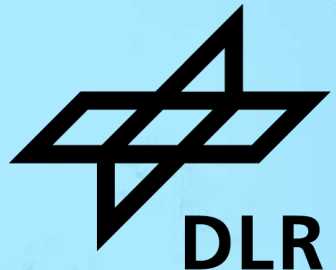


EXPERIMENTAL RESULTS OF A FLUID ACTUATED MORPHING WINGLET TRAILING EDGE

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Experimental Results of a Fluid Actuated Morphing Winglet Trailing Edge



Outline

- Motivation
- Structural Design
- Systems Design
- Experimental Results
- Conclusion and Outlook



Supported by the Clean Sky 2 Joint Undertaking (JU)

MANTA – „Movables for Next Generation Aircraft“

AEROMO2 – „Towards the application of Morphing Movables in Aerostructures“

The JU is financed by the Horizon 2020 Research-Programme of the European Union

Motivation Application

Winglet-Tab

- Reduced induced drag, but increased load
- Reduction of load desirable → Potential for reduced weight
- Active load-reduction by integration of Winglet tab as control surface

- Small available volume, thin profiles
- Big distance from fuselage

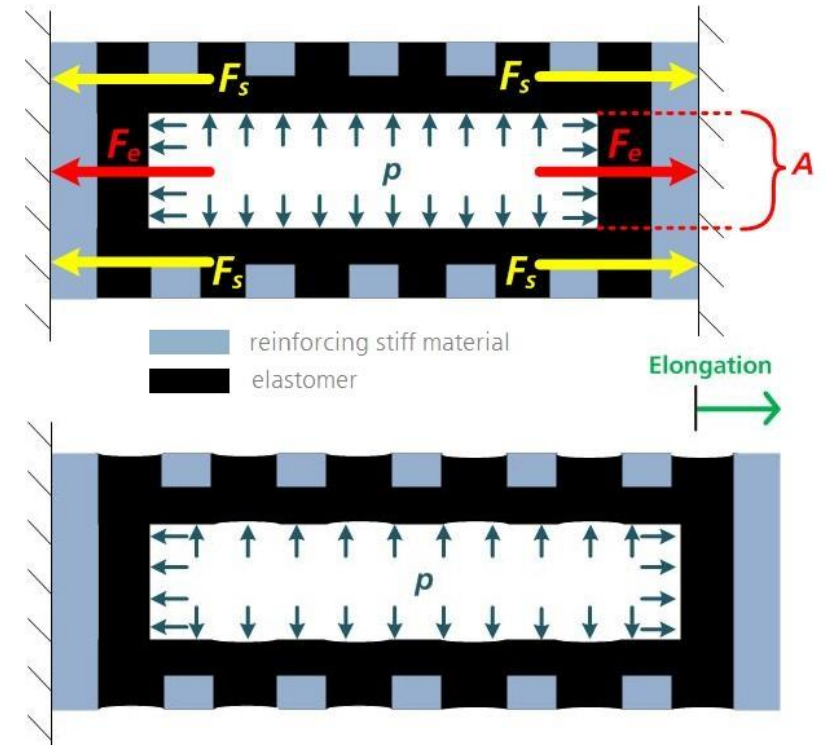
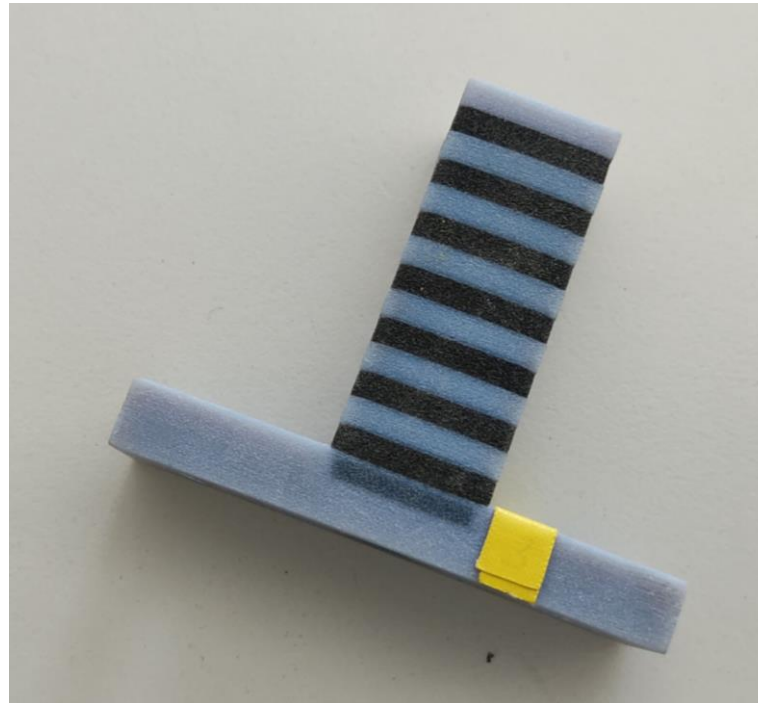


Motivation

Working Principle - Objectives

FAMoUS

- Fluid Actuated Morphing Unit Structures
- Look into feasibility
- Raise TRL
 - Starting at TRL 1
 - Goal: TRL 2-3
- Build structure AND system



Motivation Concept for Winglet tab

Challenges:

stretchable skin + flexible inner structure + fluid pressure:
integrated system contributing to both stiffness and flexibility

height only 45 mm

aluminium stiffening rings

EPDM elastomer chamber

smooth curvature transition



fluid in



fluid out



bimorph capability



distributed miniature electro-hydrostatic actuation (EHA)

GFRP laminate with integrated fiber optic shape sensors

adequate moment authority & rate

clean surface:
no actuator protrusion, no fairings
(fluid pumps fit in the profile forward of the spar)

Motivation Requirements



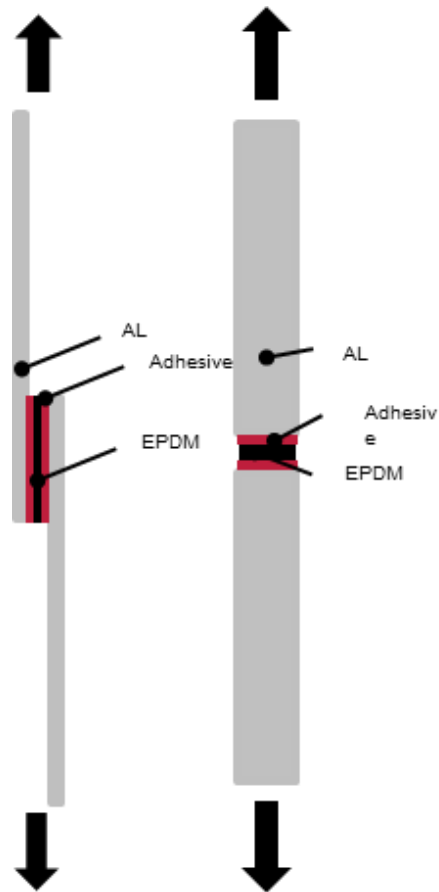
	1 st Priority		2 nd Step
Function	High Speed Performance	Maneuver load alleviation	Gust load alleviation
Deflection Range	+/- 10°	+/- 15°	+/- 15°
Deflection Speed	10°/s	20°/s	80°/s
Position Accuracy	+/- 0,1° Rationale: to ensure device position compliant with aerodynamic tolerance		
Design Assumptions	Either 2 actuators or 1 actuator and damper Rationale: Damper to avoid flutter in case of actuator failure		
System Architecture	Hydraulics is baseline, MEA (More Electric Actuation) as trade		



STRUCTURAL DESIGN

Structural Design Design steps

Coupon / Material

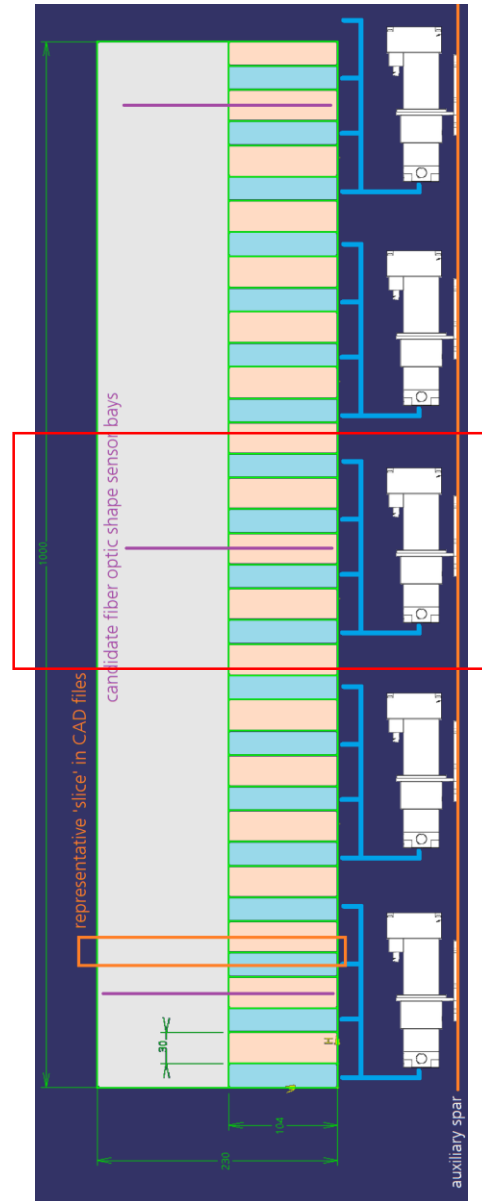
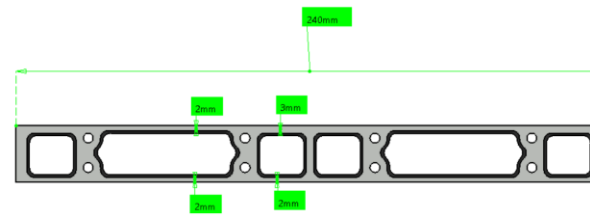
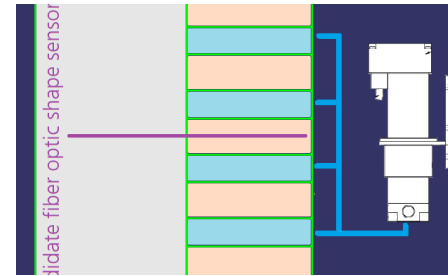


Singles Unit-Structure



1 m Demonstrator

Subsystem



Structural Design FE Model

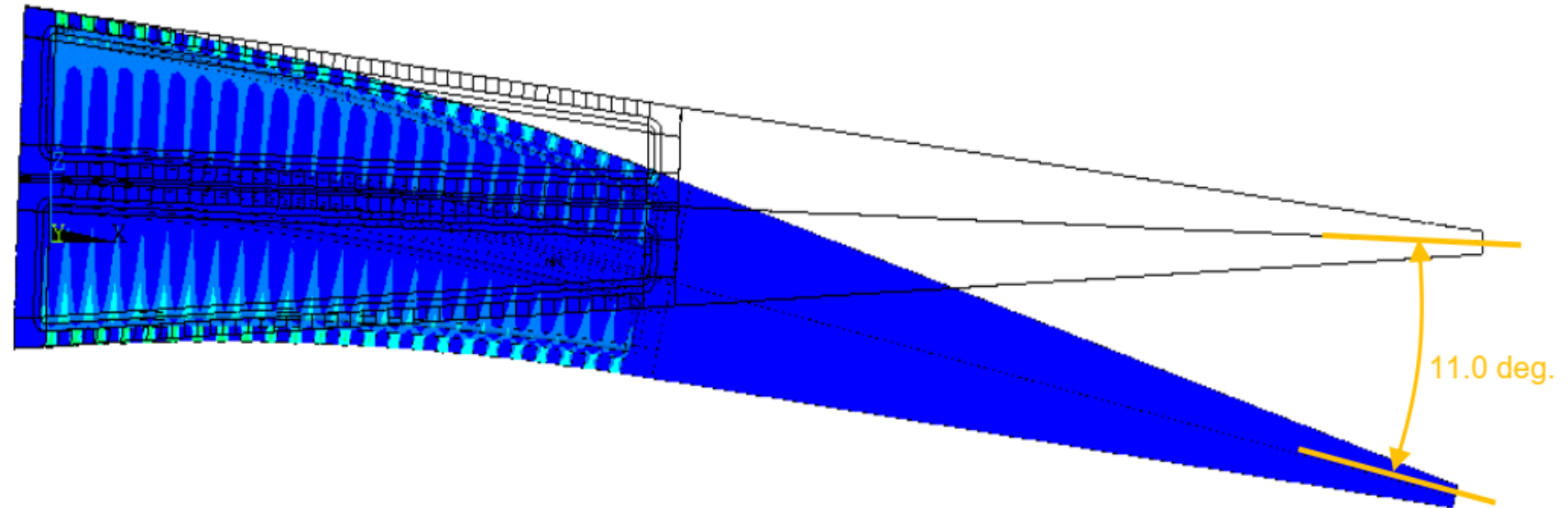


Material Selection

- Aluminium
- EPDM

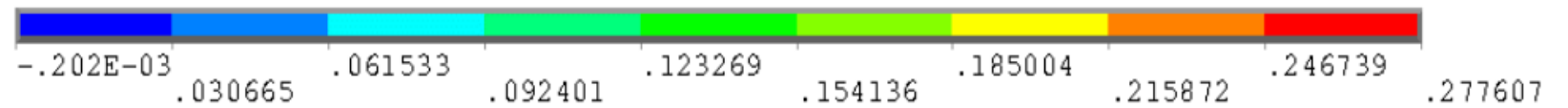
Resulting Parameters

- Fillet size 3 mm in stiffening rings
- Percentage of span comprising active units: 46.4%
- max ring stress = 110 Mpa
- Final TE angle = 16.08°
- **0,79°/bar deflection**
- Fluid volume change for 15 bar differential pressure: +4.78% and -2.98%
- Thickness of EPDM and stiffening rings: 4 mm



Upper unit: 15 bar
Lower unit: 1 bar
No aero loads

Fluid volume changes: upper = + 1.002 mL (+4.78%); lower: - 0.624 mL (-2.98%)
Max. ring stress: 133.9 MPa
Max. first principal elastomer strain: 27.8%



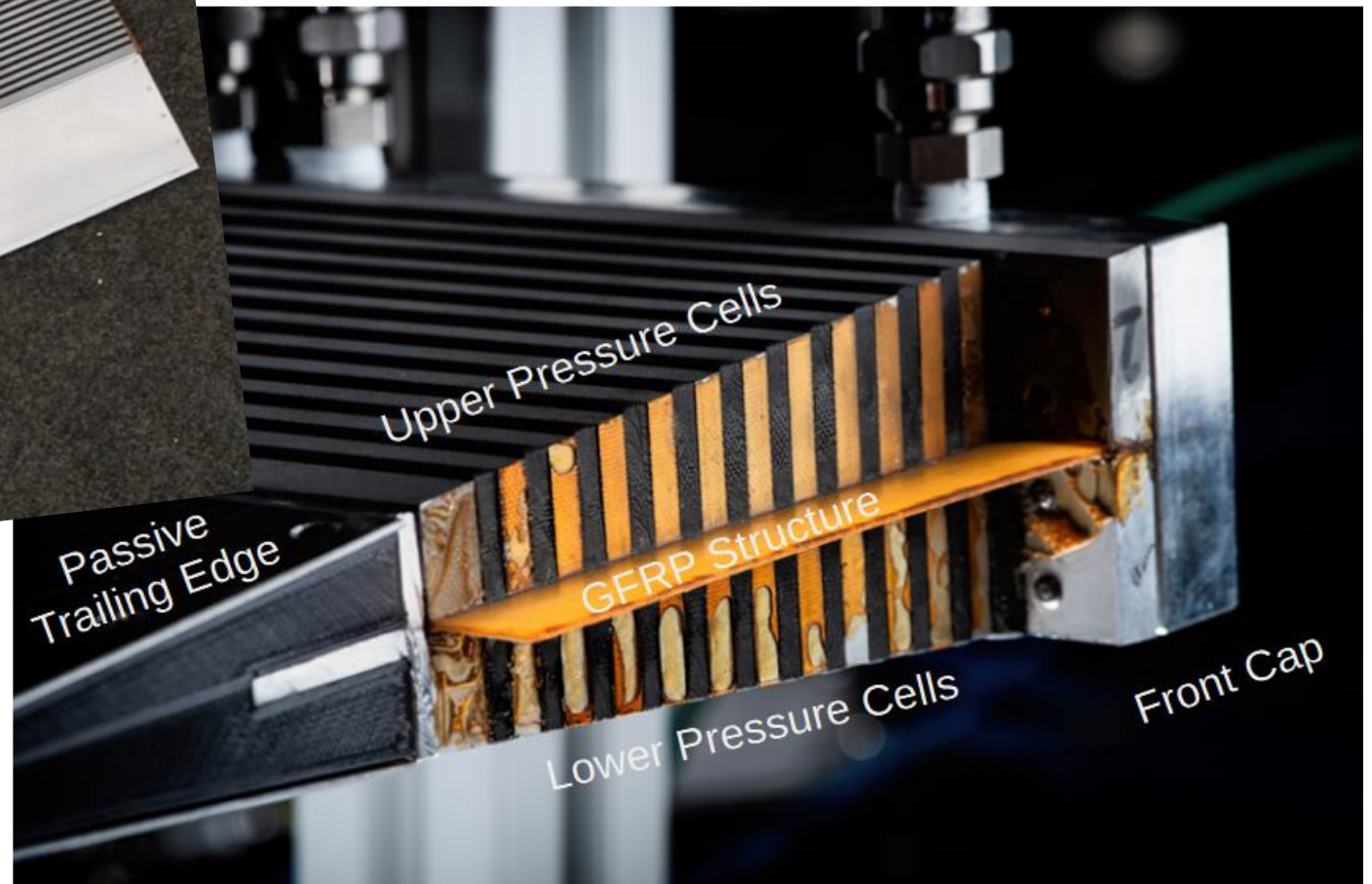
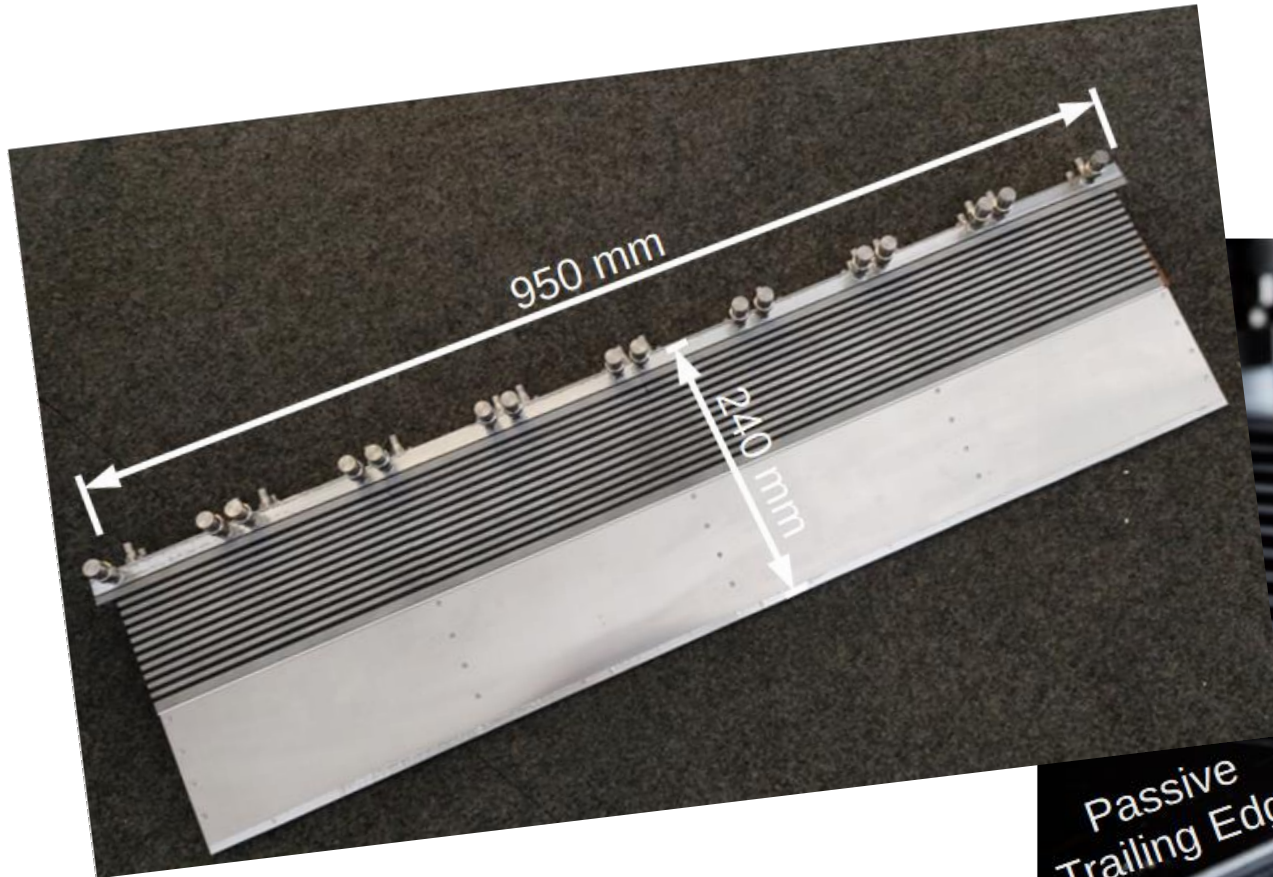
Structural Design Manufacturing



- EPDM Bladders preventing leakage



Structural Design 1m Morphing Structure



SYSTEMS DESIGN

Systems Design

Fluid Selection

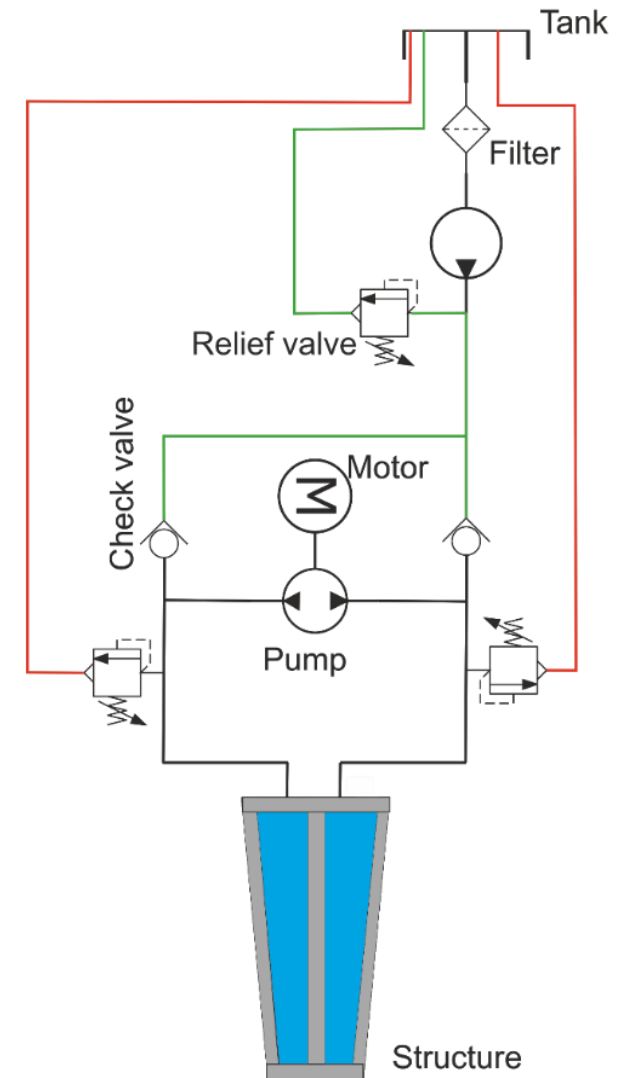


- Pressure-cells can be driven with pneumatic or hydraulic systems
 - Pneumatic: lightweight medium – compressible
 - Hydraulic: heavy medium – incompressible
- High deflection rate is driver:
 - Hydraulic system
- Chemical Compatibility with EPDM:
 - mandates water-based fluid, selected fluid is able to work from -42°C to 60°C

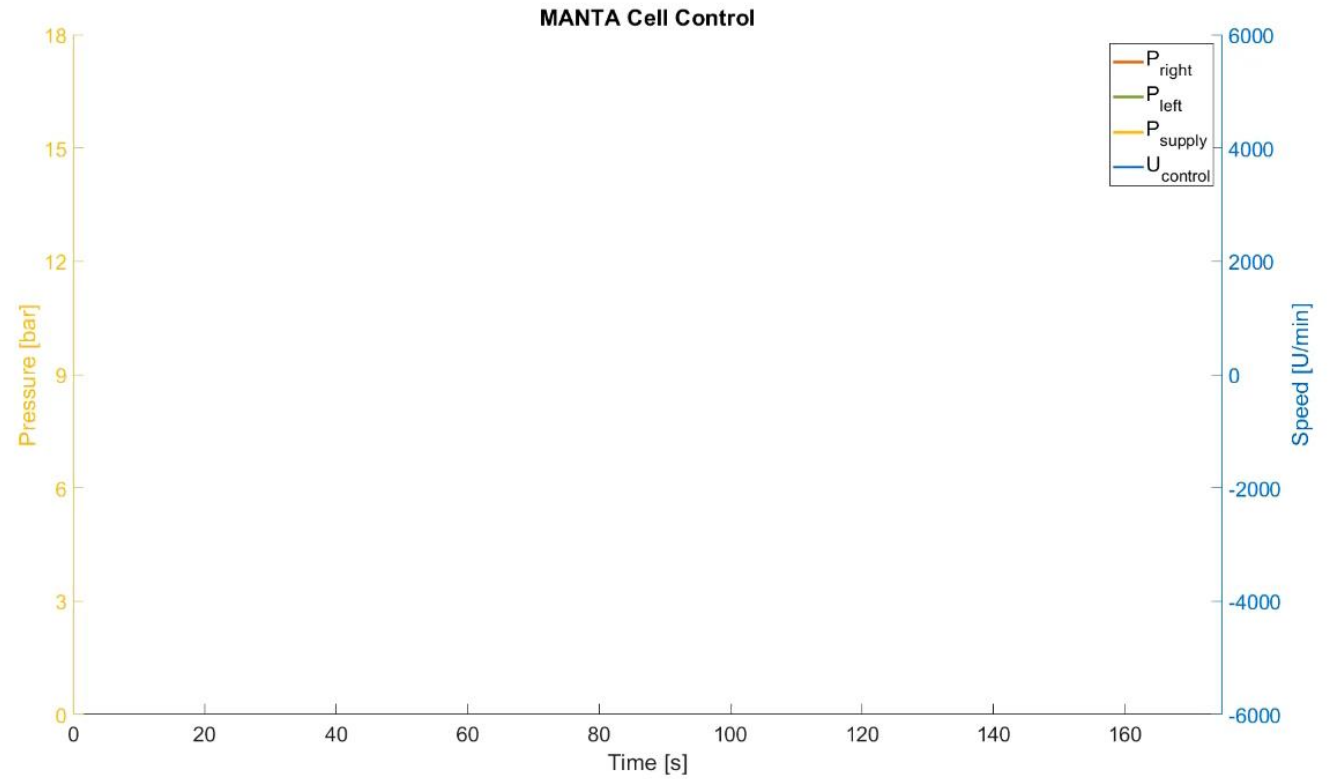
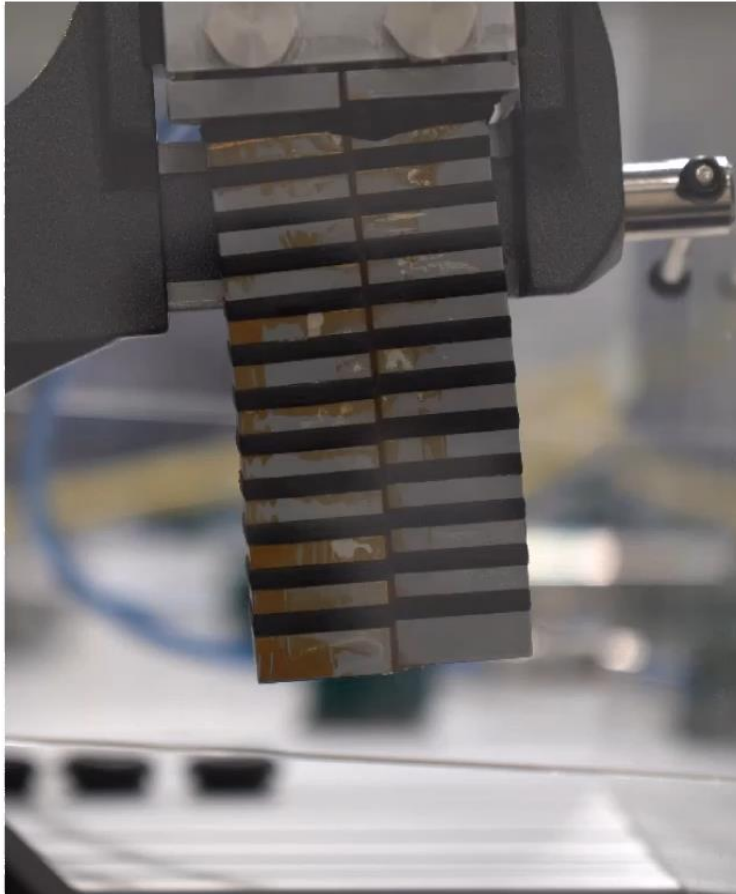
Systems Design

Core Hydraulic System

- Simplified Electro Hydrostatic Actuator
- Water-based fluid (Lubesave-Fe-46-EAL-HFC)
- Pump moving Fluid Volume between upper and lower cells (black)
- Relieve Valves release Overpressure to Tank (red)
- Second Pump insures minimum Pressure in all Cells through Check Valves (green)
- Input: Direction and Speed of 1. Pump



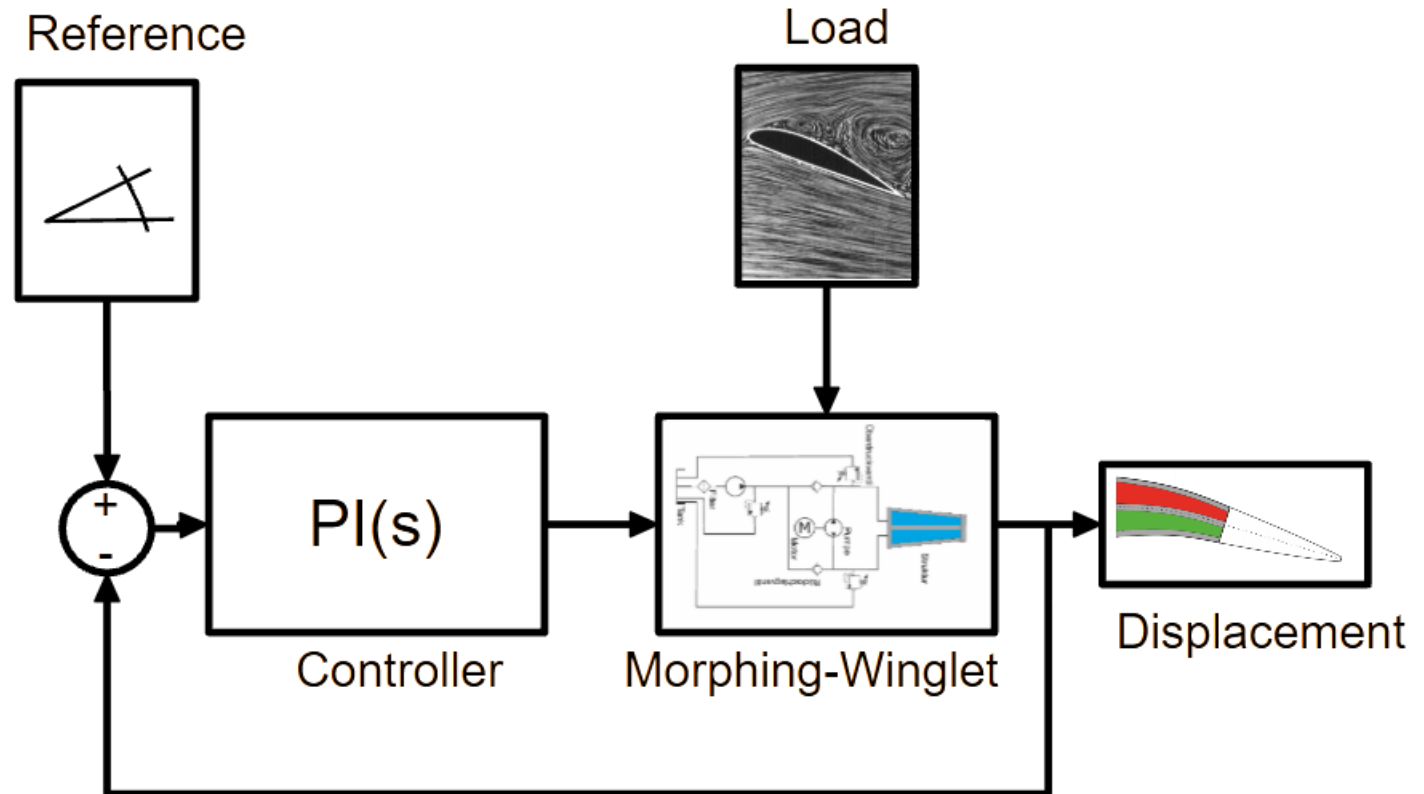
Systementwurf Test Hydrauliksystem



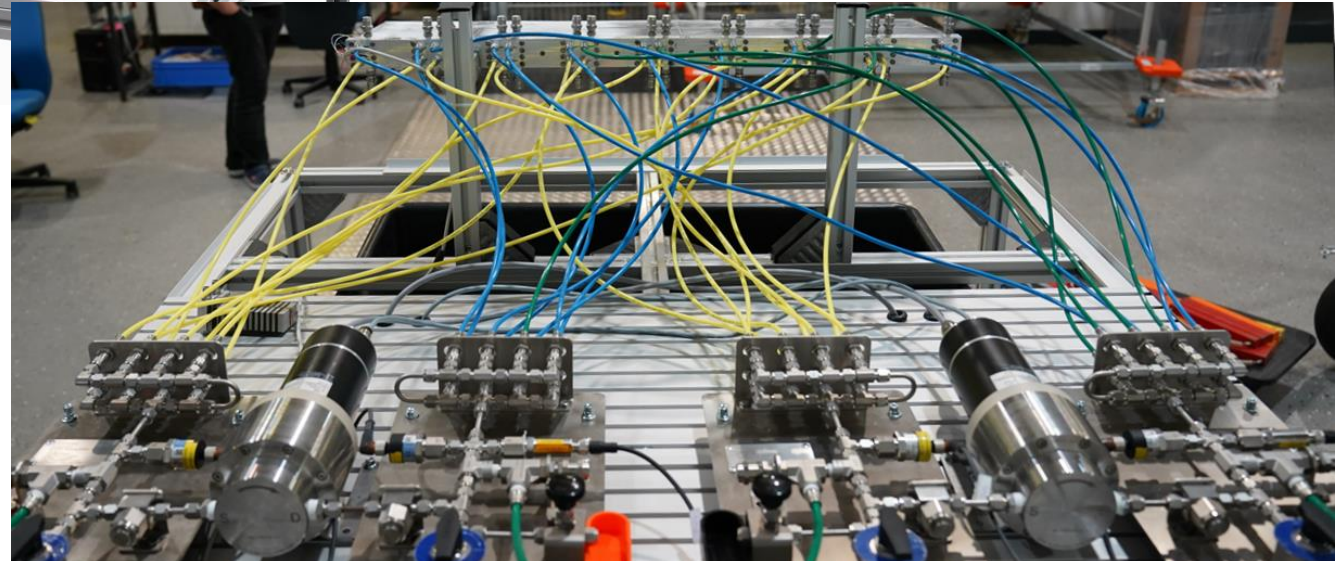
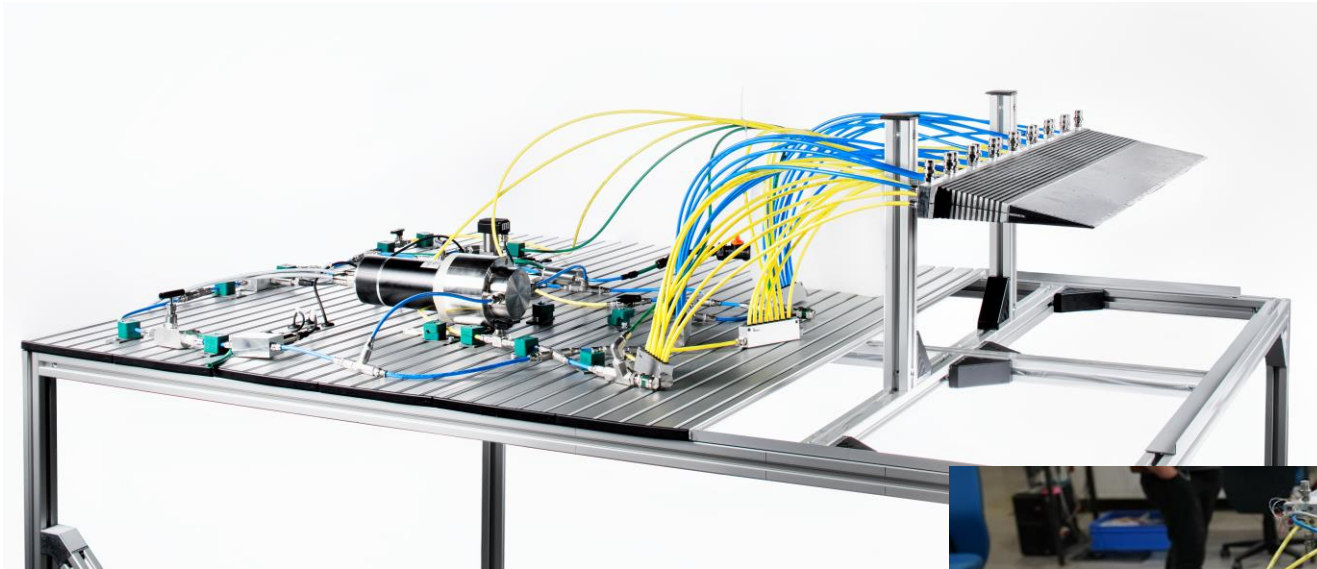
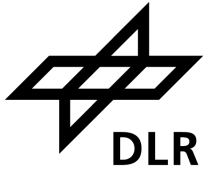
Systems Design

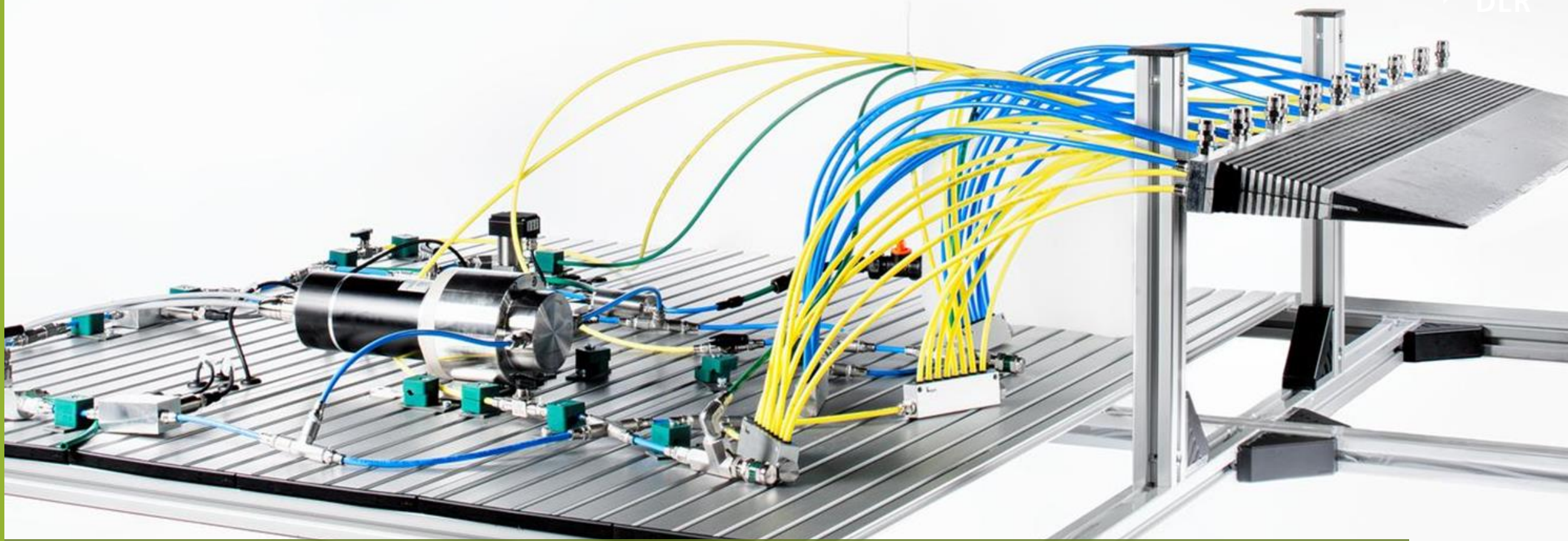
Closed Loop Control

- Closed loop control needed, due to non-linearity of EPDM
- Sensors
 - Fibre optical sensors (Primary)
 - Strain-gages (Backup)



Systems Design Redundancy





EXPERIMENTAL RESULTS

Experimental Results

Goals

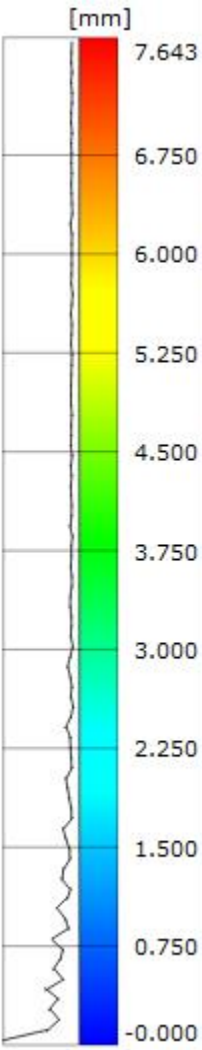
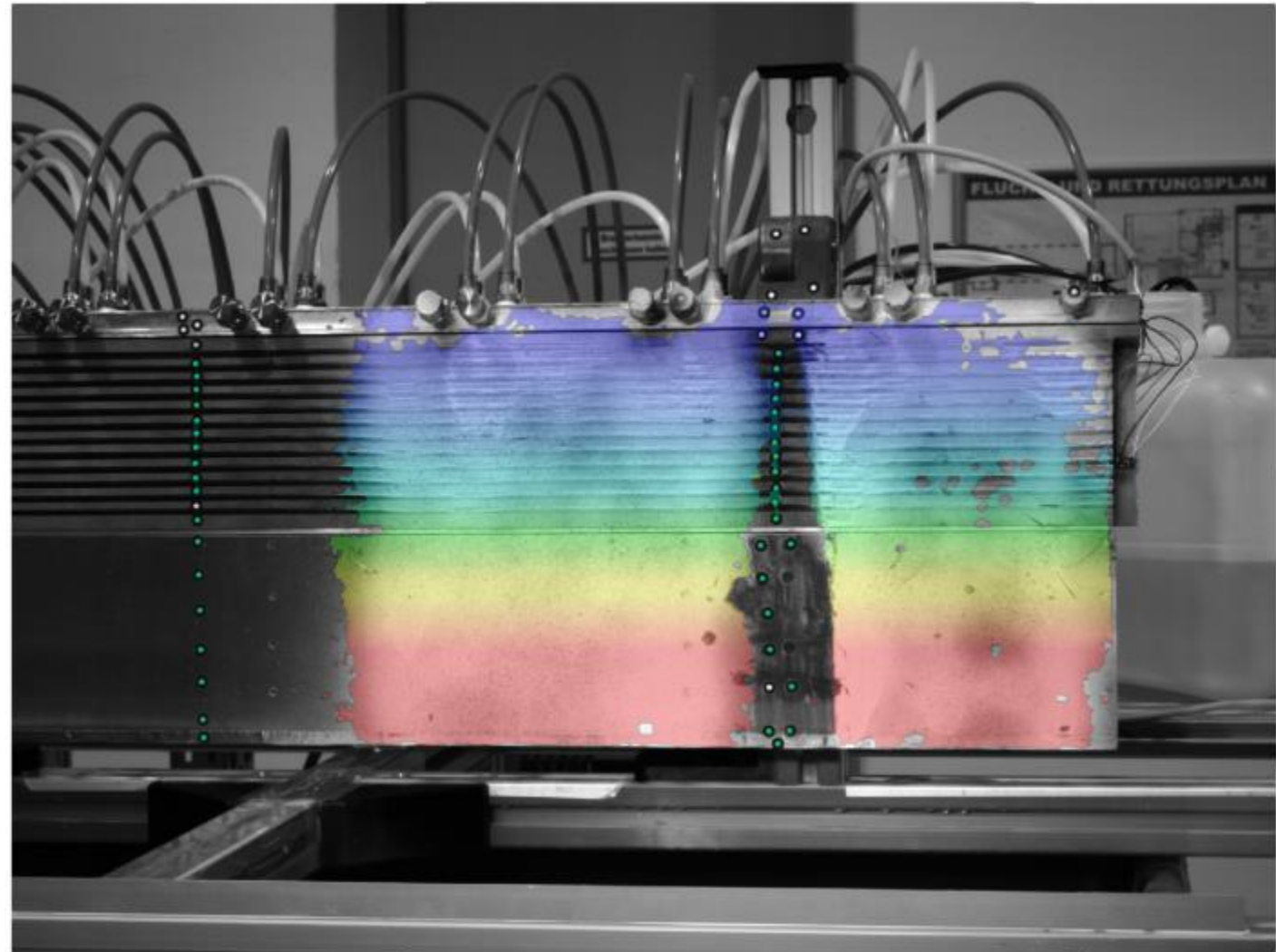


- Calibrating closed-loop control sensors (feedback sensors)
- Static performance tests:
 - Angular deflection versus pressure → validate FE
 - Spanwise uniformity of angular deflection
- Dynamic performance tests:
 - Accuracy of deflection
 - Deflection rates

Experimental Results

Calibrating Closed-Loop Control Sensors

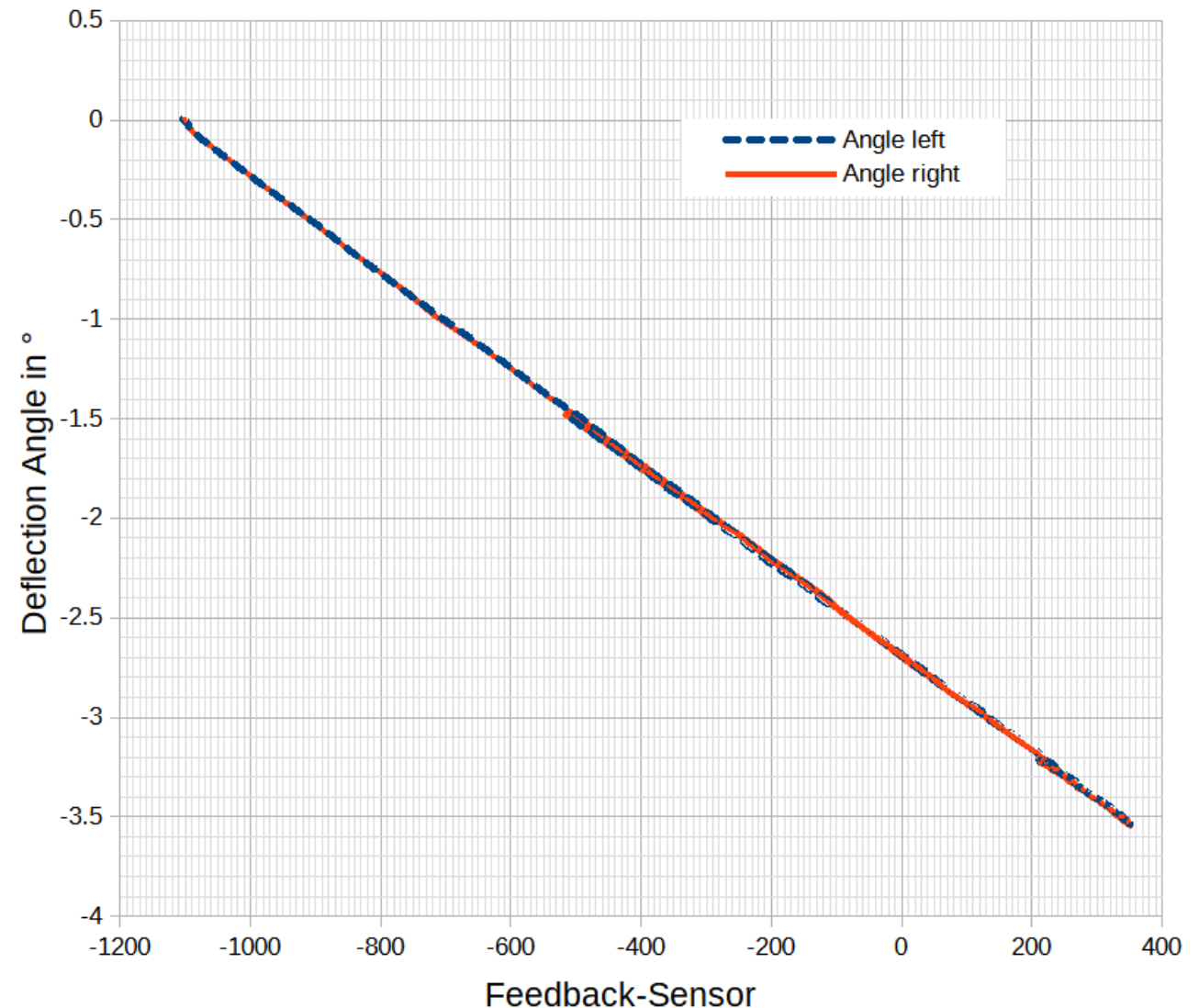
- Using external photogrammetry system



Experimental Results

Calibrating Closed-Loop Control Sensors

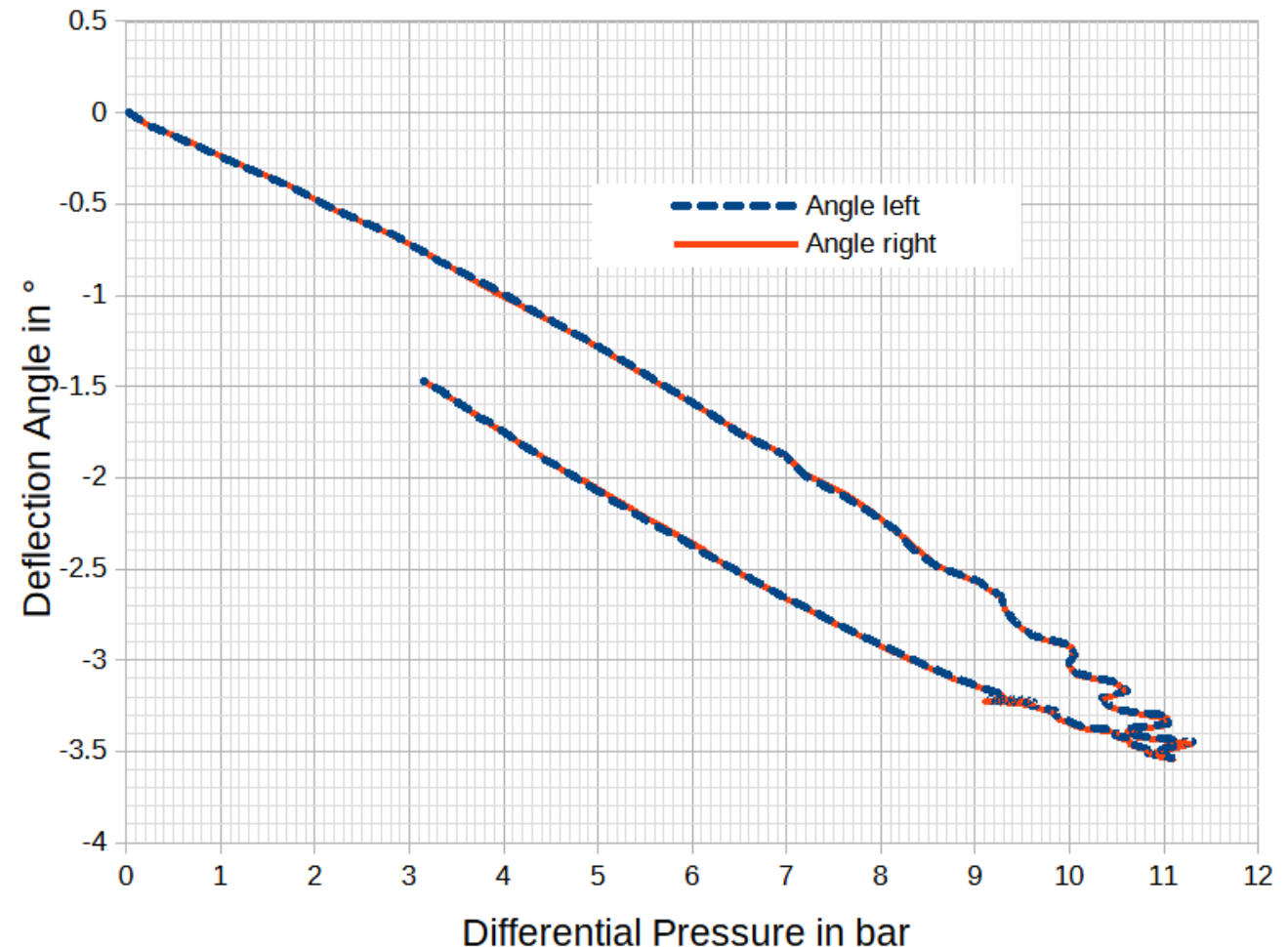
- Using external photogrammetry system
- Linear correlation between feedback-sensors and external system
- Spanwise uniformity of deflection



Experimental Results

Static Performance Tests

- Non-linear correlation between differential pressure and angular deflection
- Spanwise uniformity of deflection
- Discrepancy of angular deflection
 - FE 11° at 14 bar
0,79°/bar
 - Test 3,5° at 11 bar
0,32°/bar → **40%**

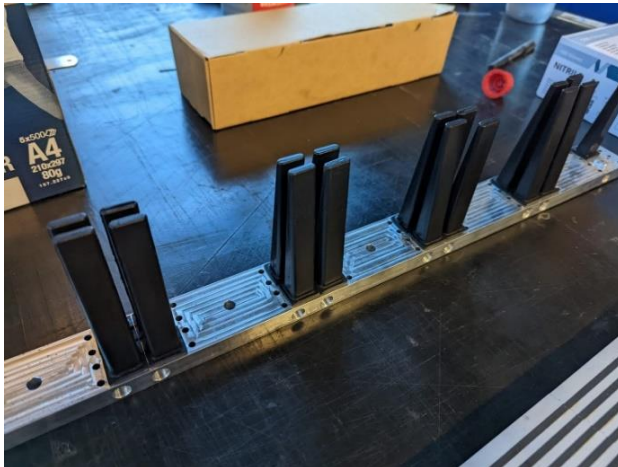


Experimental Results

Static Performance Tests

Accounting for Discrepancy

Model	FE
Measurement vs Original FE	40%
Bladders reducing Force 30%	58%
Active Passive Ratio decreased from 46.4% to 27%	95%



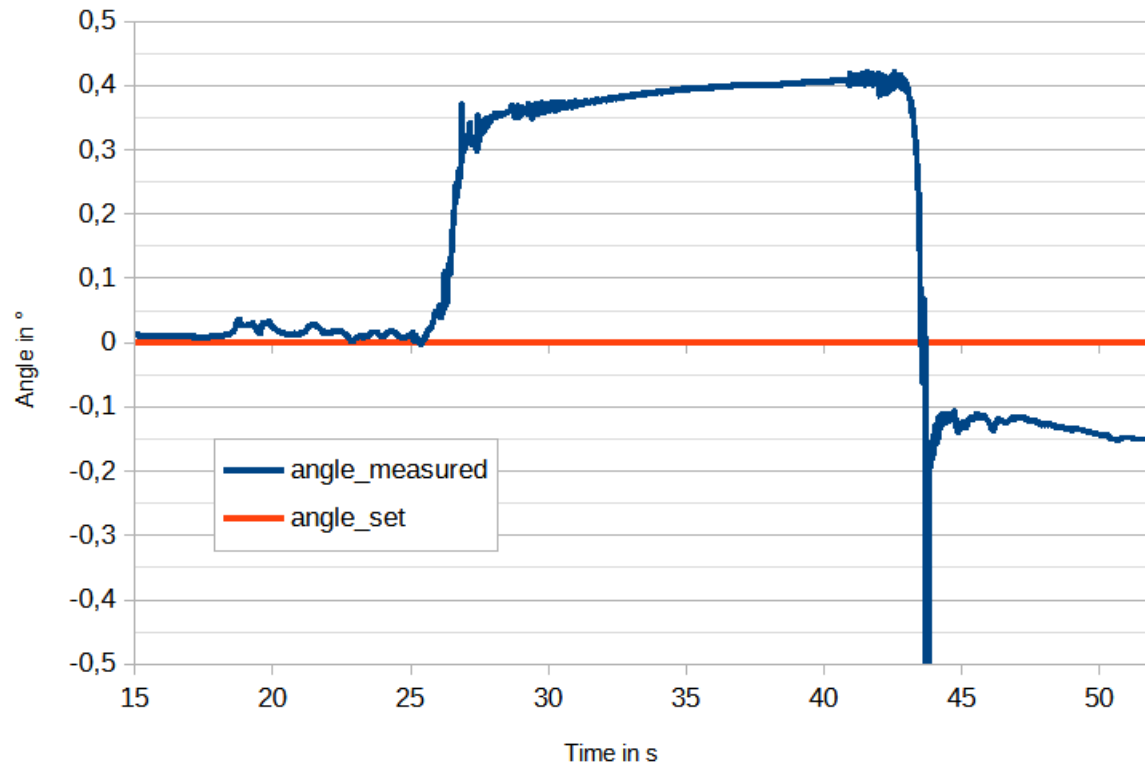
Experimental Results

Dynamic Performance Tests

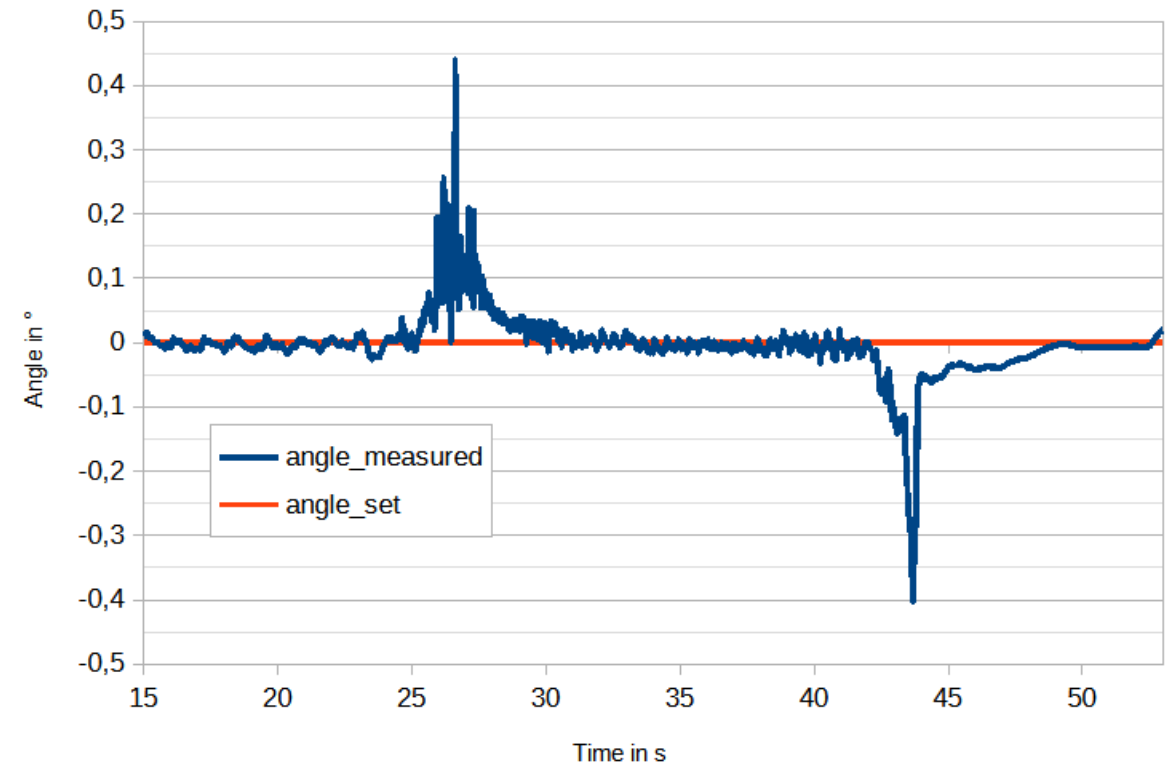


Accuracy of Angular Deflection

Open Loop



Closed Loop

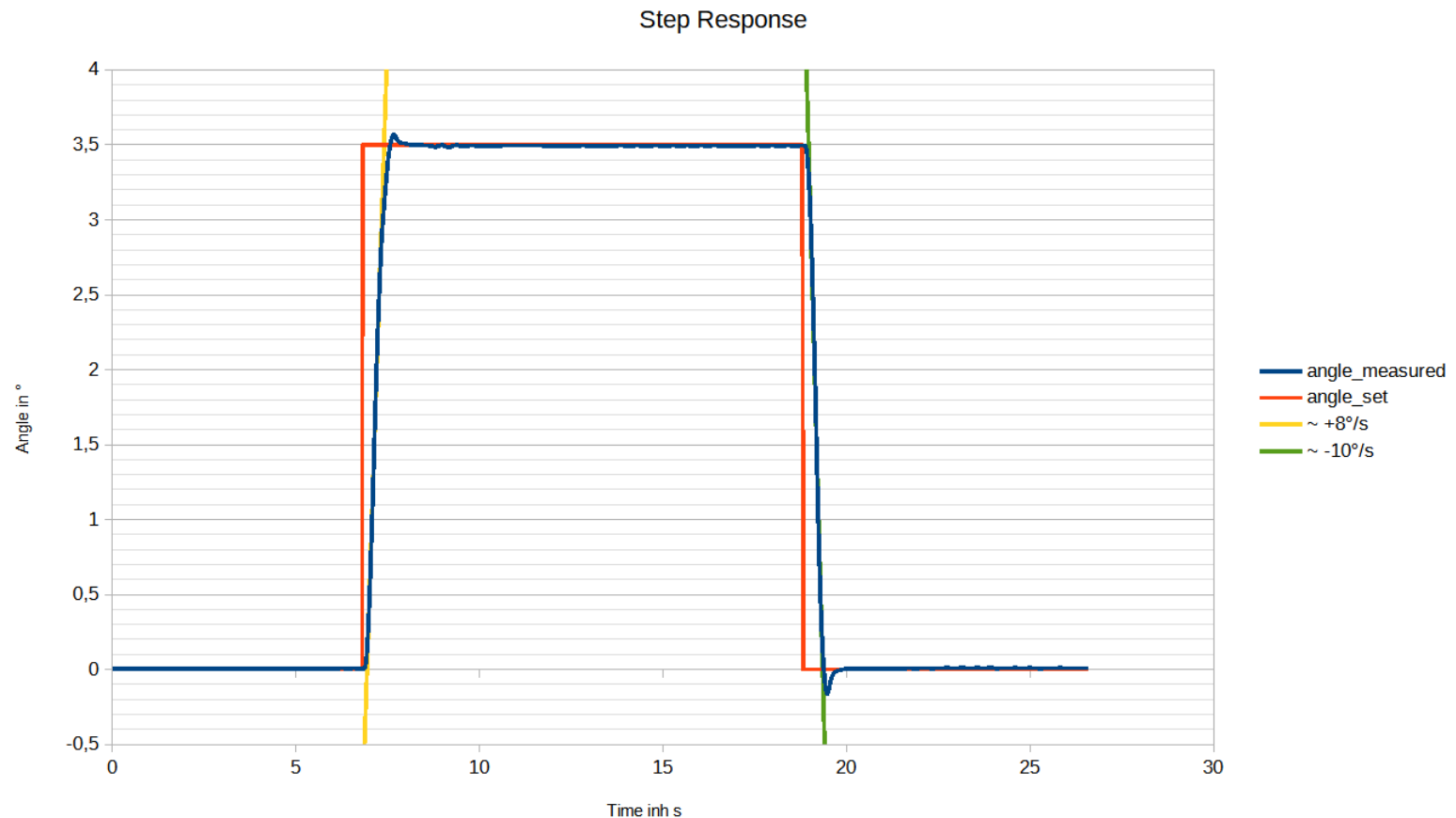


Experimental Results

Dynamic Performance Tests

Deflection Rates

- Deflection rate sufficient for high speed performance

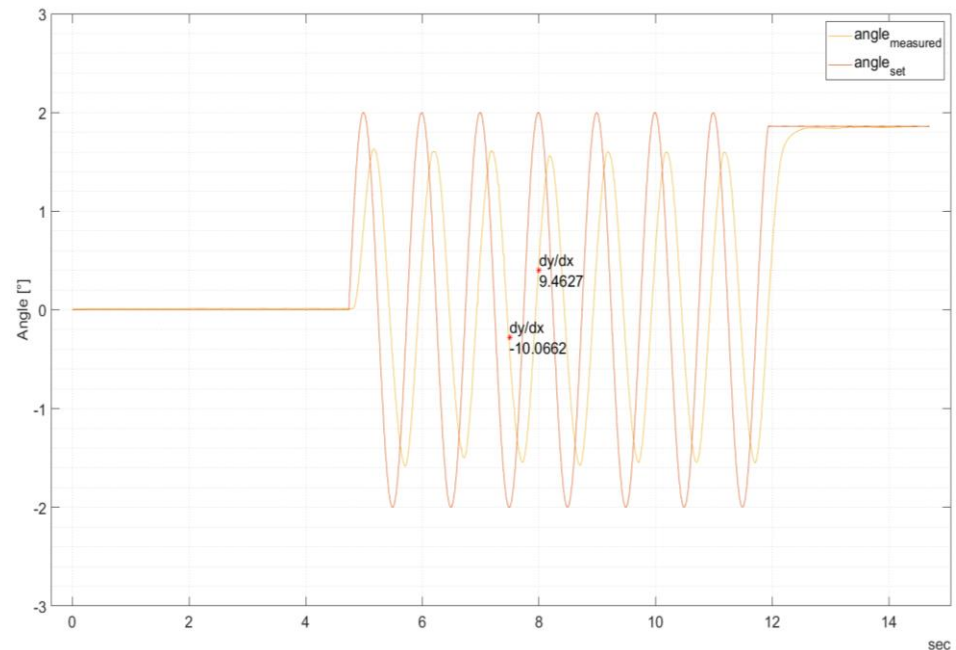


Experimental Results

Dynamic Performance Tests

Deflection

- Deflection rate sufficient for high speed performance
- Deviation of -3dB from set value

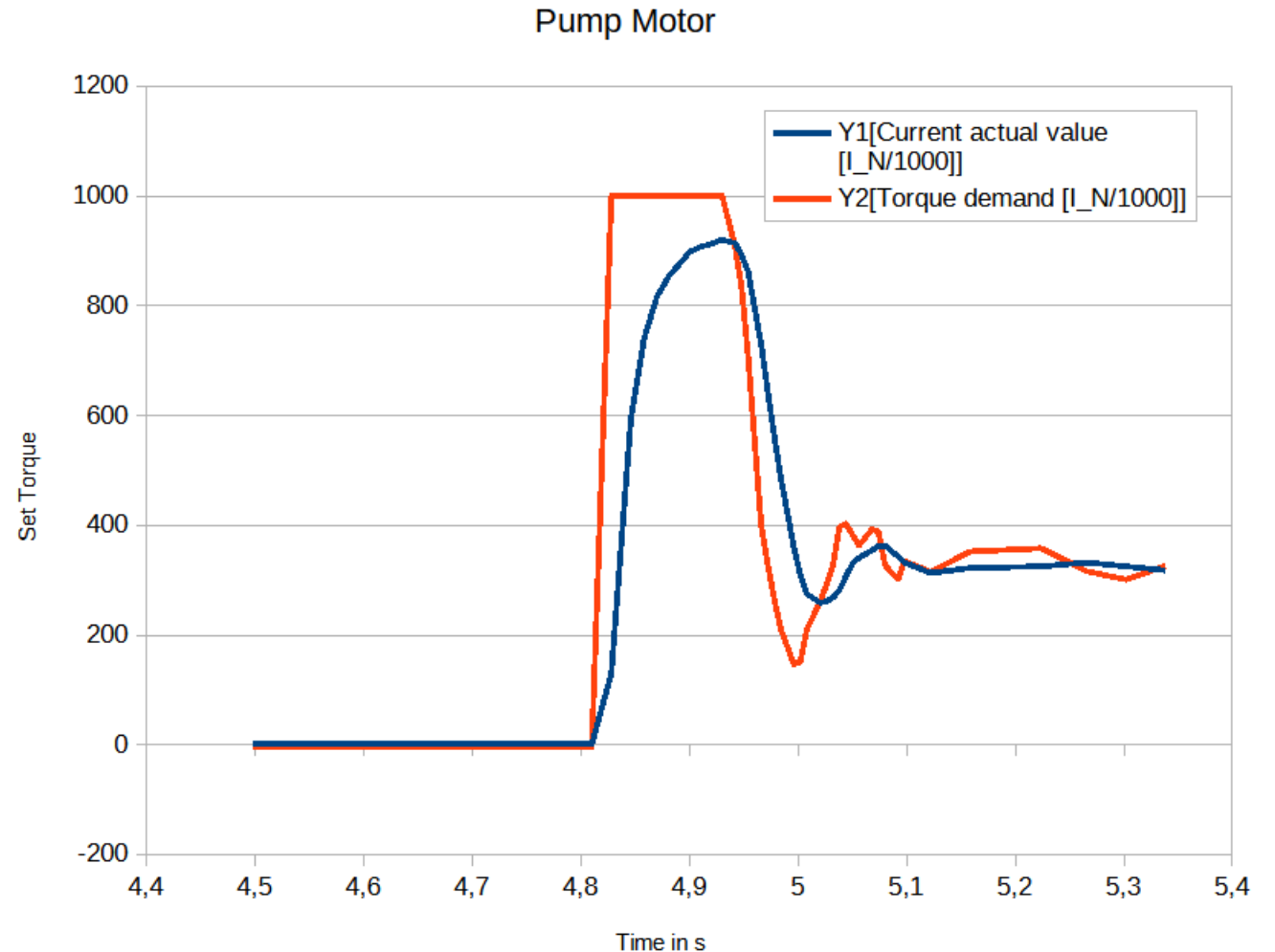


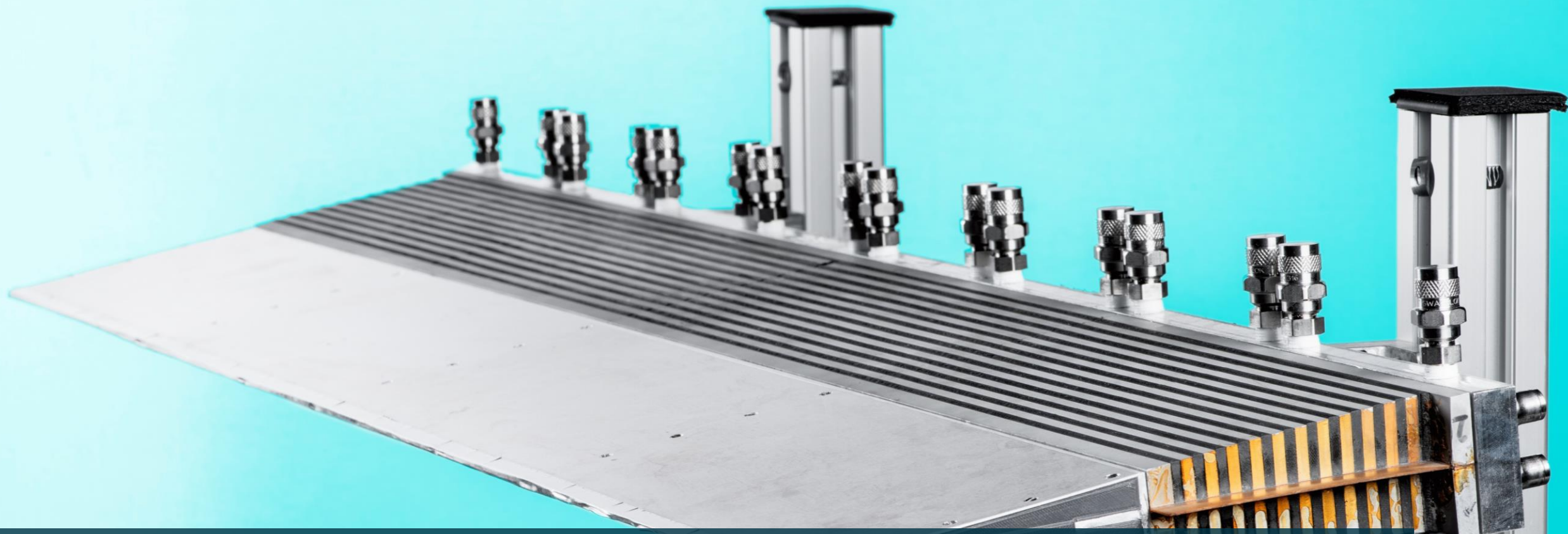
Experimental Results

Dynamic Performance Tests

Deflection

- Deflection rate sufficient for high speed performance
- Deviation of -3dB from set value





CONCLUSION & OUTLOOK

Conclusion & Outlook

Summary



- The main objective of this demonstrator to raise TRL to 2-3 is reached.
- An experimental proof of concept is done, analytical and experimental critical function and characteristics have been identified.
- 1 m spanwidth demonstrator is designed, manufactured and tested. This includes the morphing hardware, sensors, redundant hydraulic actuation system and a control-system
- The tests show that the requirements for size of deflection ($\pm 15^\circ$) and control accuracy (better than 0.1°) are met, or can safely be assumed to be met in the next design due to validation of the design model.
- The requirement of deflection rate ($> \pm 20^\circ/\text{s}$) is not yet reached, but a way forward is identified and will be tested going forward.

Conclusion & Outlook

Way Forward



- Control and Test
 - Tuning of control parameters
 - A deeper understanding of dynamic behavior of the pressure cells has to be assessed, especially separating control-parameters and material properties as cause
 - Testing the structure with simulated aero-loads
 - Testing under full range of temperatures is of interest
- Structural Design
 - Weight Optimization should be investigated in more detail
 - Account for observed failure-modes (e.g. as occasional leakage) in design
 - Aspects of Repairability should be taken into account.
- Material and Manufacturing
 - Reducing manufacturing complexity

FAMoUS Pressure Cells

Way Forward



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