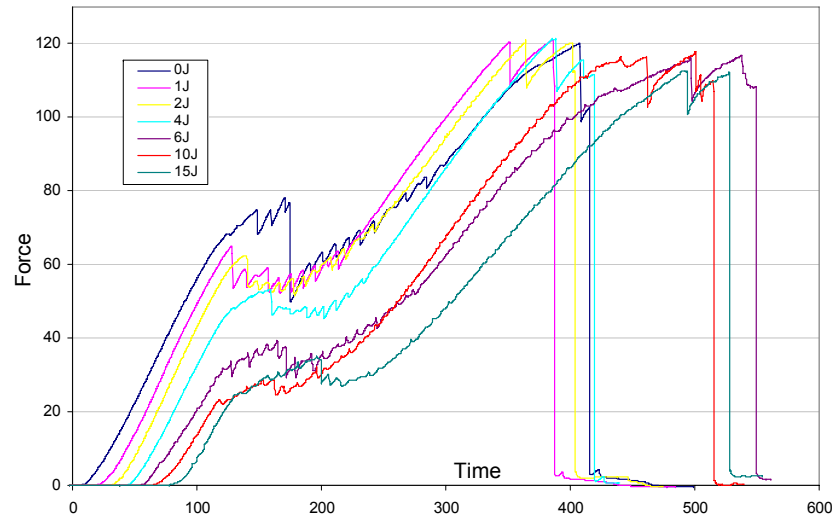


Failure phenomenon:

- impacted face sheet: dent propagation transverse to the loading direction
- failure of the impacted face sheet: buckling across the whole specimen width → sudden load decrease
- further load increase
- failure of second face sheet

Source: ILR TU Dresden

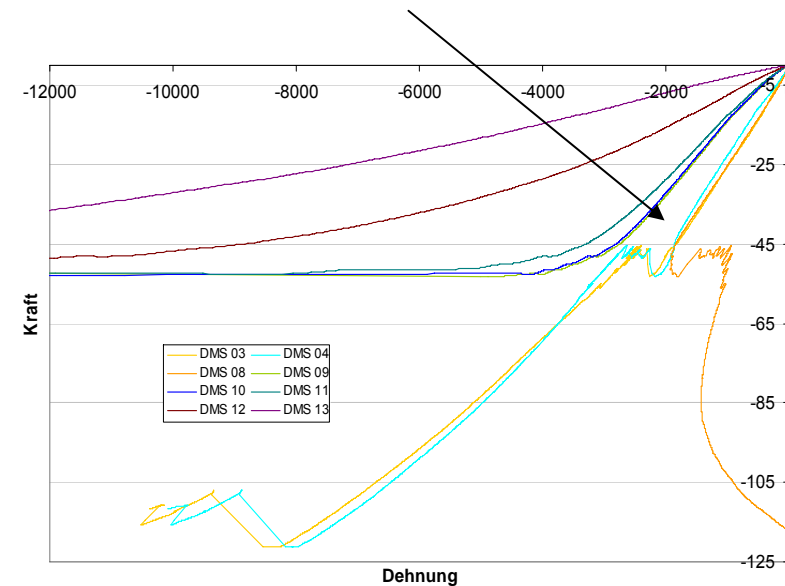
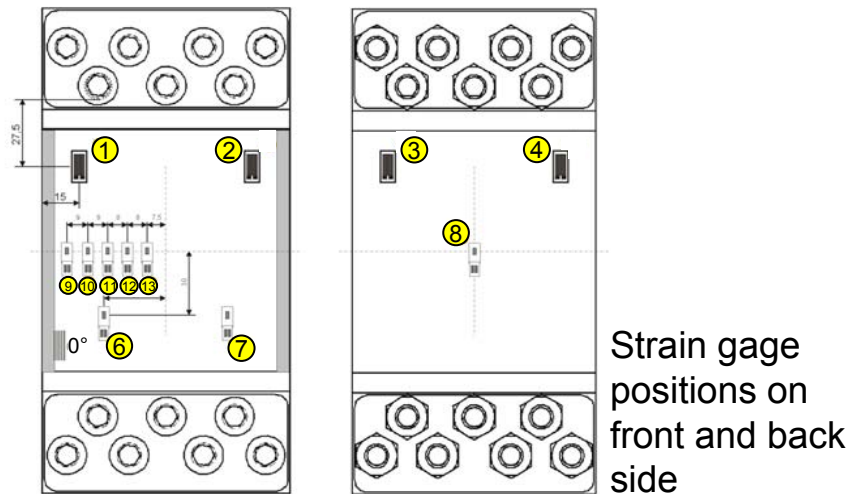
3 Residual strength analysis: CAI-Tests



Test data:

- Force vs. time
- Strain vs. time at specified location

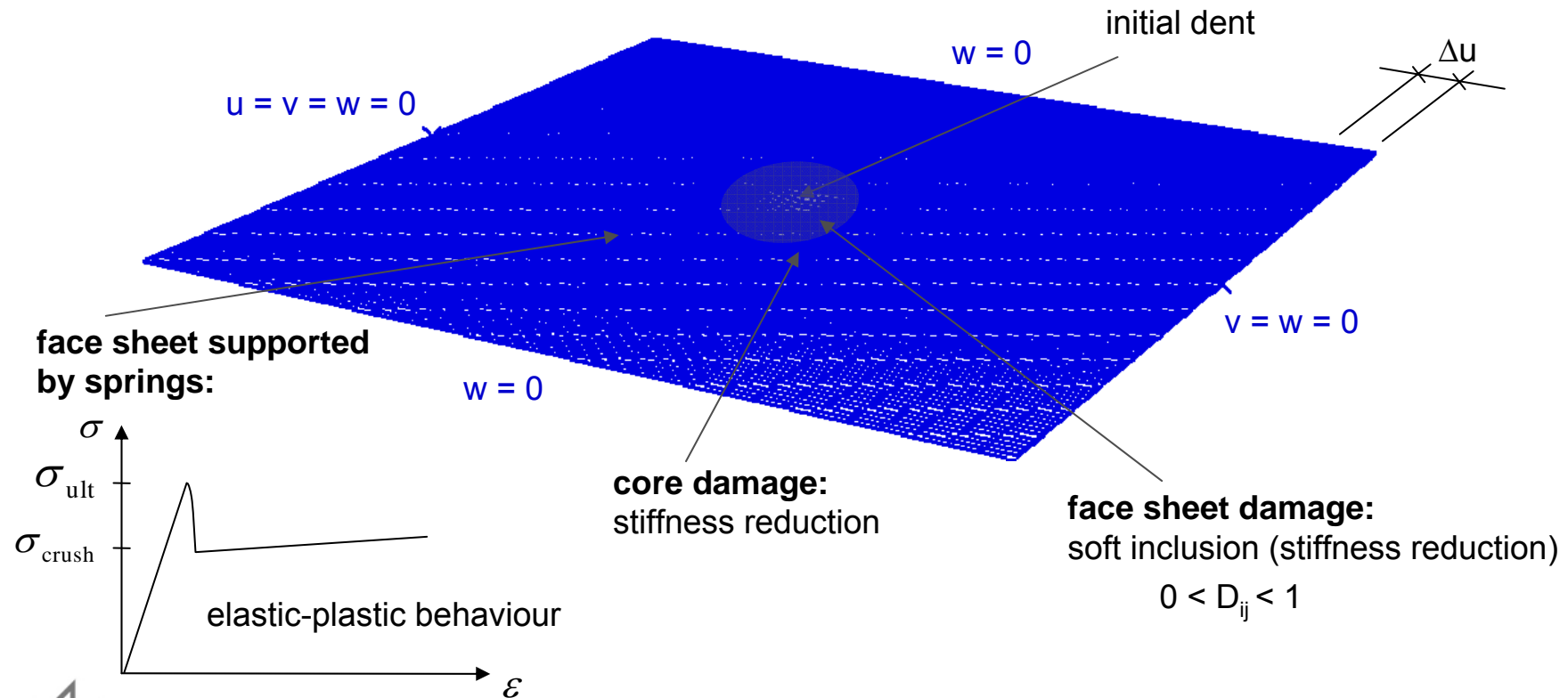
Force vs. strain for 4 Joule specimen:
higher compression on the front side
compared to the back side (bending)



3 Residual strength analysis: Non-linear simulation using ABAQUS

Non-linear FE analysis of impacted face sheet:

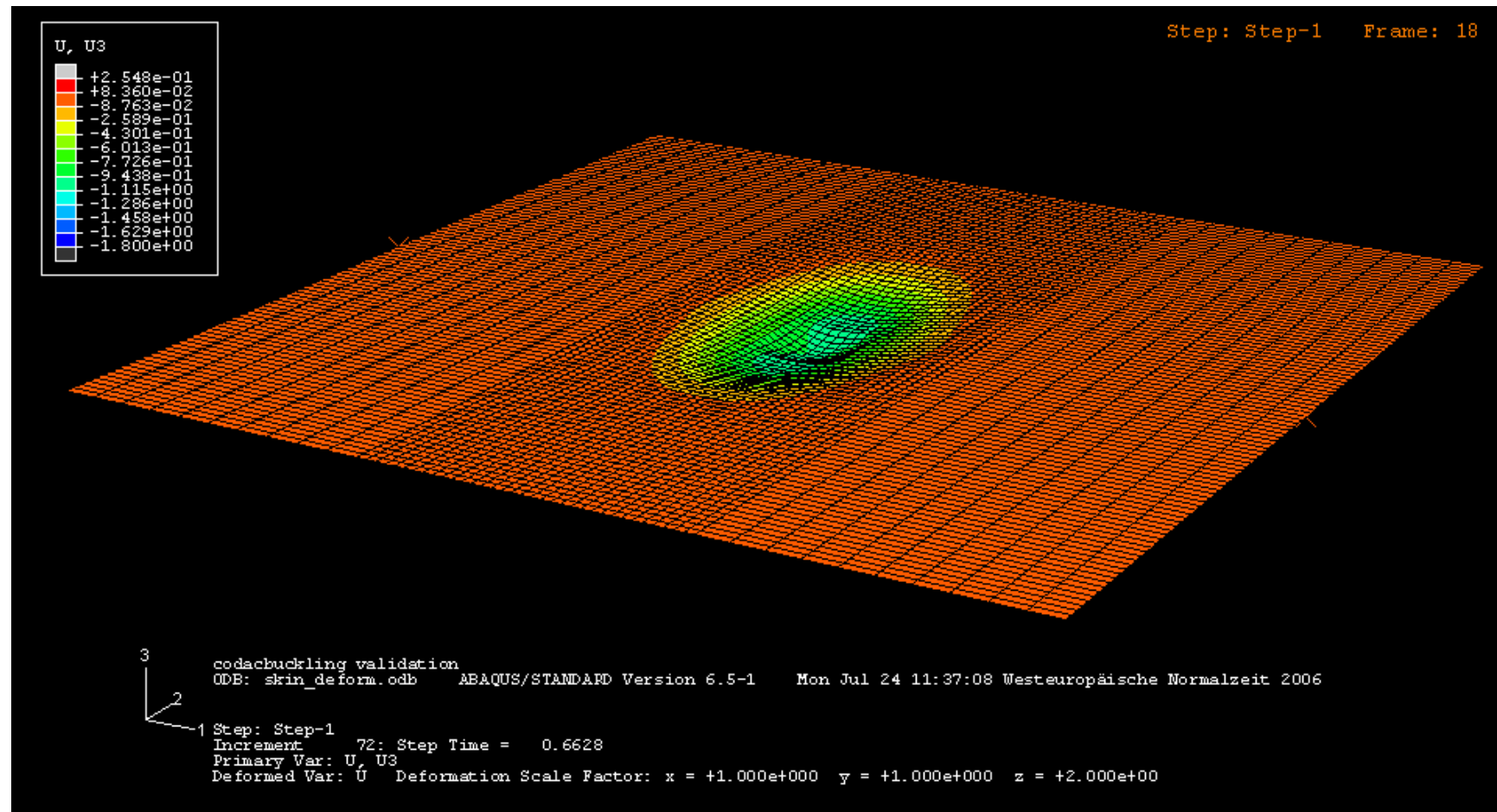
- uniaxial in-plane loading (displacement-driven, Δu)
- face sheet supported by springs representing the core
- including initial dent, face sheet and core damage due to impact
- including core damage growth
- using automatic stabilization because of local instabilities



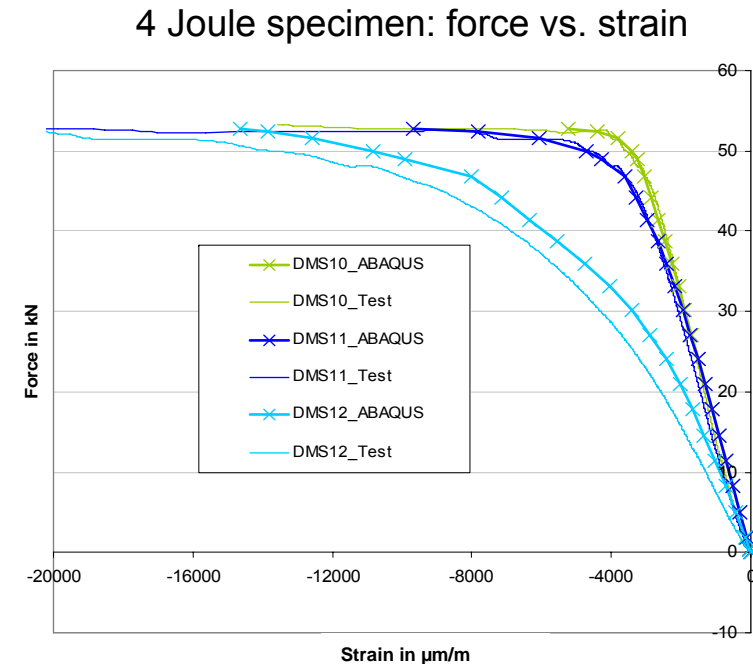
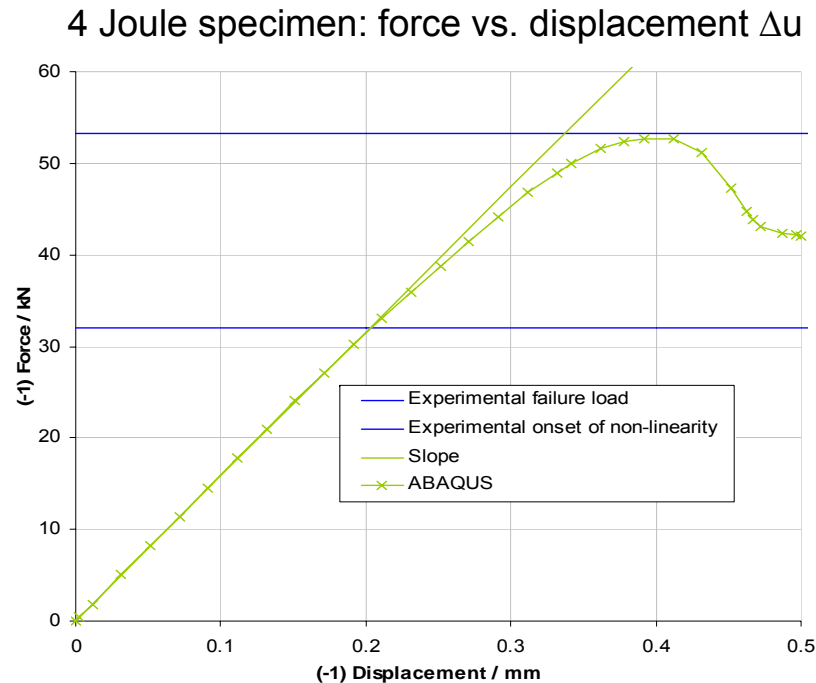


3 Residual strength analysis: Simulation results

Dent growth with increasing in-plane loading:



3 Residual strength analysis: Simulation results



For 4 Joule damage, $D_{ij}=0.7$, $k_{core}=E_{zz}/h$, E_{zz} and σ_{ult} according to data sheet, $\sigma_{plat}/\sigma_{ult} = 0.3$:

- very good correlation between experimental failure load and maximum load of ABAQUS simulation
- good correlation between experiment and ABAQUS simulation for strains at strain gage locations



1 Double shell structures

2 Impact analysis

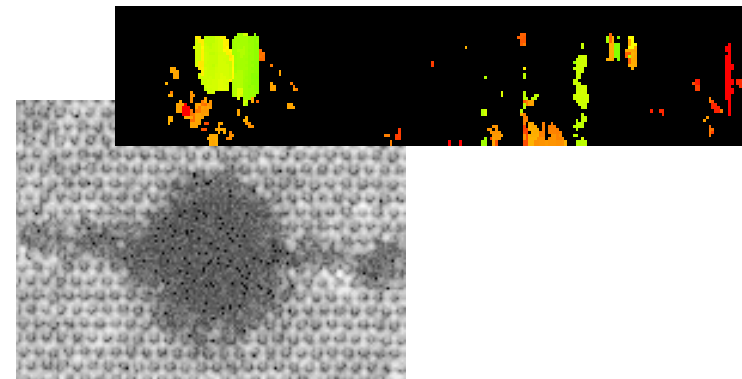
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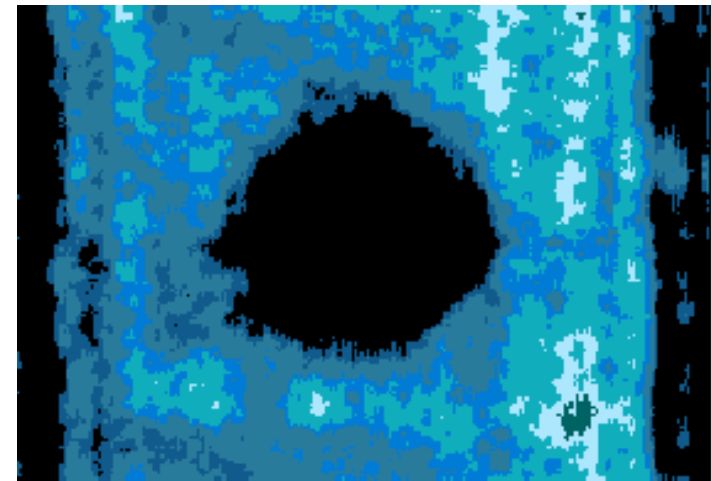
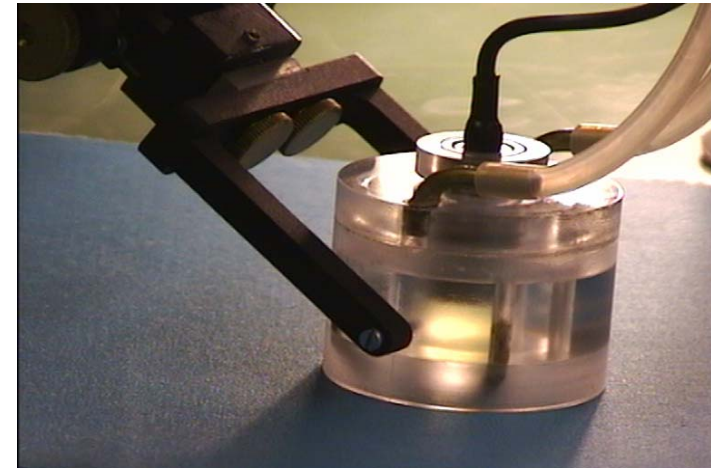
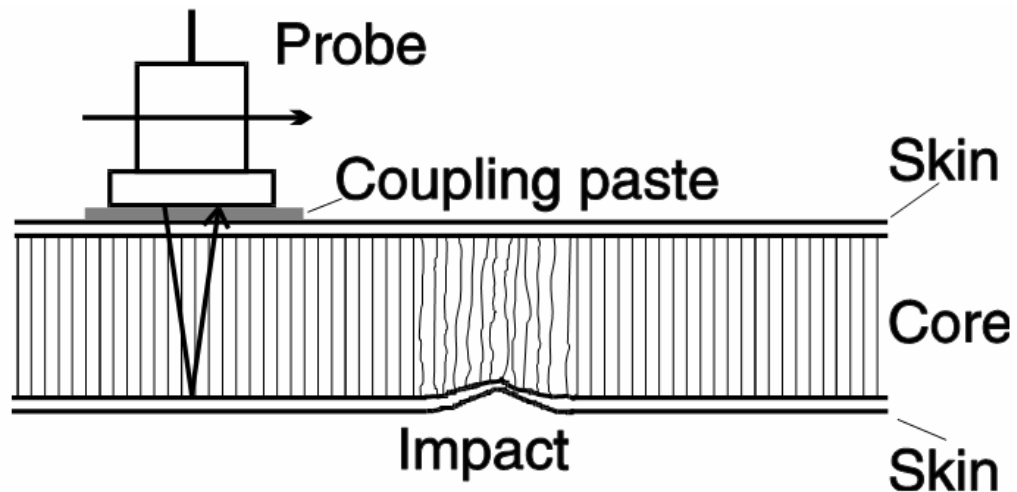
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4 Non-destructive Testing

Air-coupled Ultrasonic Testing for Sandwiches
Signal Processing for monolithic CFRP (skins)



Sandwich inspection with water coupling



Optimal frequency for Sandwiches: $<500\text{kHz}$
=> Bad Focussing,
low spatial resolution



Air-coupled Ultrasonics

Benefits:

- Contact free, without coupling media
- Constant and reproducible coupling
- No incoming water
- <500kHz *and* focussed

Challenges:

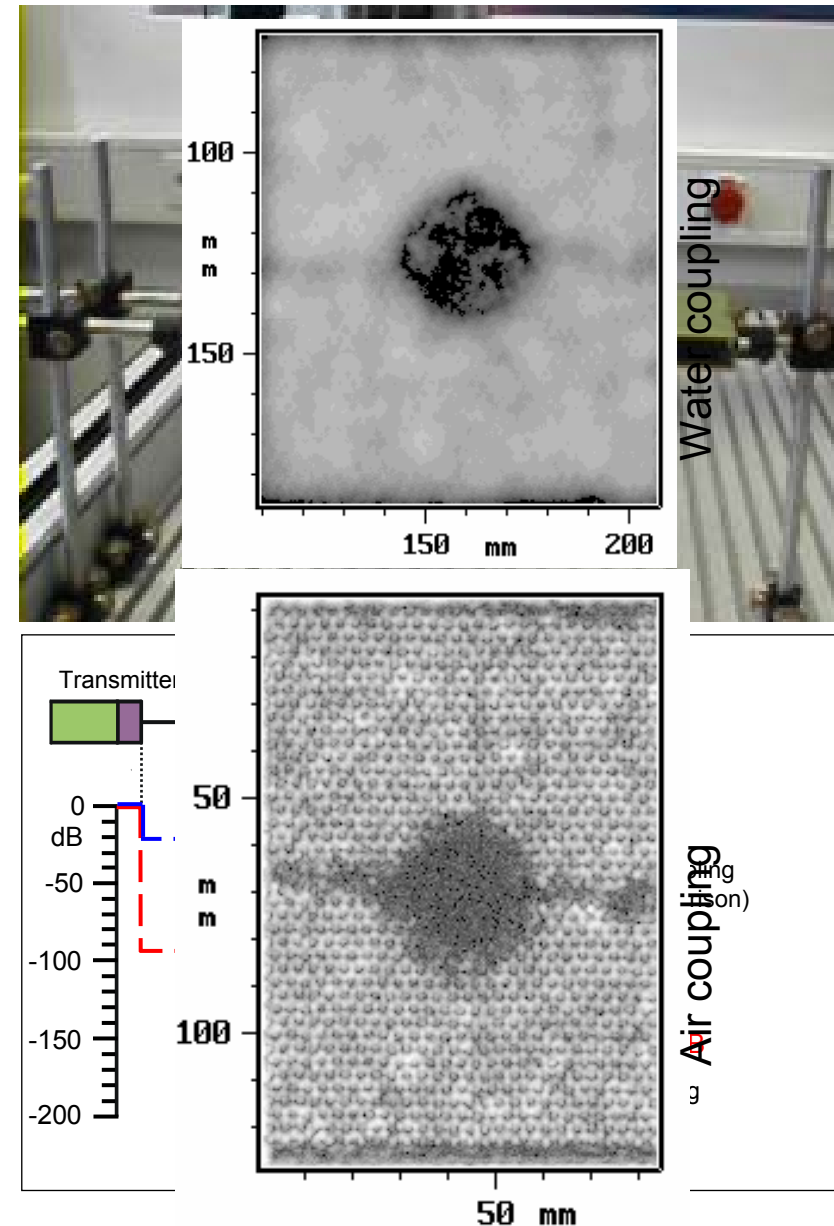
- Bad acoustic matching
- More than 160 dB amplitude loss

Approach

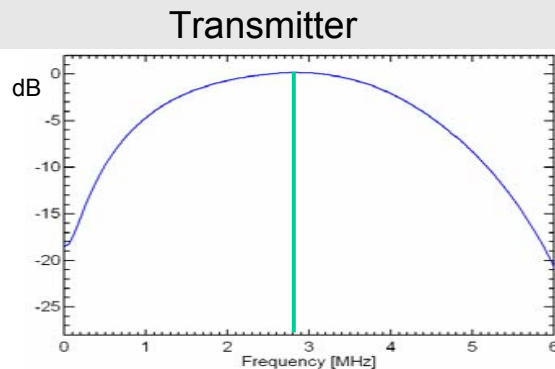
- Transducers with optimised matching layers
- Optimised transmitter and receiver electronics

Our results

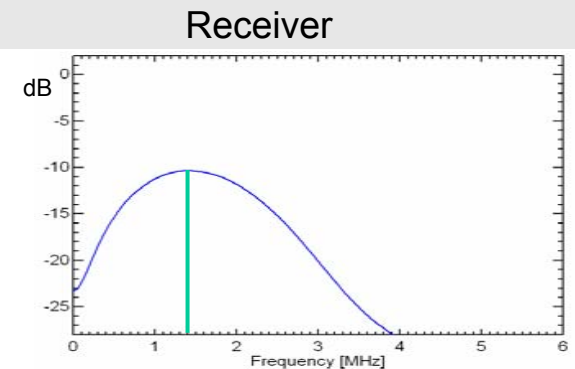
- Signal-to-noise ratio in transmission: 30 dB
- Narrow band, strongly focussing transducers
- Best choice for sandwich testing



CFRP: High and frequency-specific sound attenuation

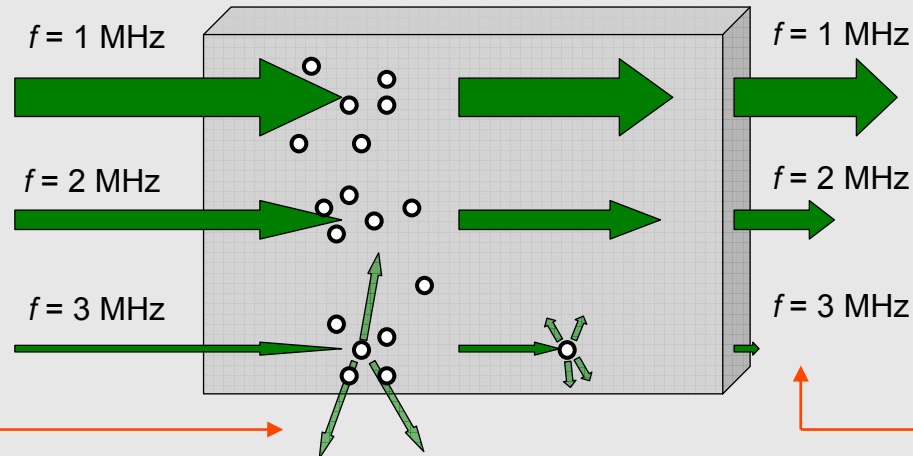


Arrow thickness ~ wavelength
 Arrow length ~ amplitude
 ○ = signified microstructure



delivered signal:
 E.g. same amplitude
 at 1, 2 and 3 MHz

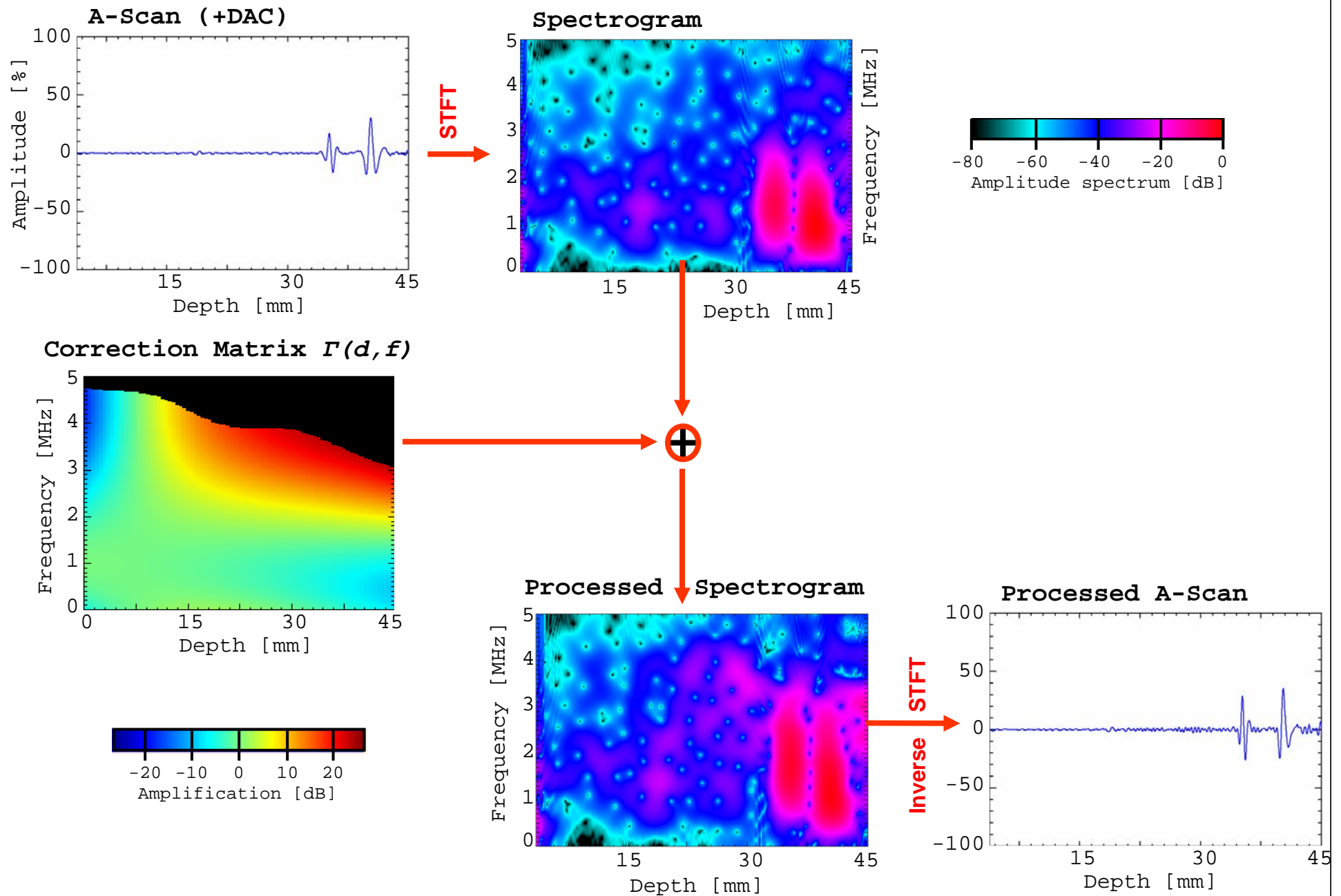
Loss by scattering is
 dominating at high
 frequencies



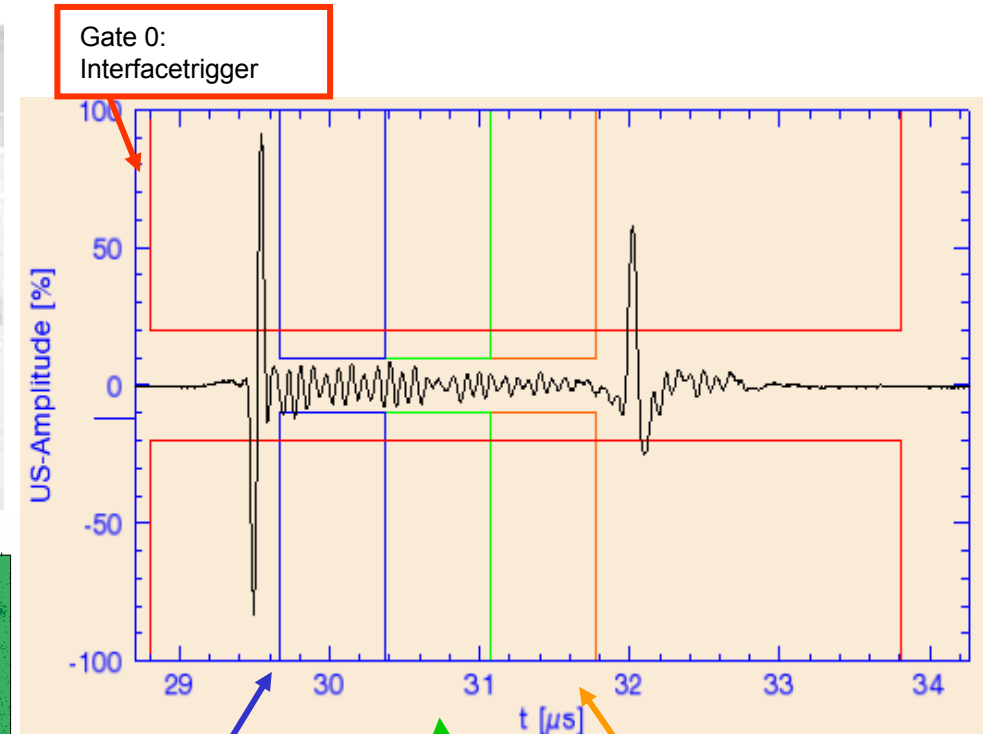
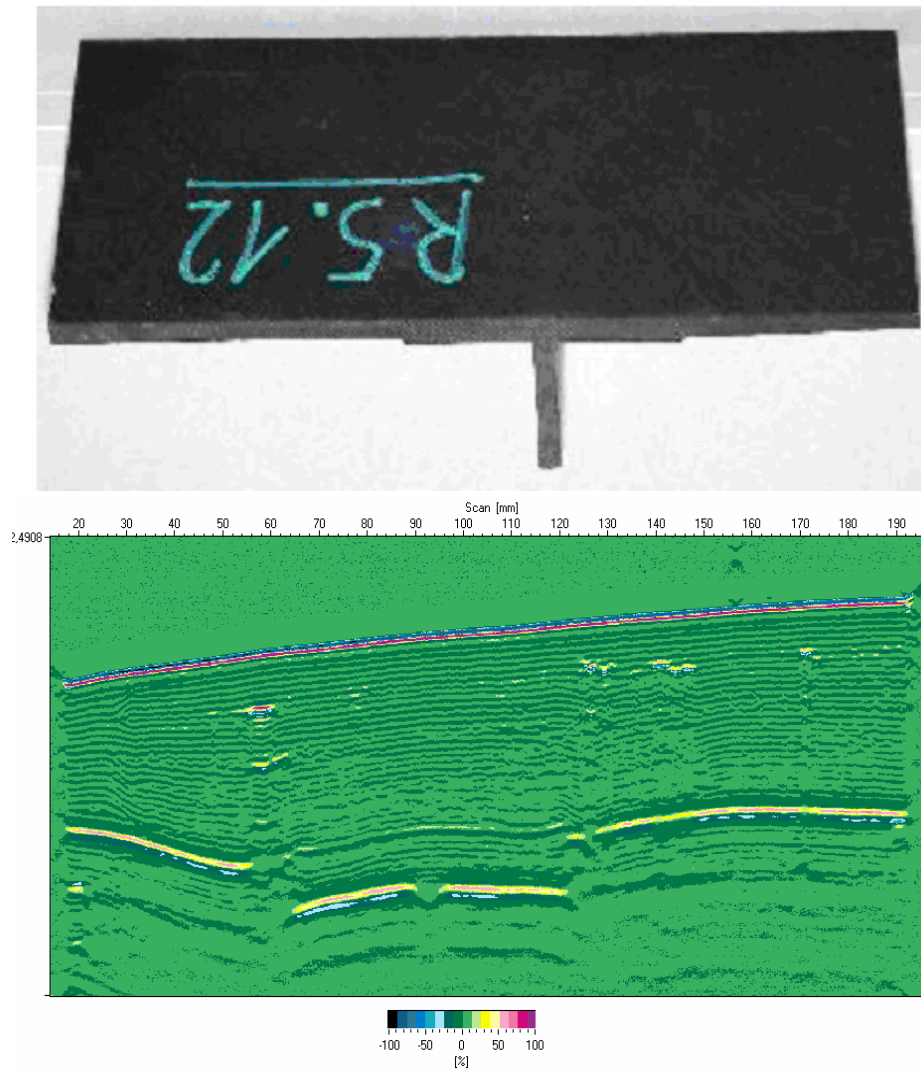
Transmission:
 Spectrum shifted to
 low frequencies

The HF fraction is
 attenuated
 superproportional

Spectral Distance Amplitude Correction „SDAC“



Stringer Specimen: Analysis of Layer Echoes



Gate 1:
0.2 – 0.8 μs
„shallow“

Gate 2:
0.8 – 1.4 μs
„middle“

Gate 3:
1.4 – 2.0 μs
„deep“



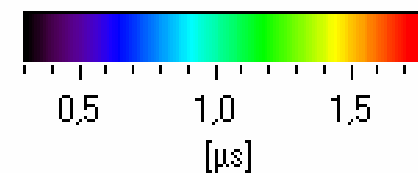
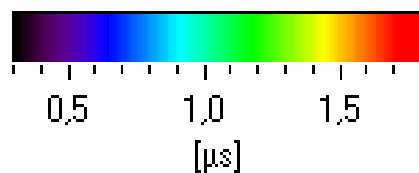
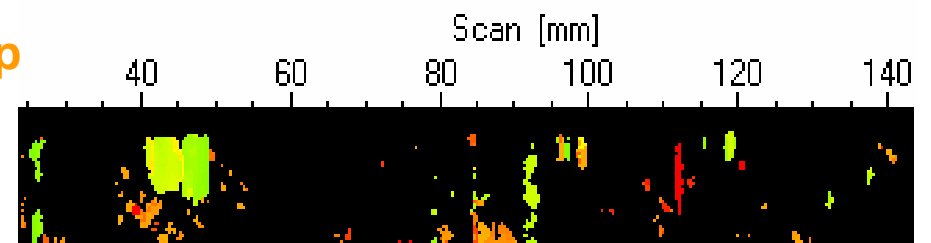
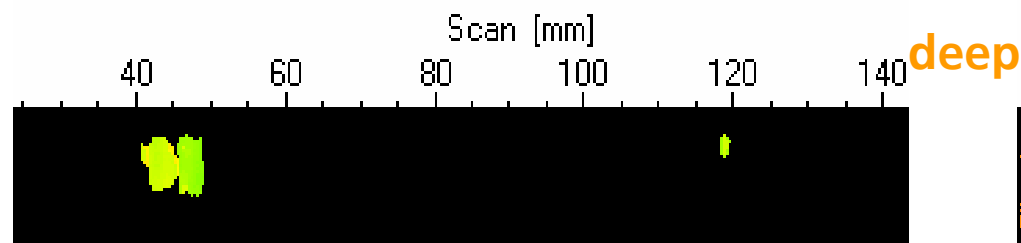
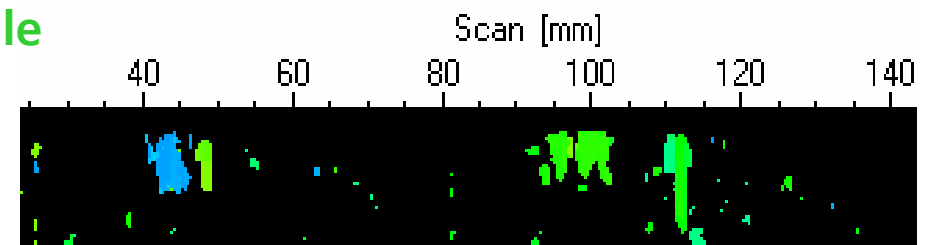
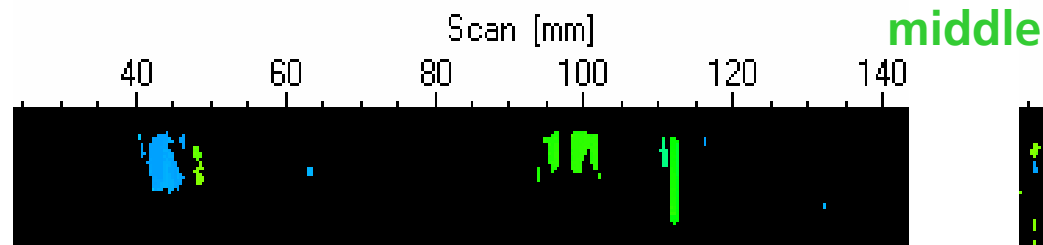
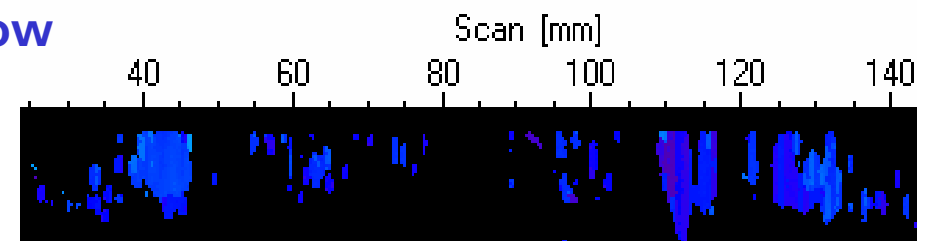
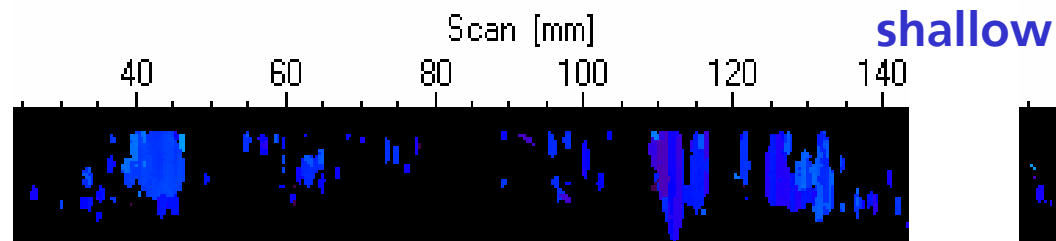
Flaw Detection in various depths

(Time-of-Flight D-Scans)

SDAC: Spectral Distance Amplitude Correction

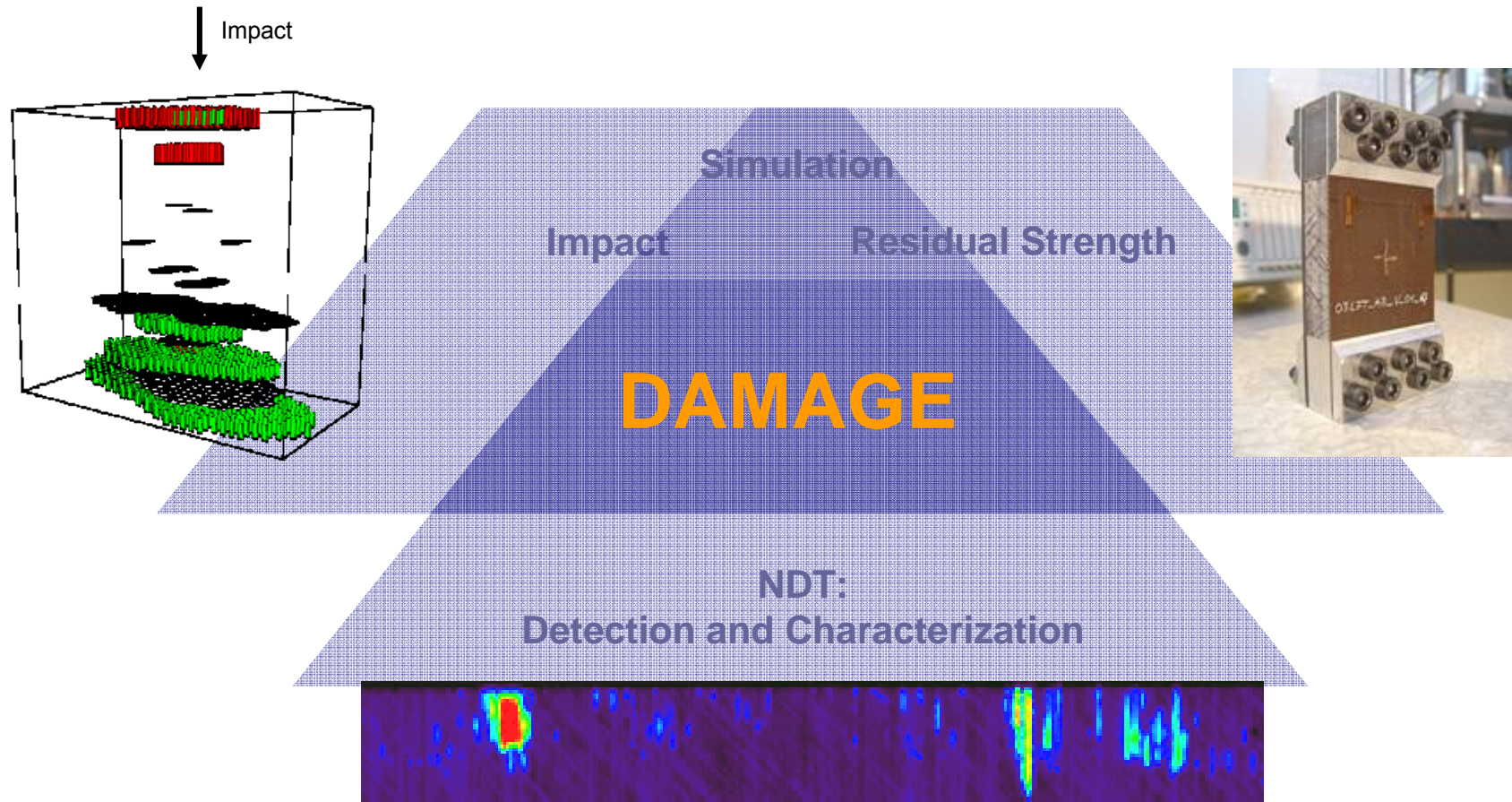
unprocessed

SDAC processed



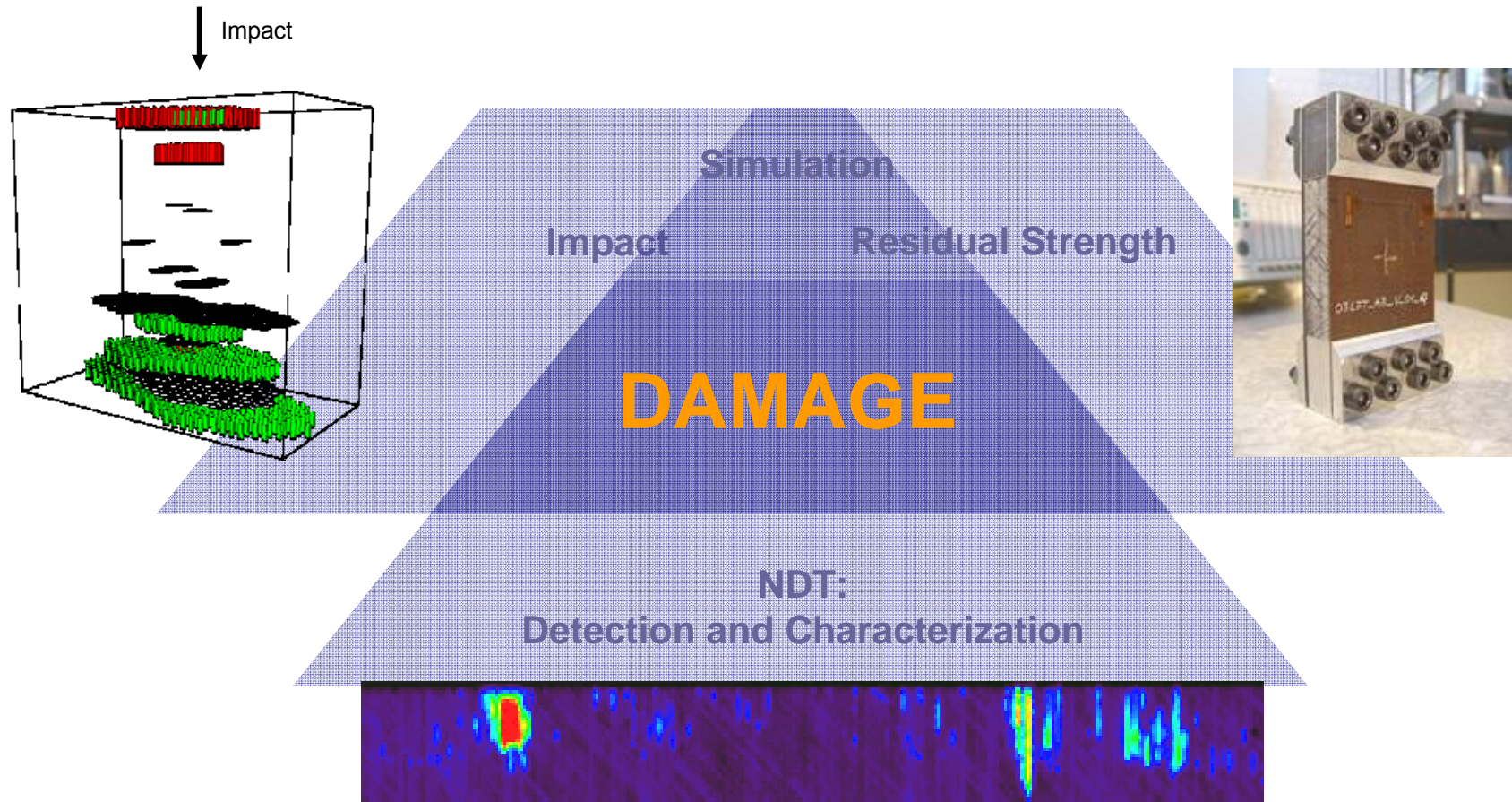


High Performance Composites in Aerospace Structures





High Performance Composites in Aerospace Structures



Thank you for your Attention!
Contact: jan.tessmer@dlr.de