

Article

# Low Order Reactor Network Based Prediction of Pollutant Emissions Applied to FLOX<sup>®</sup> Combustion

Felix Grimm <sup>1</sup>

<sup>1</sup> felix.grimm@dlr.de <sup>1</sup> German Aerospace Center (DLR), Pfaffenwaldring 38-40, 70569 Stuttgart, Germany

\* Correspondence: felix.grimm@dlr.de

Version February 22, 2022 submitted to Energies

**Abstract:** Prediction of pollutant emissions is a key aspect of modern combustor design in energy conversion systems. In the presented work, a simple and robust model based on low order reaction networks is applied to a FLOX<sup>®</sup> laboratory combustor at atmospheric conditions. The applied approach is computationally cheap and therefore highly suited for design studies.

Steady state CFD RANS simulations are carried out, serving as basis for the network generation algorithm. CFD results are validated with experimental data for flow field and combustion. Different degrees of fidelity of reactor network models are taken into consideration and findings are opposed to measurements, evaluating the quality of the low fidelity models.

Validation of CO and NO<sub>x</sub> emissions results of reactor network modeling provides accurate qualitative and quantitative reproduction of experimental findings, depending on the degree of heat loss applied on the combustion system. The introduced approach is therefore readily applicable to large scale, industrial, and gas turbine combustion.

**Keywords:** CFD Simulation; RANS; Chemkin Reactor Network Models; NO<sub>x</sub> and CO Emissions; FLOX<sup>®</sup> Combustion

## 1. Introduction

Combustor development for energy related and aero-engine applications is subject to strict regulations regarding pollutant emissions [1]. This is addressed in manifold ways, one being the shift from conventional to renewable and alternative fuels, which requires a re-design of most existing systems or the creation of novel concepts for energy conversion.

In this situation, system pollutant emissions analyses have to be included in the design process from an early stage. Usually, a set of different emissions quantities are considered, namely soot, UHCs (Unburned HydroCarbons), NO<sub>x</sub> and CO. Where combustion systems with lean burn operation regime are developed, pollutant emissions are dominated by NO<sub>x</sub> and CO, which are treated in the presented framework.

NO<sub>x</sub> pollutants are known as being highly toxic and having large impact on atmospheric ozone and contributions to acid rain [2]. Energy conversion with fuels containing no nitrogen usually inhibits three pathways of NO<sub>x</sub> formation. The Zeldovich mechanism [3], also known as thermal mechanism that prevails in high-temperature regimes. The Fenimore mechanism [4], of particular importance in rich combustion. And the N<sub>2</sub>O intermediate formation [5], predominant in very lean and low temperature combustion.

CO levels are usually high when there is incomplete carbon monoxide burnout, usually dominant at part load conditions [6,7]. Formation of CO has been experimentally investigated [8–10] and also numerically [6,11–15], only to name a few.

The design of energy conversion systems, mainly referring to the combustor, is usually carried out with numerical methods. Those are early stage 0-D tools up to flow field and combustion resolving

36 CFD (Computational Fluid Dynamics) methods for later design stages.  
 37 Prediction of emissions in CFD is a prevailing research field. It can be either covered with  
 38 computationally expensive scale resolving simulation techniques, like LES (Large Eddy Simulation)  
 39 oder DES (Detached Eddy Simulation) or with rather simple but efficient RANS (Reynolds Averaged  
 40 Navier Stokes) based methods. As a general rule of thumb, the more sophisticated the method, the  
 41 less applicable it is to combustor system development, where a lot of iterative loops and parametric  
 42 studies have to be carried out. Therefore, RANS based methods are still state of the art in the design  
 43 process of energy conversion systems.

**Table 1.** Literature overview on ERN based pollutant emission prediction, selected studies. C>E: Extraction of networks from CFD. C+E: Using CFD as guidance or estimation for ERN creation. E: Reactor networks without CFD.

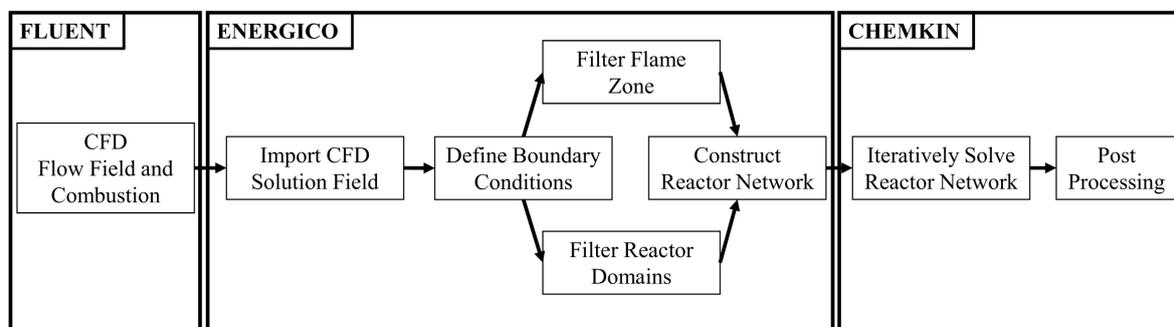
Source	Year	C > E	C + E	E	Application
Bhargava et al. [16]	2000			x	Single nozzle experiments and numerical modeling, study of pressure effects
Falcitelli et al. [17]	2002	x			NOx prediction in industrial energy combustion systems
Falcitelli et al. [18]	2002	x			NOx prediction in industrial furnaces, NOx reduction techniques
Falcitelli et al. [19]	2002	x			Methodical assessment of ERN pollutants prediction from CFD
Mohamed et al. [20]	2004			x	NOx, UHC and CO for gas turbine combustion (model only)
Novosselov et al. [21]	2006	x			NOx and CO in a swirl stabilized burner
Russo et al. [22]	2007		x		NOx and CO in a recuperated micro gas turbine
Benedetto et al. [23]	2008		x		Industrial furnace facilities
Fichet et al. [24]	2010	x			NOx emissions in a staged gas turbine combustor
Lee et al. [25]	2011	x			NOx emissions in a simplified combustor, GE7FA gas turbine
Lyra and Cant [26]	2013	x			NOx emissions in a high-pressure nozzle test case
De Toni et al. [27]	2013		x		NOx emissions in a BERL 300kW furnace combustor
Colorado et al. [28]	2014		x		NOx emissions in a C60 gas turbine combustor
Nguyen [29]	2017		x		NOx emissions for a generic gas turbine burner using Chemkin
Nguyen et al. [30]	2017	x			NOx prediction with Chemkin for a gas turbine combustor
Innocenti et al. [31]	2018	x			NOx and CO in swirl-stabilized aero-engine combustor
Kaluri et al. [32]	2018			x	Real-time reactor network for LBO predictions
Nguyen [2]	2019	x			Emissions in a swirl-stabilized combustor using Chemkin
Gupta et al. [33]	2019			x	Real-time reactor network for LBO predictions
Perpignan et al. [34]	2019	x			NOx and CO emissions at flameless oxidation combustion
Zhang et al. [35]	2020	x			NOx and CO emissions in a swirl stabilized aero-engine combustor sector

44 When robust models are used for CFD, it stands to reason to also choose robust, simple, and  
 45 efficient approaches for pollutant emissions prediction. Such are ERN (Emissions Reactor Network)  
 46 models. Especially in the later 2000s, a lot of research effort was put into ERN modeling. A  
 47 comprehensive overview is given in Table 1. Surprisingly, the degree of model fidelity does not

necessarily increase with time. Many early studies use automated algorithms for network extraction from CFD data [17–19,21], whereas few rely on using CFD results as guidance for manual network creation only [22,23,27]. Some works even disregard the possibility of information gathering from CFD [16,20,32,33], which is a potentially large inaccuracy.

Direct precursors for this study are later works of Nguyen et al. [29,30] and Perpignan et al. [34]. Nguyen et al. carried out ERN studies based on Chemkin, with different degrees of model fidelity and Perpignan et al. studied NO<sub>x</sub> and CO emissions in flameless oxidation, which is also studied in the presented work.

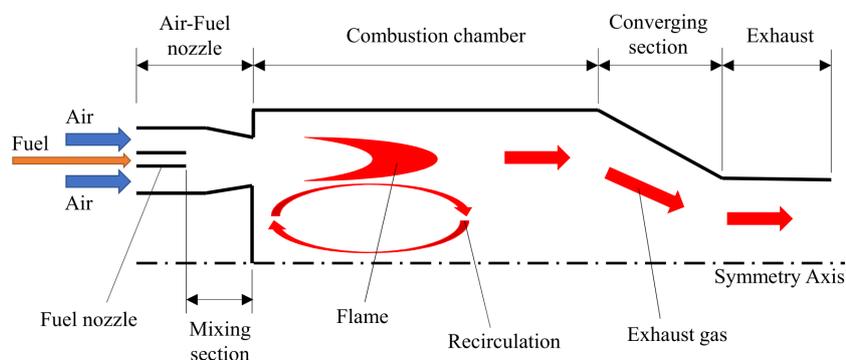
In this paper, several novelties and distinctions can be stated compared to literature works. As a major point, we study and discuss ERN models with different degrees of fidelity: Several algorithmically generated network models are treated as well as a baseline case that uses CFD data as guidance only. Thereby, robustness of the approach towards the degree of fidelity is tested. Furthermore, a novel commercial code for the extraction of ERNs from CFD is tested, the so called *Energico* module as part of the Ansys workpackage. It is combined with Chemkin for ERN solving. The modeling procedure is sketched in Figure 1.



**Figure 1.** Tool chain of the ERN setup and solution approach.

CFD flow field and combustion data is used to filter the ERNs, based on the local temperature and flow composition. The filtered field is then translated into network models with different degrees of detail (number of reactor modules). The system is iteratively solved in the Ansys Chemkin program, which can also be used for data postprocessing.

Another distinct novelty of the presented work is the application to FLOX<sup>®</sup> combustion. FLOX<sup>®</sup> burners operate in a MILD (Moderate or Intense Low oxygen Dilution) regime. They were at first applied in atmospheric, low calorific furnace applications [36,37]. The FLOX<sup>®</sup> principle, as adapted for gas turbine combustion, is illustrated in Fig. 2.



**Figure 2.** Schematic of the FLOX<sup>®</sup> burner principle.

Fuel is injected through several nozzles arranged coaxially in circumference with the aim to premix with air before issuing into the combustion chamber. A characteristic inner recirculation zone

73 is formed, so that combustion products are conveyed back into the reaction zone, igniting the fresh  
 74 gases. As a result, a homogeneous temperature distribution, a wide and stable operating range and  
 75 low emissions [38,39] are the main features of these systems. Furthermore, the risk of flashback is  
 76 reduced due to the presence of high velocity jets with high momentum, which promotes this design to  
 77 multi-fuel applications including hydrogen combustion [40,41].

78 The paper is structured as follows: The FLOX<sup>®</sup> burner test case is introduced first. This is followed  
 79 by CFD setup and results discussion. Results are validated with experimental data by means of PIV  
 80 velocity fields and flame surface density from OH\* chemiluminescence. Subsequently, setup and  
 81 modeling procedure of ERN computations are introduced, explained and results are discussed on basis  
 82 of a parametric study including model fidelity and different numbers of reactors. Resulting modeled  
 83 pollutant emissions are opposed to experimental data of the lab scale burner.

## 84 2. Laboratory Scale Test Case Combustor

85 The herein investigated laboratory scale burner setup is shown in Fig. 3. The 3kW burner was  
 86 designed for operation in an MTT (Micro Turbine Technology b.v.) gas turbine and has been scaled to  
 87 atmospheric operation conditions. This allows for a detailed examination of the combustion system by  
 88 means of exhaust gas measurements and optical measurement techniques. Therefore, studies with  
 89 OH\* chemiluminescence and PIV have been simultaneously carried out [42]. They provide insight  
 90 into flow field and reaction zone location and are used for CFD model validation. CFD serves as basis  
 91 for ERN modeling.

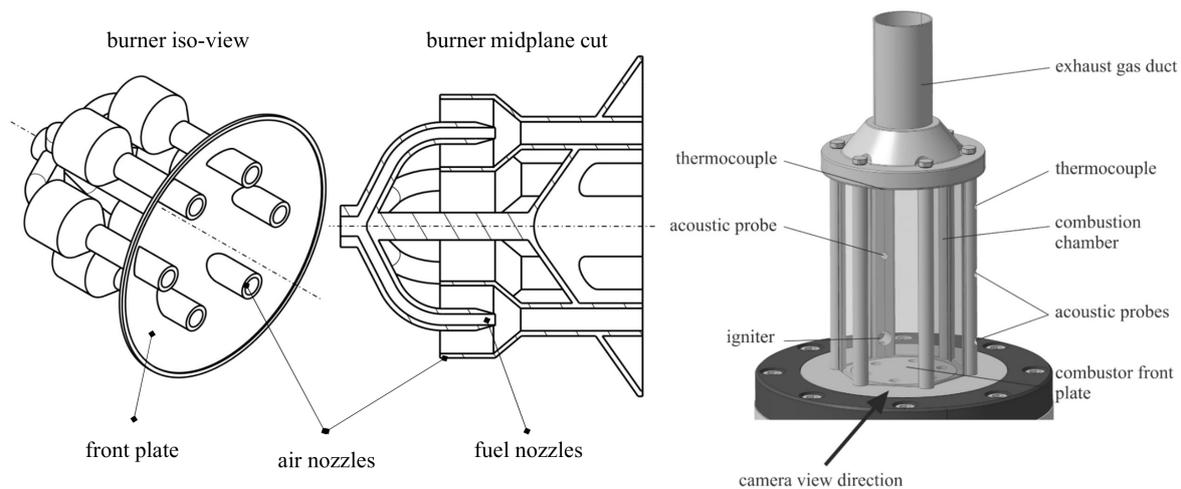
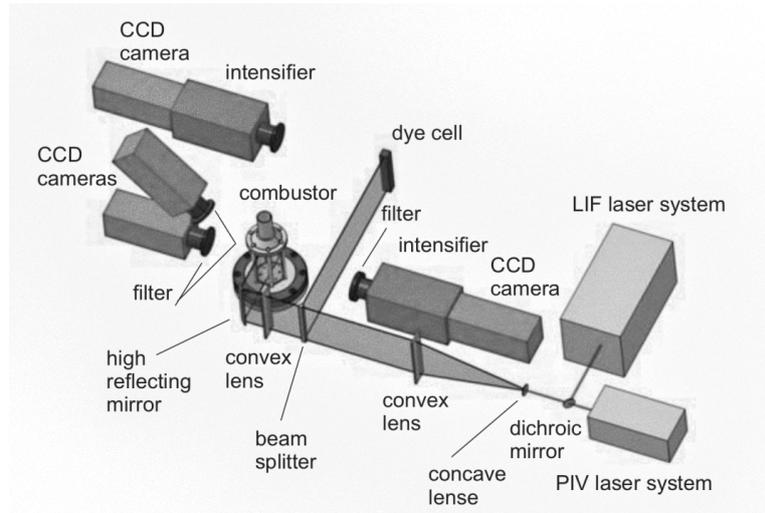


Figure 3. Setup of the laboratory scale burner system [42–44].

92 Details on the experimental setup can be found in the literature [42–44]. The burner system is  
 93 operated with methane at an optimum air fuel number of  $\lambda \approx 2.2$ . Air is fed through the system by an  
 94 upstream plenum, where the air is preheated, in order to mimic recuperation as in the MTT gas turbine.  
 95 Air is led into the combustion chamber via six circumferentially aligned nozzles, whereas the fuel is  
 96 induced in a co-flow alignment into each nozzle, as can be seen in Fig. 3. The combustion chamber  
 97 consists of quartz glass windows for optical access. Optical measurements were conducted in a way  
 98 that two air-fuel nozzles are aligned in the measurement sheet, as indicated in Fig. 3 by the camera  
 99 view direction. A more detailed sketch showing the alignment of measurement equipment is given  
 100 in Fig. 4. PIV (Particle Image Velocimetry) measurements of velocity components in the combustion  
 101 chamber are carried out with titanium dioxide particles.

102 Due to the air-fuel co-flow alignment and an offset of fuel nozzles, there is a short mixing section,  
 103 before the mixture issues as partially premixed fluid (technically premixed) into the combustion  
 104 chamber. There, discrete flames are located over each nozzle. Due to the strong axial momentum, a



**Figure 4.** Alignment of experimental tools for PIV, OH-PLIF and OH\* chemiluminescence measurements. Courtesy of [45]

105 large inner recirculation zone of hot gases develops, which increases combustion stability and favours  
 106 low pollutant emissions. This is due to long fluid residence times and homogeneous temperature  
 107 distribution.

108 NO<sub>x</sub> emissions are sampled at the combustor exhaust gas duct with an UV photometer, whereas CO  
 109 emissions are taken at the same position with IR photometer. Measurement accuracy for NO<sub>x</sub> and CO  
 110 is in the order of magnitude of 0.1ppm (parts per million) [42].

### 111 3. Computational Combustion Dynamics

112 In the presented work, chemical reactor network models are built from underlying CFD solutions.  
 113 Therefore, numerical setup and validation with experimental data is presented briefly, in order to  
 114 demonstrate feasibility of usage of the CFD numerical data as ERN construction basis.

#### 115 3.1. Numerical Setup

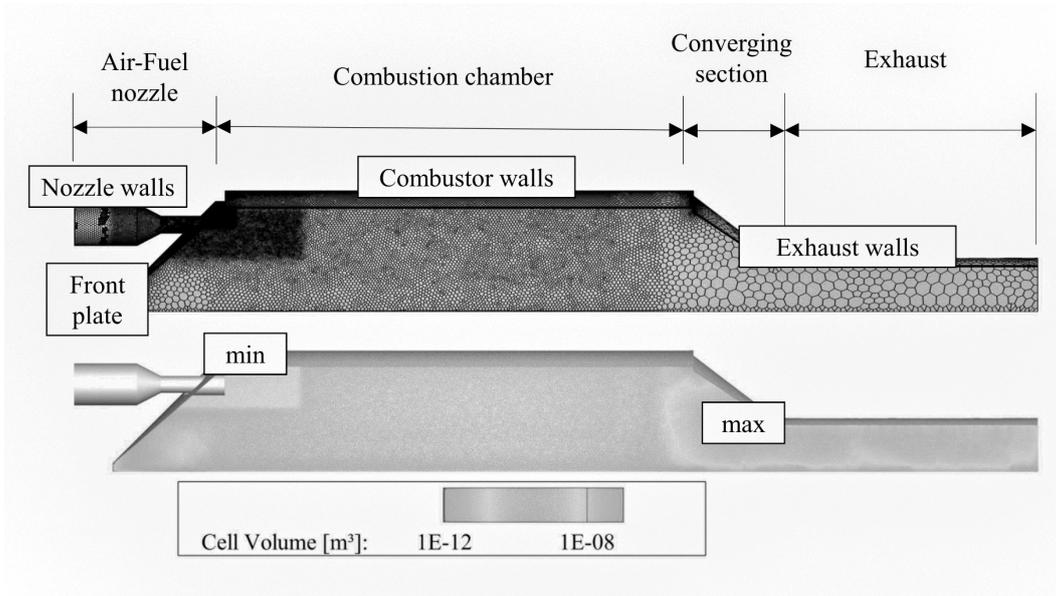
116 The numerical setup is shown in Fig. 5. A 60 degree segment of the burner is explicitly simulated  
 117 in order to save computational time and ease the reactor network creation.

118 The grid consists of 796k polyhedral elements with 238k nodes. This means an improvement  
 119 compared to previous studies [46], where pure tetrahedral grids were used in terms of convergence  
 120 and computational speed. However, a grid study in the previous work [46] gives a solid orientation on  
 121 required local grid resolution.

122 All simulations are carried out in Ansys Fluent. The mesh is refined in the reaction, recirculation and  
 123 especially the mixing zones close to the burner entry. Steady state RANS simulations are performed  
 124 with a SIMPLE solution strategy. Since the  $k\omega$ -SST turbulence model is used, near wall regions are  
 125 refined with prism layers that suffice  $y^+ \approx 1$  for the first near-wall cell layer. Ten prism layers are  
 126 inserted with a growing rate of 10 % towards the flow domain. The  $k\omega$ -SST turbulence model is chosen,  
 127 since the flow field is highly influenced by wall effects but air-fuel jets issuing into the combustion  
 128 chamber should be modeled with free stream effects. Its transport equations have the form

$$\frac{\partial}{\partial t}(\rho k) + \frac{\partial}{\partial x_i}(\rho k u_i) = \frac{\partial}{\partial x_j} \left( \Gamma_k \frac{\partial k}{\partial x_j} \right) + G_k - Y_k, \quad (1)$$

$$\frac{\partial}{\partial t}(\rho \omega) + \frac{\partial}{\partial x_i}(\rho \omega u_i) = \frac{\partial}{\partial x_j} \left( \Gamma_\omega \frac{\partial \omega}{\partial x_j} \right) + G_\omega - Y_\omega, \quad (2)$$



**Figure 5.** Grid and computational domain of the numerical CFD simulations.

129  $\rho$  denotes density,  $k$  is the turbulent kinetic energy and  $\omega$  the turbulence frequency.  $G$  are  
 130 production terms,  $\Gamma$  are effective diffusivities and  $Y$  are dissipation terms due to turbulence. The  
 131 production terms are evaluated with  $G_k = \mu_t S^2$ , with the turbulent viscosity  $\mu_t$  and the modulus of the  
 132 mean rate of strain tensor  $S$ , and  $G_\omega = (\alpha\alpha^*)/\nu_t G_k$ , with the turbulent kinetic viscosity  $\nu_t$ . Dissipative  
 133 terms follow  $Y_k = \rho\beta^*k\omega$  and  $Y_\omega = \rho\beta\omega^2$ .  $\alpha_i$  and  $\beta_i$  are evaluated with blending functions

$$\alpha_m = F_1\alpha_{m,1} + (1 - F_1)\alpha_{m,2}, \quad (3)$$

$$\beta_n = F_1\beta_{n,1} + (1 - F_1)\beta_{n,2}, \quad (4)$$

134 where formulations for  $\alpha_m$  and  $\beta_m$  can be found in the literature [47]. As a particularity, the eddy  
 135 viscosity of the  $k\omega$ -SST turbulence model is treated with

$$\mu_t = \frac{\rho k}{\omega} \left( \max\left[\frac{1}{\alpha^*}, \frac{SF}{\alpha_1\omega}\right] \right)^{-1}, \quad (5)$$

136 the explicit formulation of which can also be taken from the literature [47]. All model constants  
 137 are taken as standard model values, particular values for the  $k\omega$ -SST model are

$$\sigma_{k,1} = 1.176; \quad \sigma_{k,2} = 1.0; \quad \sigma_{\omega,1} = 2.0; \quad \sigma_{\omega,2} = 1.168; \quad (6)$$

$$\alpha_1 = 0.31; \quad \beta_{i,1} = 0.075; \quad \beta_{i,2} = 0.0828. \quad (7)$$

138 Combustion is depicted with the Eddy Dissipation Concept (EDC), whereas a detailed reaction  
 139 scheme from Li et al. [48] is employed for the modeling of methane combustion kinetics. Since  
 140 FLOX<sup>®</sup> combustion shows a large range of combustion regimes due to partial or technical premixing of  
 141 fuel and air, it is essential to provide a combustion scheme that features combustion based on local  
 142 kinetics rates. The EDC therefore allows for this. It is driven by the main assumption that reaction  
 143 occurs in small turbulent structures, as in jet-and-recirculation stabilized FLOX<sup>®</sup> systems. Those length  
 144 fractions are modeled with

$$\xi = 2.1377 \left( \frac{\nu\epsilon}{k^2} \right)^{0.25}, \quad (8)$$

145 and species are assumed to react over a time scale

$$\tau = 0.4082 \left( \frac{\nu}{\epsilon} \right)^{0.5}. \quad (9)$$

146 In practice, multiple constant pressure reactors are solved, where initial conditions are the local  
 147 species and temperature of a computational cell. This approach for combustion modeling has been  
 148 proven to be feasible from experience of FLOX<sup>®</sup> system reacting computations. The combustion source  
 149 term consequently results as

$$R_i = \frac{\rho(\xi)^2}{\tau[1 - \xi^2]} (Y_i^* - Y_i), \quad (10)$$

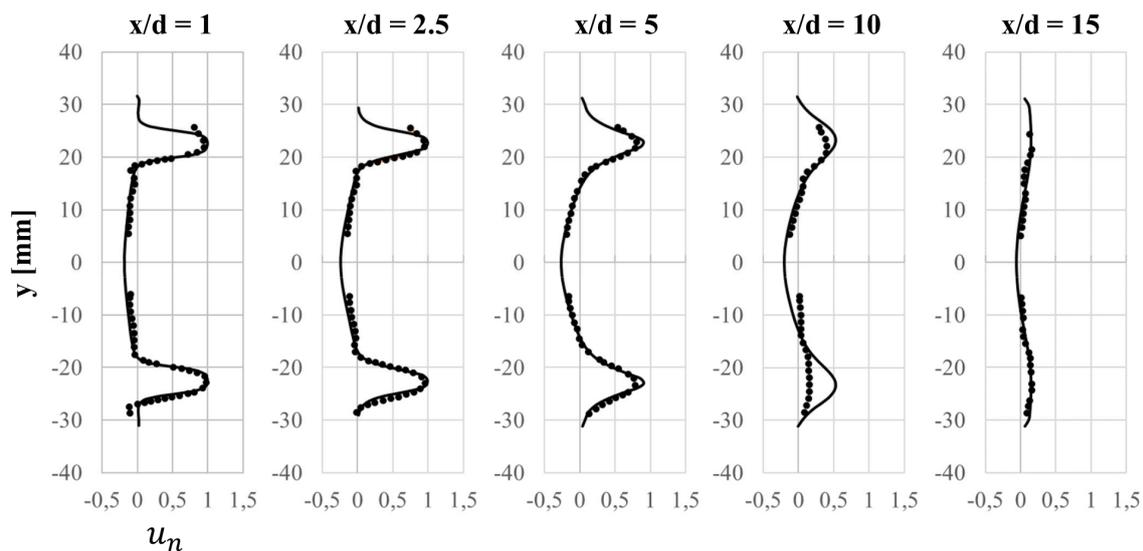
150 where  $Y_i^*$  is the fine scale species mass fraction after reaction with the previously defined time  
 151 scale.

152 At first, a cold flow simulation is established, the results of which is compared to experimentally gained  
 153 velocities from PIV. Combustion is then simulated with the previously introduced model framework.  
 154 Reactor network modeling is finally carried out on basis of the reacting field solution, taking into  
 155 consideration velocities, temperatures, and local compositions.

156 In terms of boundary conditions, mass flows are specified at the domain inlets for fuel and air, whereas  
 157 a pressure outlet boundary condition is set. As thermal boundary conditions, only inlet temperatures  
 158 are predefined. Heat losses are applied as a parametric study on selected reactors in the later on  
 159 constructed network model, since heat loss has great effects on the prediction of pollutant emissions.  
 160 Heat losses applied to the system also account for radiation heat loss, which is not explicitly modeled  
 161 in the CFD simulations.

### 162 3.2. Validation with Experiments

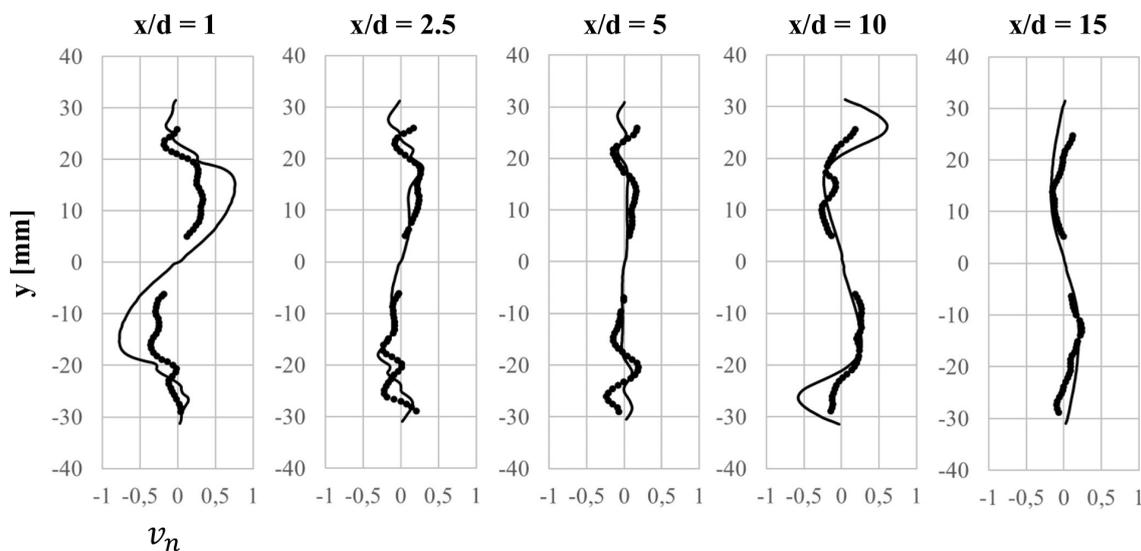
163 Compared are averaged axial and radial velocities from PIV measurements with CFD velocities  
 164 and flame surface density (FSD) from optical OH\* chemiluminescence and specific reaction heat from  
 165 the numerical simulations. The validation of the velocity field is carried out based on results in Fig. 6,  
 166 7, and 8.



**Figure 6.** Comparison of PIV (dots) [42] and CFD (lines) axial velocities at different heights  $x/d$  above the burner nozzles. Normalized velocities.

167 Axial velocities in Fig. 6 show in total excellent agreement between experiment and numerical  
 168 CFD simulation. Velocity peaks induced by the high-momentum jets are well depicted in terms of

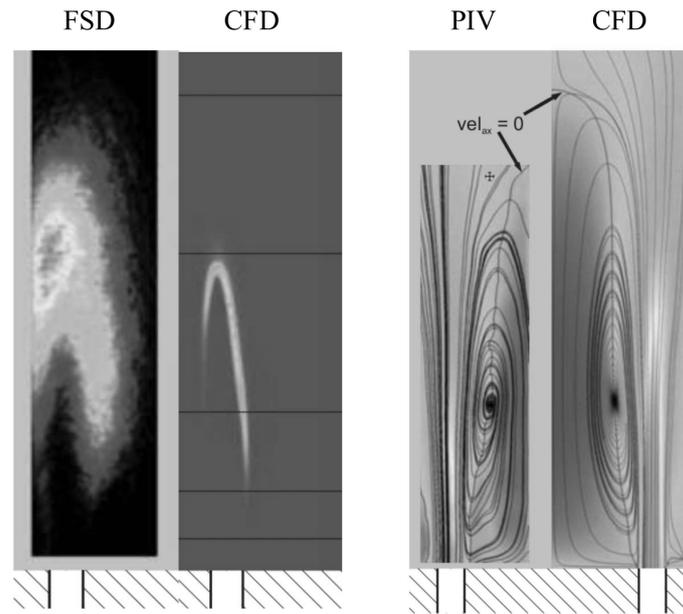
169 lateral position and velocity magnitude. Also, inner recirculation zone in terms of magnitude and  
 170 extension is well met by the simulation, denoted by the negative values around the burner center axis.  
 171 Deviations between measurements and simulation are present at the profile line  $x/d = 10$ , however it  
 172 becomes obvious that the experimental profiles show a strong asymmetry despite the symmetry of the  
 173 burner setup and therefore larger downstream uncertainties are assumed in the experiment.  
 174 Another deviation from experiments becomes evident from Fig. 8, where a field plot from PIV  
 175 streamlines is compared with a CFD midplane cut of axial velocities (right side comparison). There,  
 176 the location of the downstream stagnation point is farther downstream compared to the experimental  
 177 value which is however not expected to induce major errors in the reactor networks, since this deviation  
 178 is relatively far away from the upstream reaction zones. In total, the axial velocity field from CFD is  
 179 close enough to experimental findings in order to apply low-fidelity reactor network modeling.



**Figure 7.** Comparison of PIV (dots) [42] and CFD (lines) radial velocities at different heights  $x/d$  above the burner nozzles. Normalized velocities, scaled by factor 8.

180 Radial velocities are compared in Fig. 7. Experimental and numerical data is normalized with  
 181 the same peak value as in the axial velocities and therefore the data is depicted with an upscaling of a  
 182 factor of eight. In total, the numerical values nicely represent the characteristics of the experimental  
 183 findings, however, larger deviations can be observed for  $x/d = 1$  and  $x/d = 10$ . Those deviations  
 184 are rated as of minor importance for the extraction of reactor networks, since radial velocities play  
 185 a minor role in the overall flow field compared to the axial velocity, due to the overall large axial  
 186 momentum of the flow in the combustion chamber. Deviations for  $x/d = 10$  are pronounced by the  
 187 factoring of the results, as previously introduced. Another aspect is assumed in an overprediction of  
 188 inner recirculation zone extension in the combustor that can lead to increased radial velocities towards  
 189 the stagnation point at this downstream position.

190 Chemical reactions are evaluated with a qualitative comparison between flame surface density  
 191 and specific reaction heat in Fig. 8. Lift-off heights between experiment and numerical simulation  
 192 are in good agreement, whereas the spatial extension of the reaction zone is much more volumetric  
 193 compared to the simulation. Possible explanations for this are that intermediate species production is  
 194 under-predicted by the chemical kinetics reaction mechanism on the simulation side or that optical  
 195 measurement results are scaled differently compared to CFD findings, resulting in regions being less  
 196 prominent in terms of reaction are more highlighted in the experiment.  
 197 This is however a general trend that has been seen in preceding simulations [42] and is not to be  
 198 expected to be improved by parametric studies regarding reaction schemes or combustion models.



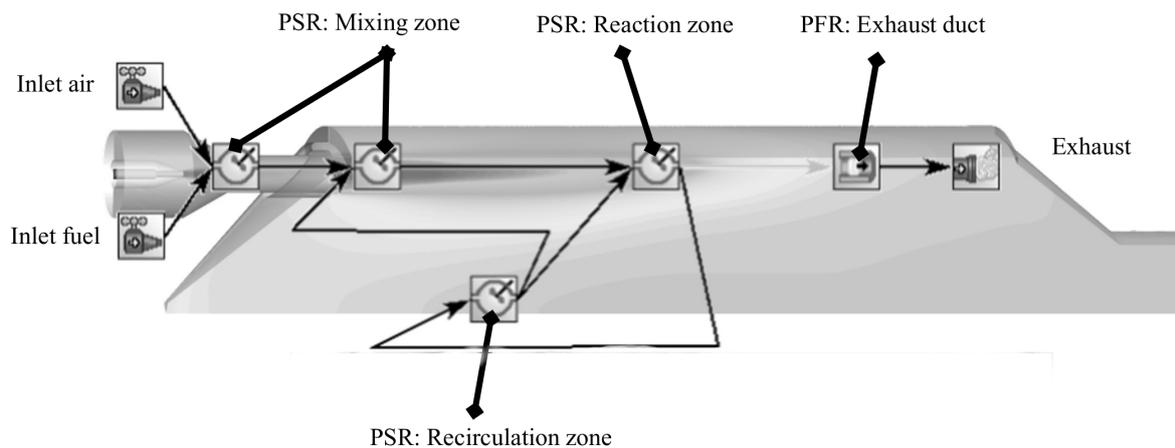
**Figure 8.** Left: Comparison of OH\* chemiluminescence flame surface density (FSD) [42] with specific reaction heat. Right: Comparison of experimental PIV axial velocities [42] and averaged CFD flow field. Light colors denote high values and vice versa.

#### 199 4. Reaction Network Modeling

200 This section consists of setup, modeling procedure, and results of ERN modeling with the Ansys  
201 Energico and Chemkin tool chain.

##### 202 4.1. Setup and Modeling Procedure

203 A major aspect of the presented work is the comparison of manually and algorithmically  
204 constructed ERN models. The manually constructed network denotes the most simple one. Alignment  
205 and distribution of reactors is shown in Fig. 9.

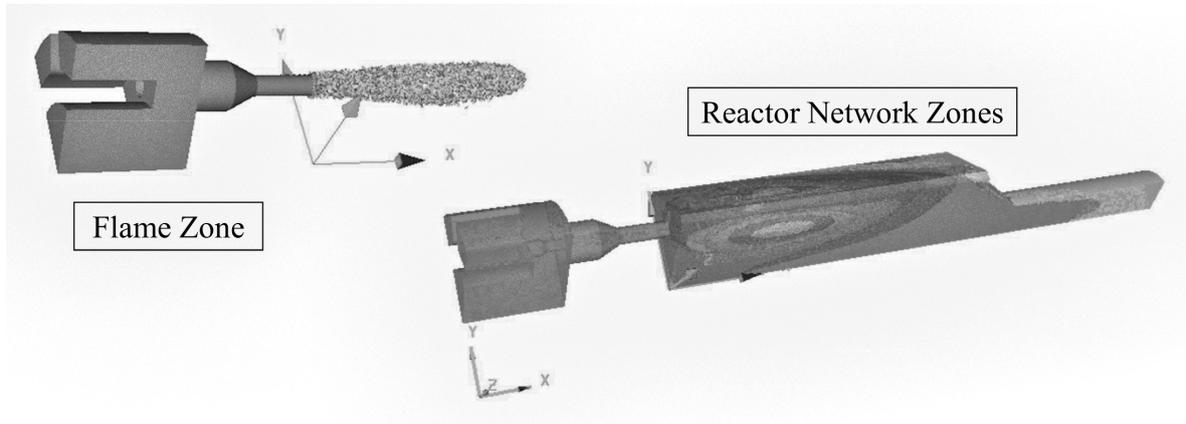


**Figure 9.** Manually constructed ERN scheme of the FLOX® single nozzle setup with underlying CFD contour of static temperature.

206 The reactor network in Fig. 9 consists of mainly PSR (perfectly stirred reactor) elements, which  
207 are clustered for the chemistry solver Chemkin. Fuel and air are fed separately to the system, as in  
208 the real FLOX configuration. The PSR cluster consists of elements for the upstream mixing section, a

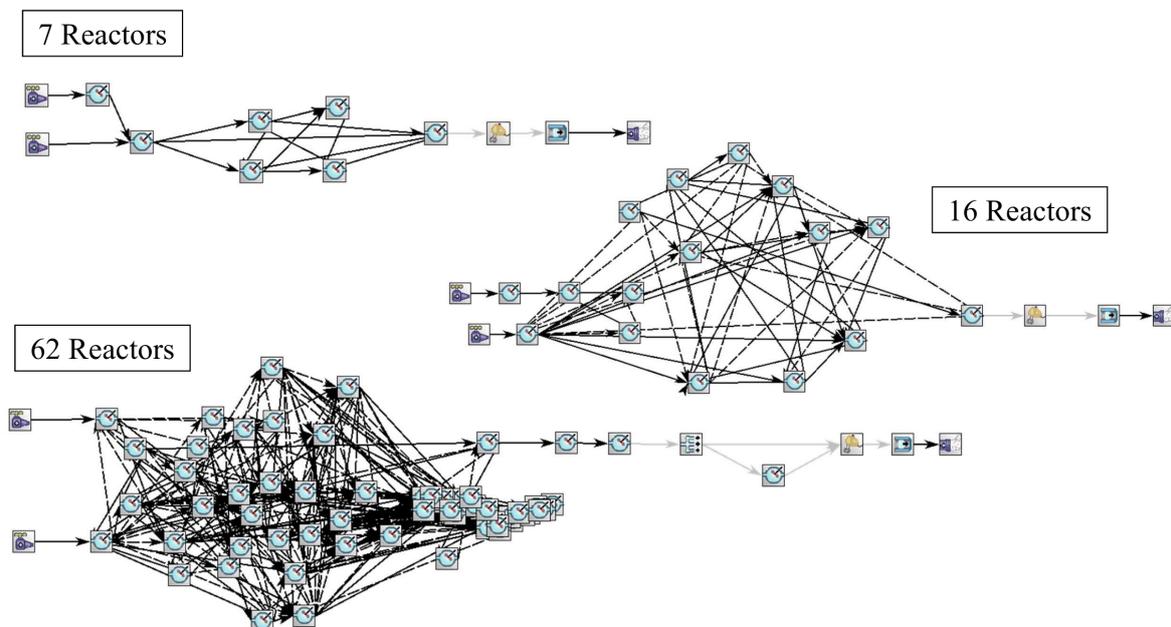
209 reaction and a large recirculation zone. The burner exhaust section is modeled with a PFR (plug flow reactor) element.  
 210 reactor) element.

211 This most simple model is compared with algorithmically created networks. Filtering of reactors for  
 212 the sections mixing, combustion and recirculation are filtered based on the local temperature gradient.  
 213 Therefore, a flame zone is identified and based on the temperature, most reactor sections are gathered  
 214 around the flame zone, as indicated in Fig. 10.



**Figure 10.** Results of zone identification by Energico for the flame zone and reactor network zones.

215 In order to test different degrees of ERN resolution, the filter width based on temperature is  
 216 adapted in Energico, resulting in different network models, which are shown in Fig. 11. Three different  
 217 reactor networks are investigated, mainly differing in the number of PSR reactor elements. As becomes  
 218 obvious from Fig. 11, a decreased filter width results in a higher reactor resolution in the mixing,  
 219 reaction and recirculation zones. This is expected due to the fact that temperature gradients are used as  
 220 filtering criteria and largest temperature gradients are present in the mixing and combustion section.



**Figure 11.** FLOX® ERNs with different numbers of PSR reactors, automatically constructed in Ansys Energico.

221 Again, for the algorithmically created networks, PSR elements are clustered and solved iteratively  
 222 by Chemkin, before the PFR section is computed. An energy equation is solved for each PSR element,  
 223 in the algorithmically created networks as well as in the manually created setup.

224 In cases where heat loss due to convection and radiation is applied to the network, at first adiabatic  
 225 simulations are carried out. Therefrom, reactors with a significantly increased heat production rate are  
 226 identified. Heat loss is then applied to those PSR reactors. Different methods of heat loss application  
 227 to the whole PSR cluster with even distribution were tested but resulted in non-converging Chemkin  
 228 computations.

229 For all investigated cases, the detailed GRI3.0 reaction scheme is used for the chemical properties in  
 230 the Chemkin simulations. Computational turnaround times are in the order of magnitude of seconds  
 231 for the manually created network and range up to minutes for the largest investigated network with  
 232 62 PSR elements.

#### 233 4.2. Results and Discussion

234 Selected reactor network statistics are shown in Fig. 12. Compared are the automatically and  
 235 manually reproduced distributions of reactor volumes and residence times. It has to be noted that the  
 236 manually created ERN was constructed and solved prior to automatic network generation with the  
 237 more complex cases.

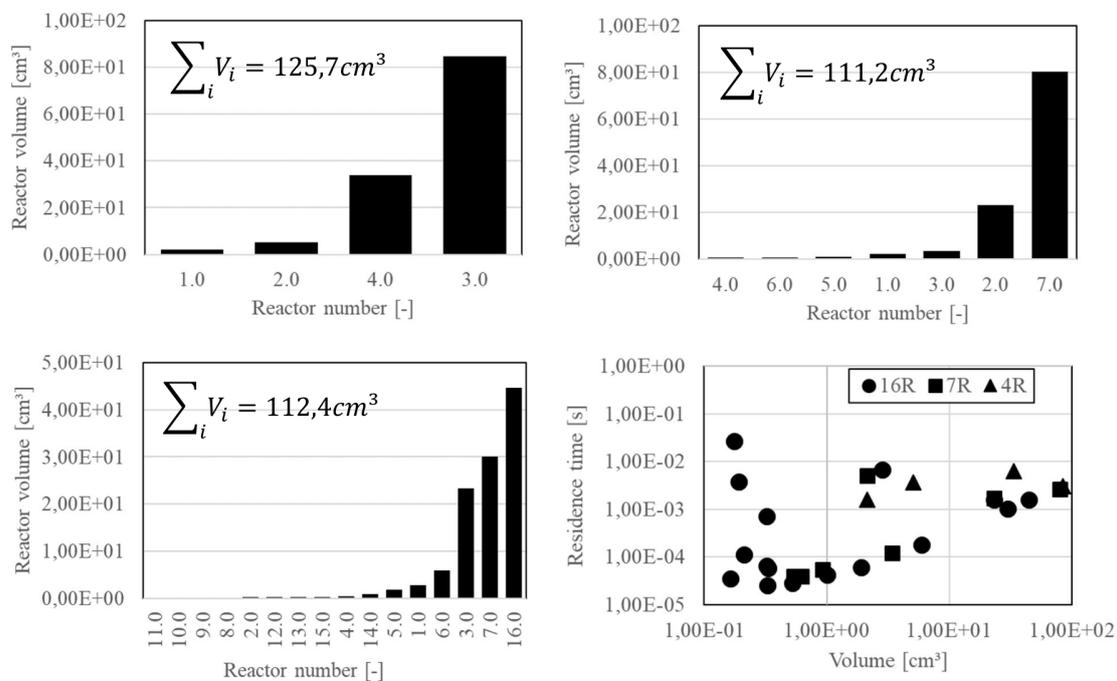


Figure 12. ERN volume distribution and comparison of residence time distribution.

238 From the statistics in Fig. 12 it appears that the manually created network considers an overall  
 239 slightly higher network volume of the combustor compared to the automatically created 7R and 16R  
 240 cases, however absolute values and the distribution of volumes are similar. Automatically created  
 241 networks for 7R and 16R consider almost identical total volumes, indicating a highly consistent  
 242 network composition method, where ERN refinement takes place within the reaction zone regions.

243 In terms of residence times, as shown in the bottom right figure of Fig. 12, similar ranges are covered  
 244 for all displayed cases, whereas the more detailed 16R network adds information for smaller reactors in  
 245 terms of volume with a relatively broad range of residence times. It has to be concluded from detailed  
 246 emissions results, whether this leads to significantly different results compared to the more simple  
 247 ERNs. However, the depiction of similar residence times of the 7R and 16R cases with respect to the  
 248 ERN volumes further substantiates that automatic ERN creation is consistent in terms of chemical  
 249 properties reproduction from CFD.

250 Detailed results of the emissions calculations with Chemkin are displayed in Fig. 13. Shown are

emissions simulation results compared with experimental data for NO<sub>x</sub> and CO over air-fuel number  $\lambda$  for different degrees of reactor network model fidelities. For each study, the applied heat loss is varied in a sensible range for atmospheric testing [49].

From a qualitative standpoint, NO<sub>x</sub> and CO trends over air-fuel number give a good representation of experimental findings, for all investigated levels of model fidelity (numbers of PSR reactors). For NO<sub>x</sub> the trends are more distinctive towards less heat loss applied and vice versa for CO. Another global observation is that no general improvement can be stated with increasing degree of model fidelity. It therefore appears feasible to use a simple CFD discretization approach with reactors in order to get estimations for emissions in the combustor design process, which is highly beneficial, since very simple and robust design studies can be carried out.

In terms of heat loss application to the ERN, it can be stated that values between 20 and 30% w.r.t. overall fuel energy content give best agreement with experimental findings for NO<sub>x</sub> and towards 40% heat loss application for CO prediction. As shown in preceding studies, those fractions represent realistic values [49].

At this point it has to be stated that results shown in Fig. 13 predict absolute values of emissions, results are not artificially scaled. Therefore, overall agreement is in a quantitative agreement that allows for reliable consideration of the method for combustor design.

As previously stated, having an idea about the amount of heat loss in a system is important in order to get quantitative emissions predictions. This plays a major role for either NO<sub>x</sub> and CO simulation results. From a combustor design point of view it is crucial to find an optimum range in which the system operates. As shown in Fig. 13, NO<sub>x</sub> developments are nicely reproduced by the ERN and therefore it is possible to derive optimum operation in terms of NO<sub>x</sub>, which is expectedly at higher air-fuel numbers.

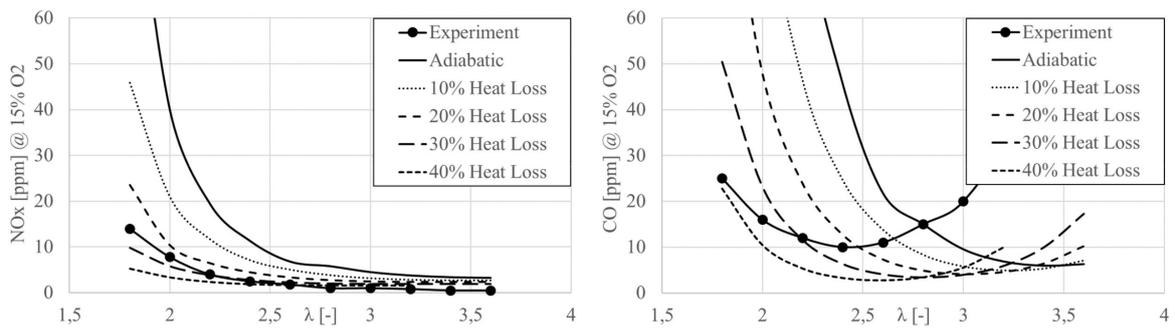
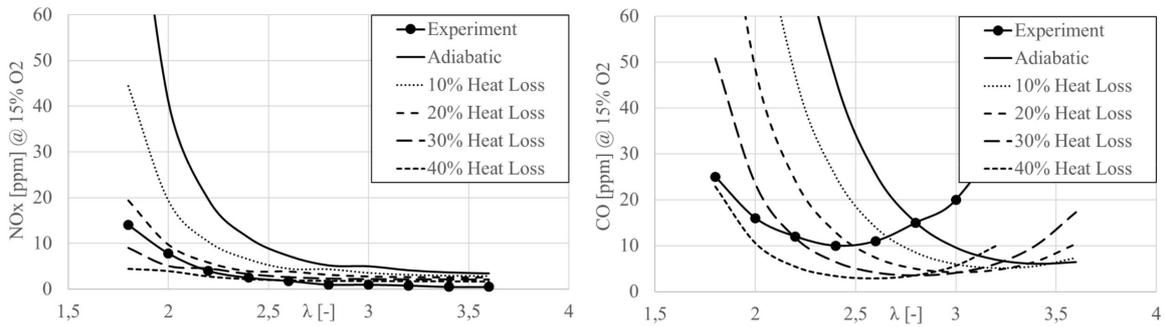
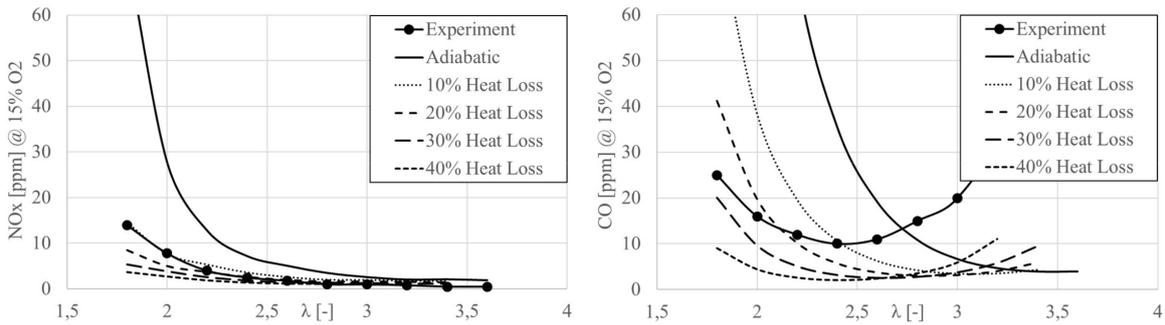
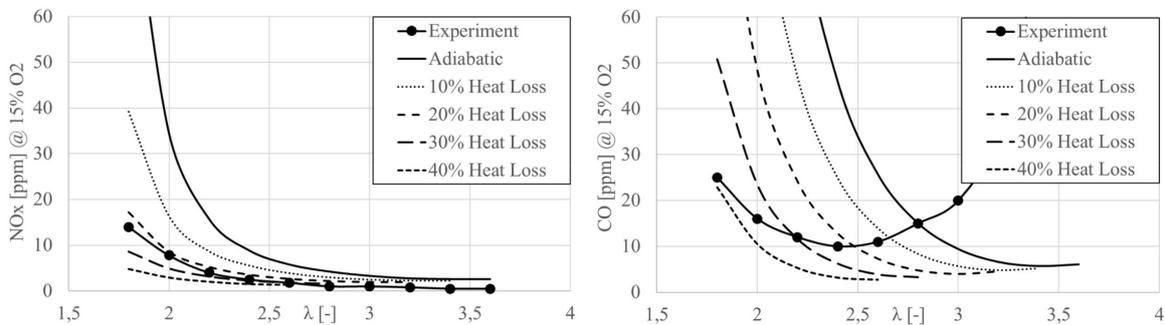
Characteristic troughs of CO emissions are also qualitatively reconstructed with the ERN approach, almost regardless of the degree of model fidelity. Higher values of CO are due to a hindered reaction of CO to CO<sub>2</sub>. The overall trough shape is displaced towards lower air-fuel numbers for increasing heat loss, which is plausible, since this effect is more pronounced, the more heat loss is applied to the system. Surprisingly, with increased model fidelity, the trough shape of CO emission curves is more poorly predicted. The main reason for this circumstance is assumed to be the application method of heat loss on reactors with pronounced chemical reaction. With higher model fidelity (increased number of PSRs), reaction is more distributed to more reactor elements. This leads to computational instabilities when a significant amount of heat loss is applied and consequently regions with larger air-fuel number give no sensible results. Thus, using a more robust ERN model with a smaller number of reactors is not only more applicable in practical situations, but also results in better prediction of CO emissions values.

Emissions prediction results in Fig. 13 furthermore show an inconsistency for  $N = 16$ , where CO increases towards richer regimes are displaced to richer conditions, compared to the other results. This is presumably also linked to inaccuracies in heat loss application to selected reactors. A more differentiated application of heat loss to selected reactors based on location in the combustor is however not always workable, especially when network model fidelity increases.

Overall it can be stated that the presented approach is highly suitable for the application in combustor design, since it is possible to pre-estimate optimum operation conditions in terms of NO<sub>x</sub> and CO emissions.

## 5. Conclusions

A low order reactor network approach for the prediction of NO<sub>x</sub> and CO emissions in FLOX<sup>®</sup> combustor design was introduced, validated, and discussed. The approach was tested on basis of an atmospheric test rig with a large experimental data set. The approach relies on construction of reactor networks based on CFD data and solution in Ansys Chemkin. Therefore, CFD results were successfully validated with experimental velocities from PIV and flame shape and position from

(a) Simplest reactor model,  $N = 4$ (b)  $N = 7$ (c)  $N = 16$ (d) Most complex reactor model,  $N = 62$ 

**Figure 13.** NO<sub>x</sub> and CO emissions at 15 % oxygen, evaluated for the respective reactor network models.  $N$ : Number of PSR reactors. Variation of system heat loss in sensible ranges for atmospheric testing [49] with respect to overall fuel energy content.

chemiluminescence measurements.

Subsequently, networks with different degrees of fidelity (number of PSRs) and several degrees of heat loss application were studied. It was shown that all reconstructed chemical reactor networks are able to reliably predict emissions trends as a function of global air-fuel number. Even very low order networks that are manually created with CFD results as guidance showed good results in terms of reproduction of experimental findings. In terms of NO<sub>x</sub> values, the ERN models were able to predict absolute emissions values with realistic values of heat loss application. For CO emissions, the ERN models were able to provide emissions levels in the correct order of magnitude and an orientation for the measured optimum operation conditions. However, the approach showed weaknesses in the prediction of CO emissions, especially when model fidelity was increased, presumably due to modeling issues.

Heat loss application to the Chemkin ERN model is a clear weakness of the approach, since it can lead to computational instabilities, especially in regimes with high air-fuel numbers and larger amounts of heat loss. Different approaches for heat loss application to the system were tested, like global application to the PSR cluster or application to fixed reactors. The most practical approach was the application to reactors with significant heat production, which then have to be identified a priori.

Overall the method can be seen as highly suitable for combustor design applications, since it is easy to use and features computational turnaround times in the order of minutes, even for a larger amount of PSR elements. In such cases, the modeling effort exceeds the computational times by far. The approach can be readily applied applied to large scale combustion, industrial applications or combustor applications in actual gas turbines.

**Funding:** This research received no external funding

**Acknowledgments:** Ms. Aurelia Drude from the German Aerospace Center (DLR) is gratefully acknowledged for the provision of Chemkin Pro simulation data.

**Conflicts of Interest:** The authors declare no conflict of interest.

## Abbreviations

The following abbreviations are used in this manuscript:

CFD	Computational Fluid Dynamics	—
DES	Detached Eddy Simulation	—
DLR	German Aerospace Center	—
EDC	Eddy Dissipation Concept	—
EDM	Eddy Dissipation Model	—
ERN	Emissions Reactor Network	—
FLOX <sup>®</sup>	Flameless Oxidation	—
FSD	Flame Surface Density	—
LBO	Lean Blow Out	—
LES	Large Eddy Simulation	—
MGT	Micro Gas Turbine	—
MILD	Moderate or Intense Low oxygen Dilution	—
MTT	Micro Turbine Technology b.v.	—
PFR	Plug Flow Reactor	—
PIV	Particle Image Velocimetry	—
PSR	Perfectly Stirred Reactor	—
RANS	Reynolds Averaged Navier Stokes	—
UHC	Unburned HydroCarbon	—

## References

1. Bundesministerium fuer Umwelt, N.u.R. TA Luft. 2021.

- 330 2. Nguyen, T. Improved Chemical Reactor Network Application for Predicting the Emission of Nitrogen  
331 Oxides in a Lean Premixed Gas Turbine Combustor. *Combustion, Explosion, and Shock Waves* **2019**,  
332 *55*, 267–273. doi:10.1134/S0010508219030031.
- 333 3. Zeldovich, Y. Oxidation of Nitrogen in Combustion and Explosions. Selected Works of Yakov Borisovich  
334 Zeldovich, Volume I, 1992. DOI: 10.1515/9781400862979.404.
- 335 4. Fenimore, C. Formation of nitric oxide in premixed hydrocarbon flames. *Symposium (International) on*  
336 *Combustion* **1971**, *13*, 373–380. doi:10.1016/S0082-0784(71)80040-1.
- 337 5. Malte, P.; Pratt, D. Measurement of atomic oxygen and nitrogen oxides in jet-stirred combustion. *Symposium*  
338 *(International) on Combustion* **1975**, *15*, 1061–1070. doi:10.1016/S0082-0784(75)80371-7.
- 339 6. Gruhlke, P.; Beck, C.; Janus, B.; Kempf, A. LES Analysis of CO Emissions from a High Pressure Siemens  
340 Gas Turbine Prototype Combustor at Part Load. *Energies* **2020**, *13*. doi:10.3390/en13215751.
- 341 7. Lieuwen, T.; Yang, V. Gas Turbine Emissions. Cambridge University Press, 2013.  
342 10.1017/CBO9781139015462.
- 343 8. Lovett, J.; Abuaf, N. Emissions and Stability Characteristics of Flameholders for Lean-Premixed  
344 Combustion. ASME Turbo Expo, 1992. 92-GT-120, V003T06A017, DOI: 10.1115/92-GT-120.
- 345 9. Nguyen, Q.; Edgar, B.; Dibble, R.; Gulati, A. Experimental and Numerical comparison of extractive and in  
346 situ laser measurements of non-equilibrium carbon monoxide in lean-premixed natural gas combustion.  
347 *Combustion and Flame* **1995**, *100*, 395–406. doi:10.1016/0010-2180(94)00128-F.
- 348 10. Howard, J.; Williams, G.; Fine, D. Kinetics of carbon monoxide oxidation in postflame gases. *Symposium*  
349 *(International) on Combustion* **1973**, *14*, 975–986. doi:10.1016/S0082-0784(73)80089-X.
- 350 11. Bulat, G.; Jones, W.; Marquis, A. NO and CO formation in an industrial gas-turbine combustion  
351 chamber using LES with the Eulerian sub-grid PDF method. *Combustion and Flame* **2014**, *161*, 1804–1825.  
352 doi:10.1016/j.combustflame.2013.12.028.
- 353 12. Jaravel, T.; Riber, E.; Cuenot, B.; Bulat, G. Large Eddy Simulation of an industrial gas turbine combustor  
354 using reduced chemistry with accurate pollutant prediction. *Proceedings of the Combustion Institute* **2017**,  
355 *36*, 3817–3825. doi:10.1016/j.proci.2016.07.027.
- 356 13. Wegner, B.; Gruschka, U.; Krebs, W.; Egorov, Y.; Forkel, H.; Ferreira, J.; Aschmoneit, K. CFD Prediction  
357 of Partload CO Emissions Using a Two-Timescale Combustion Model. *J. Eng. Gas Turbines Power* **2011**,  
358 *133*, 071502. doi:10.1115/1.4002021.
- 359 14. Klarmann, N.; Zoller, B.; Sattelmayer, T. Numerical modeling of CO-emissions for gas turbine  
360 combustors operating at part-load conditions. *J. Glob. Power Propuls. Soc.* **2018**, *2*, 376–387.  
361 doi:10.22261/JGPPS.C3N5OA.
- 362 15. Maas, U.; Pope, S. Simplifying chemical kinetics: Intrinsic low-dimensional manifolds in composition  
363 space. *Combustion and Flame* **1992**, *88*, 239–264. doi:10.1016/0010-2180(92)90034-M.
- 364 16. Bhargava, A.; Kendrick, D.; Colket, M.; Sowa, W.; Casleton, K.; Maloney, D. Pressure Effect on NOx and CO  
365 Emissions in Industrial Gas Turbines. ASME Turbo Expo, 2000. 2000-GT-0097, DOI: 10.1115/2000-GT-0097.
- 366 17. Falcitelli, M.; Pasini, S.; Rossi, N.; Tognotti, L. CFD+reactor network analysis: an integrated methodology  
367 for the modeling and optimisation of industrial systems for energy saving and pollution reduction. *Applied*  
368 *Thermal Engineering* **2002**, *22*, 971–979. doi:10.1016/S1359-4311(02)00014-5.
- 369 18. Falcitelli, M.; Pasini, S.; Tognotti, L. Modelling practical combustion systems and predicting NOx  
370 emissions with an integrated CFD based approach. *Computers and Chemical Engineering* **2002**, *26*, 1171–1183.  
371 doi:10.1016/S0098-1354(01)00771-2.
- 372 19. Falcitelli, M.; Tognotti, L.; Pasini, S. An algorithm for extracting chemical reactor network models from  
373 cfd simulation of industrial combustion systems. *Combustion Science and Technology* **2002**, *174*, 27–42.  
374 doi:10.1080/713712951.
- 375 20. Mohamed, H.; Ticha, H.; Mohamed, S. Simulation of pollutant emissions from a gas-turbine combustor.  
376 *Combustion Science and Technology* **2004**, *176*, 819–834. doi:10.1080/00102200490428422.
- 377 21. Novosselov, I.; Malte, P.; Yuan, S.; Srinivasan, R.; Lee, Y. Chemical Reactor Network Application to  
378 Emissions Prediction for Industrial DLE Gas Turbine. ASME Turbo Expo, 2006. GT2006-90282, DOI:  
379 10.1115/GT2006-90282.
- 380 22. Russo, C.; Mori, G.; Anisimov, V.; Parente, J. Micro Gas Turbine Combustor Emissions Evaluation  
381 Using the Chemical Reactor Modelling Approach. ASME Turbo Expo, 2007. GT2007-27687, DOI:  
382 10.1115/GT2007-27687.

- 383 23. Benedetto, D.; Pasini, S.; Falcitelli, M.; La Marca, C.; Tognotti, L. NO<sub>x</sub> Emission Prediction from 3-D  
384 Complete Modelling to Reactor Network Analysis. *Combustion Science and Technology* **2008**, *153*, 279–294.  
385 doi:10.1080/00102200008947265.
- 386 24. Fichet, V.; Kanniche, M.; Plion, P.; Gicquel, O. A reactor network model for predicting NO<sub>x</sub> emissions in  
387 gas turbines. *Fuel* **2010**, *89*, 2202–2210. doi:10.1016/j.fuel.2010.02.010.
- 388 25. Lee, D.; Park, J.; Jin, J.; Lee, M. A simulation for prediction of nitrogen oxide emissions in lean premixed  
389 combustor. *Journal of Mechanical Science and Technology* **2011**, *25*. doi:10.1007/s12206-011-0425-9.
- 390 26. Lyra, S.; Cant, R. Analysis of high pressure premixed flames using Equivalent Reactor Networks for  
391 predicting NO<sub>x</sub> emissions. *Fuel* **2013**, *107*, 261–268. doi:10.1016/j.fuel.2012.12.066.
- 392 27. De Toni, A.; Hayashi, T.; Schneider, P. A reactor network model for predicting NO<sub>x</sub> emissions in an  
393 industrial natural gas burner. *Journal of the Brazilian Society of Mechanical Sciences and Engineering* **2013**,  
394 *35*, 199–206. doi:10.1007/s40430-013-0039-5.
- 395 28. Colorado, A.; McDonell, V. Reactor Network Analysis to asses fuel composition effects on NO<sub>x</sub> emissions  
396 from a recuperated gas turbine. Proceedings of ASME Turbo Expo; , 2014. GT2014-26361.
- 397 29. Nguyen, T. Chemical reactor network application to predict the emission of nitrogen oxides in  
398 an industrial combustion chamber. *Combustion, Explosion, and Shock Waves* **2017**, *53*, 406–410.  
399 doi:10.1134/S0010508217040049.
- 400 30. Nguyen, T.; Kim, S.; Park, J.; Jung, S.; Kim, S. CFD-CRN validation study for NO<sub>x</sub> emission prediction  
401 in lean premixed gas turbine combustor. *Journal of Mechanical Science and Technology* **2017**, *31*, 4933–4942.  
402 doi:10.1007/s12206-017-0942-2.
- 403 31. Innocenti, A.; Andreini, A.; Bertini, D.; Facchini, B.; Motta, M. Turbulent flow-field effects in a hybrid  
404 CFD-CRN model for the prediction of NO<sub>x</sub> and CO emissions in aero-engine combustors. *Fuel* **2018**,  
405 *215*, 853–864. doi:10.1016/j.fuel.2017.11.097.
- 406 32. Kaluri, A.; Malte, P.; Novosselov, I. Real-time prediction of lean blowout using chemical reactor network.  
407 *Fuel* **2018**, *234*, 797–808.
- 408 33. Gupta, S.; Malte, P.; Brunton, S.; Novosselov, I. Prevention of lean flame blowout using a predictive  
409 chemical reactor network control. *Fuel* **2019**, *236*, 583–588. doi:10.1016/j.fuel.2018.09.044.
- 410 34. Perpignan, A.; Sampat, R.; Gangoli, R. Modeling Pollutant Emissions of Flameless Combustion With a Joint  
411 CFD and Chemical Reactor Network Approach. *Front. Mech. Eng* **2019**, *5*. doi:10.3389/fmech.2019.00063.
- 412 35. Zhang, Q.; Hai, H.; Li, C.; Wang, Y.; Zhang, P.; Wang, X. Predictions of NO<sub>x</sub> and CO emissions from  
413 a low-emission concentric staged combustor for civil aeroengines. *J. Aerospace Engineering* **2020**,  
414 *234*, 1075–1091. doi:10.1177/0954410019895881.
- 415 36. Wüning, J.; Wüning, J. Flameless oxidation to reduce thermal NO-formation. *Progress in Energy and*  
416 *Combustion Science* **1997**, *23*, 81–94. doi:10.1016/S0360-1285(97)00006-3.
- 417 37. Cavaliere, A.; Joannon, M. Mild Combustion. *Progress in Energy and Combustion Science* **2004**, *30*, 329–366.  
418 doi:10.1016/j.pecc.2004.02.003.
- 419 38. Lückcrath, R.; Meier, W.; Aigner, M. FLOX® Combustion at High Pressure With Different Fuel  
420 Compositions. *Journal of Engineering for Gas Turbines and Power* **2008**, *130*. doi:10.1115/1.2749280.
- 421 39. Flamme, M. Low NO<sub>x</sub> Combustion Technologies for High Temperature Applications. *Energy conversion*  
422 *management* **2001**, *42*, 1919–1935. doi:10.1016/S0196-8904(01)00051-6.
- 423 40. Flamme, M. New combustion systems for gas turbines (NGT). *Applied Thermal Engineering* **2004**,  
424 *24*, 1551–1559. doi:10.1016/j.applthermaleng.2003.10.024.
- 425 41. Lammel, O.; Stöhr, M.; Kutne, P.; Dem, C.; Meier, W.; Aigner, M. Experimental Analysis of Confined Jet  
426 Flames by Laser Measurement Techniques. Proceedings of the ASME Turbo Expo; , 2011. GT2011-45111.
- 427 42. Seliger-Ost, H. Entwicklung eines FLOX-basierten Brennersystems für eine rekuperierte Mikrogasturbine  
428 im kleinen Leistungsbereich. PhD thesis, Universität Stuttgart, 2019.
- 429 43. Seliger-Ost, H.; Kutne, P.; Zanger, J.; Aigner, M. Experimental Investigation of the Impact of Biogas on a 3  
430 kW Micro Gas Turbine FLOX-based Combustor. ASME Turbo Expo 2020: Power for Land, Sea and Air,  
431 2020. GT2020-15556.
- 432 44. Seliger-Ost, H.; Kutne, P.; Zanger, J.; Aigner, M. Experimental Investigation of the Impact of Biogas on a  
433 3kW Micro Gas Turbine FLOX-Based Combustor. *Journal of Engineering for Gas Turbines and Power* **2021**,  
434 *143*, 081020.

- 435 45. Seliger, H.; Stöhr, M.; Yin, Z.; Huber, A.; Aigner, M. Experimental and Numerical Analyses of a FLOX-based  
436 Combustor for a 3 kW Micro Gas Turbine under Atmospheric Conditions. *ASME Turbo Expo 2017: Power  
437 for Land, Sea and Air*, 2017. GT2017-63317.
- 438 46. Cirigliano, D.; Grimm, F.; Kutne, P.; Aigner, M. Thermo-Structural Analysis of a Micro Gas Turbine Jet-  
439 and Recirculation- Stabilized Combustion Chamber. *ASME Turbo Expo 2020: Power for Land, Sea and Air*,  
440 2020. GT2020-14561.
- 441 47. ANSYS, I. *ANSYS Fluent Theory Guide*, 2020. Release 2020R1.
- 442 48. Li, J.; Zhao, Z.; Kazakov, A.; Dryer, F.L. An updated comprehensive kinetic model  
443 of hydrogen combustion. *International Journal of Chemical Kinetics* **2004**, *36*, 566–575,  
444 [<https://onlinelibrary.wiley.com/doi/pdf/10.1002/kin.20026>]. doi:<https://doi.org/10.1002/kin.20026>.
- 445 49. Grimm, F.; Lingstaedt, T.; Kutne, P.; Aigner, M. Numerical and Experimental Study of a  
446 Jet-and-Recirculation Stabilized Low Calorific Combustor for a Hybrid Power Plant. *Energies* **2021**,  
447 *14*. doi:10.3390/en14030537.

448 © 2022 by the authors. Submitted to *Energies* for possible open access publication under the terms and conditions  
449 of the Creative Commons Attribution (CC BY) license (<http://creativecommons.org/licenses/by/4.0/>).